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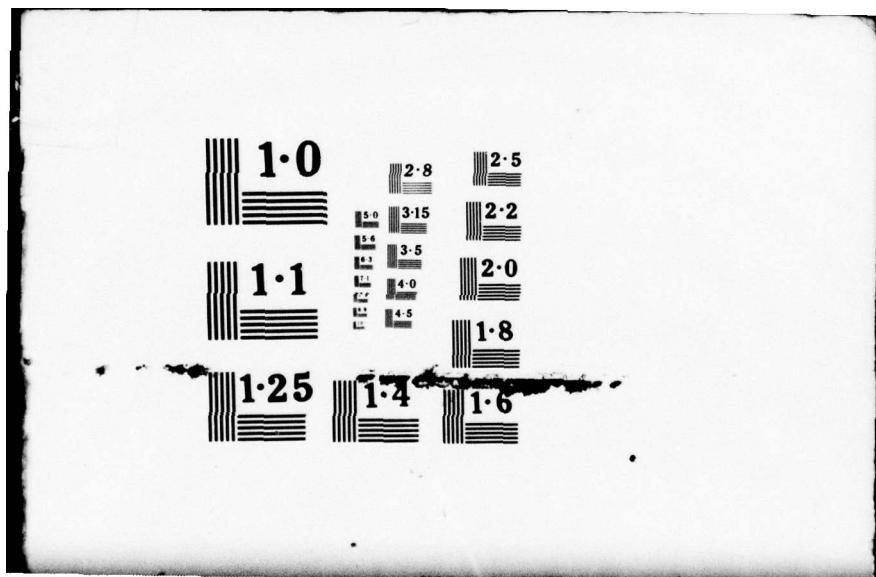
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## ANALYSIS OF MULTISTAGE, AXIAL FLOW TURBOMACHINE WAKE PRODUCTION, TRANSPORT, AND INTERACTION

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**ANALYSIS OF MULTISTAGE, AXIAL FLOW  
TURBOMACHINE WAKE PRODUCTION,  
TRANSPORT, AND INTERACTION**

**J. H. Wagner  
T. H. Okiishi  
December 1977**

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DEPARTMENT OF MECHANICAL ENGINEERING  
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## SUMMARY

A periodic-average flow measurement technique involving a hot-wire anemometer system was used to measure the periodically unsteady and three-dimensional fluid velocity field between blade rows in the first stage of a low-speed, multistage, axial-flow research compressor. These data suggest that the fluid flow through the imbedded rotor and stator rows are appreciably unsteady, in a periodic fashion, in portions of the compressor annulus. Illustrative examples of periodic-average fluid flow field variation with rotor blade sampling position in stop-action sequence are presented for different locations in the compressor. A simple, first order approximation physical description of the blade wake flow transport and interaction process largely based on experimental data interpretation is proposed to organize and to help explain the observations made. Blade span variations of flow data reflect end-wall effects. Inlet guide vane exit flow data involve some unusual unsteady flow effects.

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## SYMBOLS AND NOTATION

A	cross section area, $\text{m}^2$
$\vec{A}$	unit vector along hot-wire sensor (Figure 8)
$b_0, b_1,$	
$b_2 \dots b_9$	effective cooling velocity/actual velocity ratio correlation coefficients
c	blade chord length (Figure 3), meters
$E_\ell$	linearized anemometer bridge voltage, volts
g	local acceleration of gravity, $9.8026 \text{ m/s}^2$
$g_c$	conversion factor, $1.0 \text{ kgm/Ns}^2$
$h_{hg}$	height of barometer mercury column, meters
$K_1, K_2, K_3$	effective cooling velocity equation constants (Equation B-9)
m	constant hot-wire probe turning measurement angle increment (Figure 9), degrees
$P_{\text{atm}}$	barometric pressure (Equation B-1), $\text{N/m}^2$
PHH	percent passage height from hub (Equation B-4), percent
$Q_v$	Venturi metered volume flow rate (Equation B-5), $\text{m}^3/\text{s}$
r	radius from compressor axis, meters
R	gas constant, $\text{Nm/kg}^0\text{K}$
$R_{cb}$	cable resistance, ohms
$R_{ph}$	probe holder resistance, ohms
$R_{pl}$	probe lead resistance, ohms
$R_{s,c,d}$	cold resistance read off anemometer deck, ohms
$R_{s,op,d}$	sensor operating resistance anemometer deck setting (Equation 17), ohms
RPM	rotor rotational speed, rpm
R,Y,Z	compressor coordinate system (Figure 10)

S	circumferential space between blades, blade pitch (Figure 3), meters or degrees
t	temperature, $^{\circ}$ K
$t_{\text{baro}}$	barometer ambient temperature, $^{\circ}$ K
$t_{\text{max}}$	blade section maximum thickness (Figure 3), meters
U	rotor blade velocity (Equation B-6), m/s
V	calibration nozzle jet velocity (Equation B-7), m/s
$\vec{V}$	absolute velocity (Figure 10), m/s
$V'$	relative velocity (Equation B-17), m/s
$V_e$	hot-wire effective cooling velocity (Equation 5b), m/s
$V_z$	axial component of fluid velocity (Figure 10; Equation B-15), m/s
$V_{\theta}$	tangential component of absolute fluid velocity (Figure 10; Equation B-16), m/s
$V'_{\theta}$	tangential component of relative fluid velocity (Equation B-17), m/s
x, y, z	hot-wire probe coordinates fixed to probe (Figure 8)
Y	circumferential traversing position, degrees
Y <sub>0</sub>	circumferential blade row setting position when Y is equal to zero, circumferential distance from the probe traversing measurement stations to blade stacking axis, positive in the direction of rotation, degrees
$\alpha$	sensor yaw angle, angle between the velocity vector and hot-wire sensor (Figure 8; Equation 4), degrees
$\beta_{\text{mv}}$	approximate tangential flow angle (Figure 9), degrees
$\beta_r$	radial flow angle (Figure 10; Equation B-13), degrees
$\beta_{\theta}$	absolute tangential flow angle with respect to axial direction (Figure 10; Equation B-12), degrees
$\beta'_{\theta}$	relative tangential flow angle with respect to axial direction (Equation B-19), degrees
$\gamma$	blade stagger angle (Figure 3), degrees

$\gamma_{H_2O}$	specific weight of water (Equation B-3), $N/m^3$
$\gamma_{hg}$	specific weight of mercury, $N/m^3$
$\Delta P_n$	differential pressure between calibration nozzle plenum pressure and atmospheric pressure, meters of water
$\Delta P_{vent}$	differential pressure across venturi, meters of water
$\theta_0$	hot-wire sensor angle with respect to a plane normal to the probe axis (Figure 8), degrees
$\theta_{off}$	measurement off-set angle (Figure 9)
$\theta_p$	probe pitch angle (Figure 9), degrees
$\theta_y$	probe yaw angle (Figure 8), degrees
$\rho$	density of air (Equation B-2), $kg/m^3$
$\phi_v$	venturi flow coefficient (Equation B-8)

Additional General Subscripts

a,b,c	hot-wire probe measurement positions
h	annulus inner surface, hub
IGV	inlet guide vane
off	offset
R	rotor
S	stator
t	annulus outer surface, tip

## 1. INTRODUCTION

Modern design systems for turbomachines rely on, to a great extent, empirical correlations to reflect real fluid flow effects. This state of design competence often leads to good machines. However, more often than can be afforded, the resultant hardware involves undesirable deficiencies that can be related to insufficient knowledge about the fluid mechanics involved. For example, mis-understanding of the substantial relationships between the unsteadiness of turbomachine flow and the efficiency, aerodynamic and aeroelastic stability, and noise production of such machines can result in costly disappointments. Mikolajczak [1] suggests that further significant improvements in turbomachine technology should be sought through better understanding and control of the unsteady flows involved.

Progress in fluid flow measurement has resulted in the development of a variety of techniques (see, for example, References 2-27) for observing the unsteady aspects of turbomachine flows. Precise coordination of data acquisition with rotor sampling position has made possible the extraction of the periodically unsteady flow (periodic-average flow) data from the entire collection of information. The periodic-average flow is important because of the role it plays with respect to forced blade vibration, discrete frequency noise generation, and turbomachine energy transfer. Data yielding detailed information about the sequential variation of periodic-average flow with rotor sampling position are rare [14,15,16,24,26].

In this report are offered detailed periodic-average, three-

dimensional flow data for the first stage of a low-speed, multistage, axial-flow research compressor that demonstrate flow field variation with sequential change in rotor sampling position. A physical description of the complicated fluid mechanical processes involved is proposed.

## 2. RESEARCH COMPRESSOR FACILITY

The research compressor facility of the Iowa State University Engineering Research Institute/Mechanical Engineering Department Turbomachinery Components Research Laboratory was used for the present study. Briefly reviewed below is the research compressor and related equipment and instrumentation. For more detailed information see Reference 28.

### 2.1. Axial Flow Research Compressor

Figure 1 is a sketch of the entire research compressor apparatus. Figure 2 depicts, in more detail, the compressor portion only. The inlet guide vane (IGV) row and three identical rotor-stator stages were within an annulus having constant hub (0.284 m) and tip (0.406 m) diameters. All stationary blade rows had the same number of blades (37) and were mounted on separate ring assemblies which could be moved independently or in desired combinations. The rotor blade rows each included the same number of blades (38) and were assembled together so that all of the respective blade stacking axes were aligned axially. All of the blades were constructed of a plastic material (Monsanto ABS) with British C4 sections reflecting a free vortex design. Blade characteristics are summarized below:

Number of blades per row	IGV and stator rows - 37
	rotor rows - 38
Blade span (constant)	6.10 cm (2.4 in.)
Blade chord (constant), c	3.05 cm (1.2 in.)

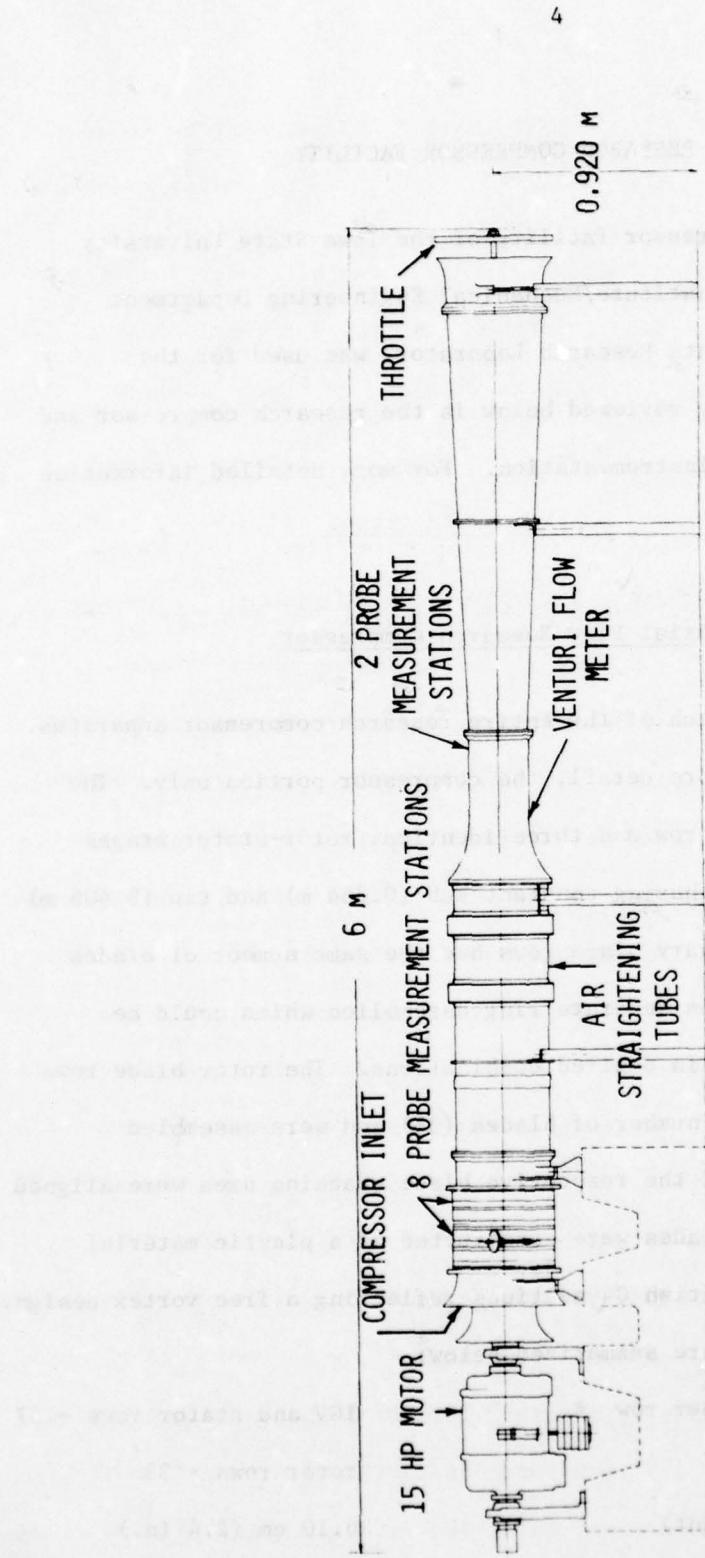


Figure 1. Research compressor apparatus side view.

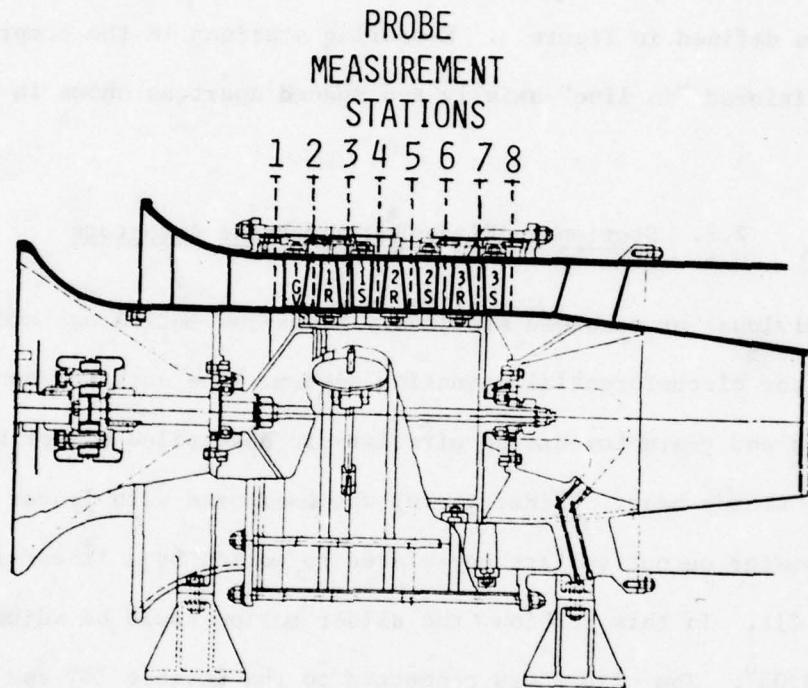


Figure 2. Research compressor with probe measurement stations.

Blade section maximum thickness/	10%
chord ratio, $t_{\max}/c$	

Blade geometry details are tabulated in Table 1 with associated variables defined in Figure 3. Measuring stations in the compressor were positioned "in line" axially and spaced apart as shown in Figure 4.

## 2.2. Stationary Blade-Row and Probe Actuators

Individual or combined stationary blade-row motion was accomplished with the circumferential actuating system. The actuator consisted of a rack and gearmotor-driven circular-arc dovetailed slider moving within a sturdy base. Slider travel was monitored with linear potentiometer output voltage correlated to motion by a linear least squares fit. In this fashion, the slider motion could be adjusted to within  $0.05^{\circ}$ . The slider was connected to the movable IGV and stator blade row rings with adjustable linkages so that a variety of stationary blade row positioning schedules could be achieved. Scales on each blade-row ring and the outer compressor casing were used for ascertaining the precise location of each blade row. The scales were calibrated in terms of degrees, positive in the direction of rotation. A reading of  $0.0^{\circ}$  corresponded to the stacking axis of a predetermined reference blade in each row being "in line" with the measurement stations of the compressor. Any circumferential distance of a reference blade stacking axis from this zero location was denoted as  $Y_0$ , its circumferential position. Throughout these tests, IGV and stator blade row rings were positioned for minimum sound as follows:

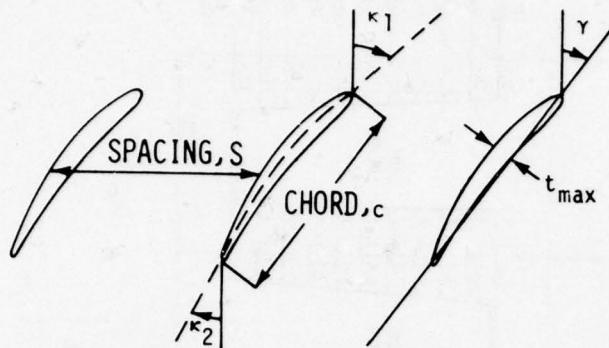


Figure 3. Blade nomenclature.

Table 1. Geometric blade tables for ICV, rotors, and stators at several radial locations.

Blade Row	Percent Passage Ht. From Hub PHH	Solidity c/S	Blade Angles			
			Stagger γ degrees	Inlet κ <sub>1</sub> degrees	Outlet κ <sub>2</sub> degrees	Camber κ <sub>1</sub> - κ <sub>2</sub> degrees
IGV	0	1.263	20.35	0.00	42.10	-42.10
	10	1.211	20.05	0.00	40.77	-40.77
	20	1.164	19.69	0.00	39.47	-39.47
	30	1.121	19.25	0.00	38.23	-38.23
	40	1.080	18.65	0.00	37.08	-37.08
	50	1.041	18.15	0.00	36.05	-36.05
	60	1.004	17.63	0.00	35.02	-35.02
	70	0.971	17.05	0.00	33.93	-33.93
	80	0.940	16.45	0.00	32.92	-32.92
	90	0.913	15.65	0.00	32.10	-32.10
Rotor	100	0.887	14.15	0.00	31.40	-31.40
	0	1.299	-20.54	-42.40	3.90	-46.30
	10	1.250	-24.39	-44.76	-2.84	-41.92
	20	1.205	-28.11	-46.85	-9.51	-37.34
	30	1.164	-31.70	-48.53	-15.96	-32.57
	40	1.123	-35.15	-49.82	-21.88	-27.94
	50	1.078	-38.47	-50.81	-27.06	-23.75
	60	1.035	-41.66	-51.77	-31.64	-20.13
	70	0.999	-44.71	-52.90	-35.78	-17.12
	80	0.968	-47.63	-53.98	-39.26	-14.72
Stator	90	0.939	-50.41	-54.82	-41.91	-12.91
	100	0.909	-53.07	-55.50	-44.10	-11.40
	0	1.263	40.24	54.80	26.70	28.10
	10	1.211	39.32	53.48	25.67	27.81
	20	1.164	38.39	52.36	24.68	27.68
	30	1.121	37.46	51.43	23.74	27.69
	40	1.080	36.54	50.25	22.77	27.48
	50	1.041	35.61	48.56	21.72	27.84
	60	1.004	34.68	47.13	20.76	26.37
	70	0.971	33.75	46.65	20.01	26.64
	80	0.940	32.83	46.36	19.34	27.02
	90	0.913	31.90	45.59	18.62	26.97
	100	0.887	30.97	44.50	17.85	26.65

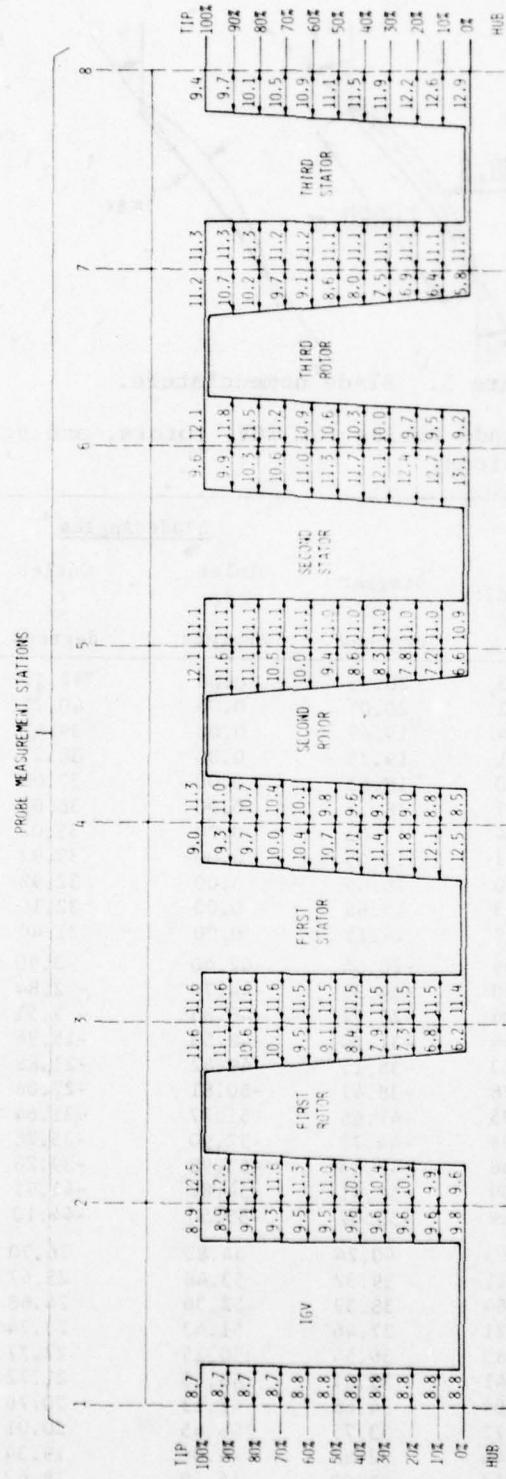


Figure 4. Schematic diagram showing axial location of probe measurement stations (dimensions in mm).

Inlet Guide Vanes	$Y_0_{IGV}/S_S = 0.00$
First Stator	$Y_0_{1S}/S_S = 0.17$
Second Stator	$Y_0_{2S}/S_S = 0.56$
Third Stator	$Y_0_{3S}/S_S = 0.77$

The relative positioning of some of these blades may be seen in Figure 5. As will be explained later, the circumferential actuator was the key to providing the means for measuring the blade-to-blade variations of flow in the compressor. Since the probes used could not be actually traversed in the circumferential direction, the stationary and "sampled" rotor blades had to be moved circumferentially relative to a fixed probe.

The probe actuator (L. C. Smith Company model 6180) was used with the control indicator (L. C. Smith Company model DI-3R) and actuator switch box (L. C. Smith Company model DI-4R-SB) to vary the probe yaw angle and immersion depth. Probe angles and positions in the compressor annulus were monitored by observing mechanical digital counter readings and linear potentiometer output voltages. Each potentiometer's output voltage was correlated with motion with a linear least squares fit which allowed probe angles and immersion positions to be measured within  $0.05^\circ$  and 0.15 mm respectively.

### 2.3. Pressure and Temperature Measurement Instrumentation

All working fluid pressure measurements were made with precision water-in-glass manometers (Meriam type Incl.) which were calibrated with a micromanometer (Meriam type Micro.). Room air temperature was

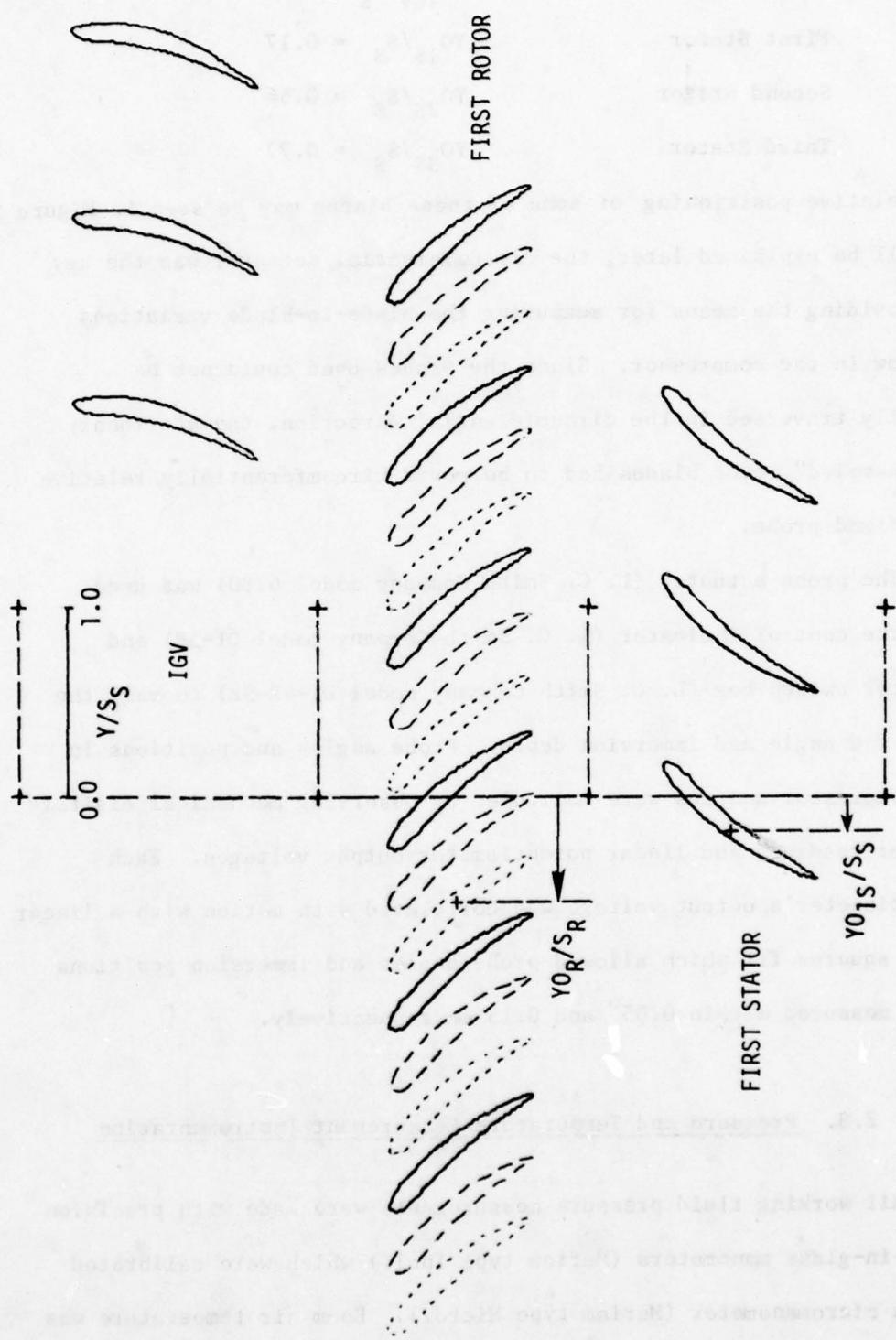


Figure 5. Blade cascade showing relative positions of blades for several rotor sampling positions.

measured with mercury-in-glass thermometers, while working fluid temperature was measured with a copper-constantan thermocouple and a precision millivolt potentiometer (Leeds and Northrup model 8686). Barometric pressure was measured using a mercury-in-glass barometer (Princo Instruments, Inc. model B-222).

#### 2.4. Periodic-Average Measurement System

The periodic-average measurement system, shown schematically in Figure 6, was composed of the following components:

- (1) Single slanted hot-wire probe (Disa model 55P02 Modified)
- (2) Constant temperature anemometer (Thermo-Systems, Inc. [TSI] model 1010A)
- (3) Linearizer (TSI model 1072)
- (4) Periodic sample-and-hold circuit
- (5) Photoelectric triggering circuit
- (6) Signal averaging circuit
- (7) Digital scanning voltmeter (Hewlett-Packard model 3480D)
- (8) Desk-top calculator (Hewlett-Packard model 9821A)
- (9) Oscilloscopes (Tektronix Inc.)

The hot-wire probe involved a  $5 \mu\text{m}$  diameter platinum-plated tungsten wire with a 1.25 mm sensing portion centered on the probe axis. The wire was copper and gold plated at the ends and positioned at an angle of  $54.7^\circ$  from the probe axis. The hot-wire anemometer and linearizer were used to produce a linear relationship between air velocity and output voltage. The periodic sample-and-hold circuit was

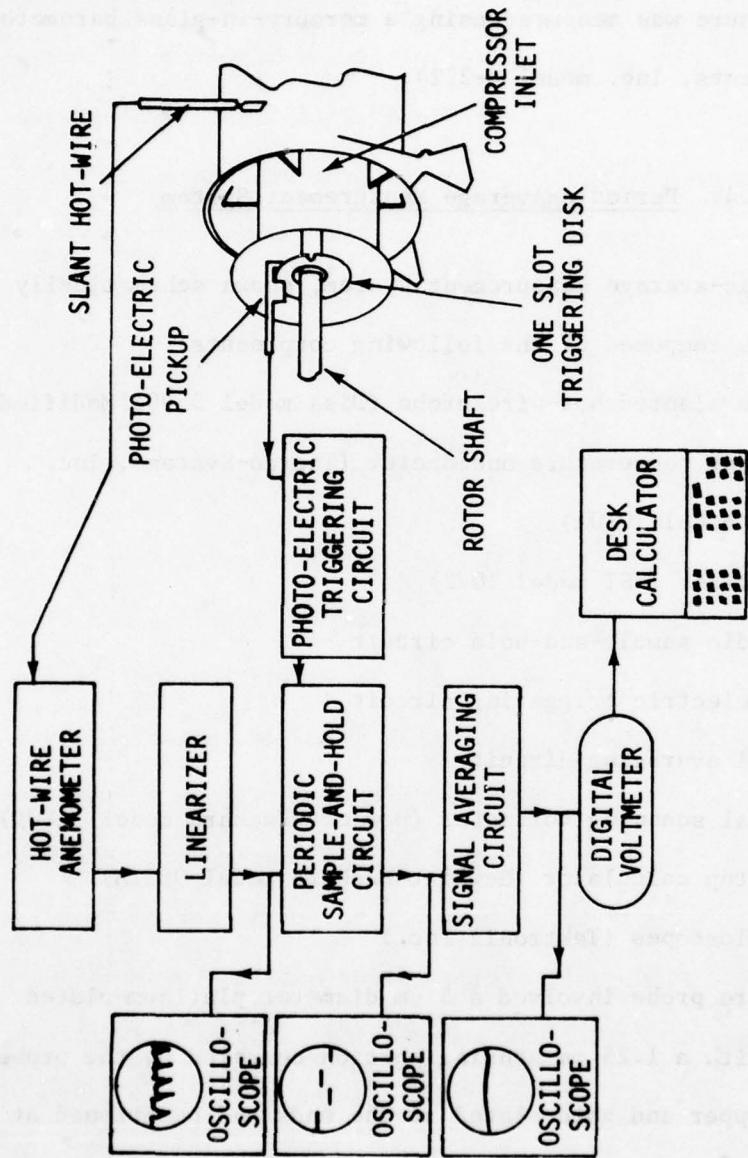


Figure 6. Schematic set-up diagram of periodic-average flow measurement system.

phase locked with the compressor rotor by the photoelectric pickup which was triggered by a slot in a disc rotating with the compressor shaft. The photoelectric pickup was connected to the stationary blade row actuator with an adjustable arm so that any desired rotor sampling position,  $Y_{0R}$ , could be obtained. The rotor sampling position was measured, in degrees, from the line of measurement stations, with the stationary blade rows set as indicated earlier (see section 2.2.). Figure 5 shows the relative position of the stationary blades and several rotor sampling positions. The sample-and-hold circuit was designed and constructed by the Iowa State University Engineering Research Electronic Shop to obtain a 5  $\mu$ sec\* sample with each revolution of the rotor shaft. Two scribe marks, one on the rotating portion of the hub surface and the other on the stationary portion of the hub, were used to set the rotor phase lock reference position  $Y_{0R}/S_R = 0.0$ . Once the reference condition was established, other rotor sampling positions were obtained, by moving the photoelectric pickup with respect to the stationary blade row actuator and by observing the amount of movement with mechanical scales. The reference setting was checked daily and set with an electronic time delay provision in the sample-and-hold circuit. The periodically sampled signal was electronically smoothed with the signal averaging circuit, subsequently read by the digital voltmeter, and finally arithmetically averaged and stored in the calculator for further reduction and tape storage.

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\*The typical rotor wake period is 1.1 msec.

A desk-top calculator and an associated multiple channel scanning digital voltmeter were used to read, store, and reduce data. The interfacing between the calculator and voltmeter enabled the calculator to selectively read different channels and store readings. The calculator possessed over 1400 memory registers. Additionally, with tape cassette capability, the calculator could be extended to store permanently on tape any data and programs to be recalled at the user's convenience. The plotting and tabulating of results were done with the Iowa State University Computation Center computing system (IBM 360/65, 370/158).

## 2.5. Calibration Nozzle

An air nozzle was used for the calibration of hot-wire sensors. The nozzle has a throat diameter of 25.4 mm (1.0 in.) and a contraction ratio of 144 to 1. The flow at the nozzle involved a uniform velocity profile for values of velocity from 0.0 to 50 m/s. Regulated compressed air provided the supply air and a variable-current heater, blower, and heat exchanger arrangement was used for air temperature control. A special probe holder permitted the sensing portion of the probe to remain in the same position in the nozzle flow while the probe yaw and pitch angles were varied. A telescope was used to visually align wires.

### 3. EXPERIMENTAL PROCEDURES AND DATA REDUCTION

All measurements presently involved were obtained with the periodic-average measurement system described earlier. The compressor operating point was held constant at the condition indicated in Figure 7 (1400 $\pm$ 1 rpm, 0.42 flow coefficient) with frequent monitoring and adjusting.

The standards for velocity, pressure, and temperature instrument calibrations were provided by the calibration nozzle, the micromanometer, and mercury-in-glass thermometers. Calibration of all electronic equipment was performed by the ISU Engineering Research Institute Electronic shop.

The desk-top programmable Hewlett-Packard calculator was used extensively for calibration, data acquisition, and data reduction. Programs were used to control step-by-step data acquisition, to make preliminary calculations, to print out data and preliminary results, and to record on tapes for further data reduction. Data reduction programs accepted the preliminary results and produced final results. A list and brief description of each data acquisition and reduction program used may be found in Appendix A.

#### 3.1. Periodic-Average Sampling Technique

The hot-wire anemometer output signal is composed of a periodic component and a random component. With controlled sampling and data averaging the random fluctuation influence can be made as small as desired, depending on the number of samples taken [4,7,18,23,25,

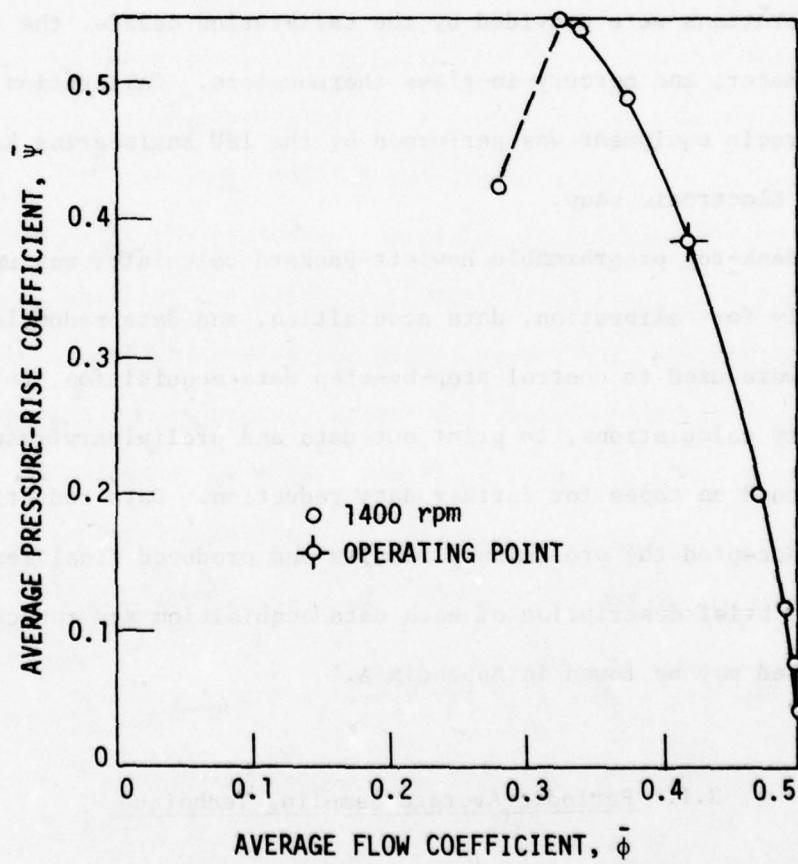


Figure 7. Research compressor performance curve and operating point.

26,28]. With the system used presently, electronic and arithmetic averaging of data could be accomplished. The electronic averaging was done with a low pass filter having a time constant of 1 sec. Schmidt and Okiishi [28] found the arithmetic average of 180 samples of electronically-smoothed data to be approximately equal to the arithmetic average of 1200 unfiltered data points. More current data indicated that 180 samples of electronically-smoothed data were sufficient to make the random component of the unsteady flow negligibly small except at the IGV row exit where 360 samples were needed because of the separated flow activity involved.

The data sampling system was triggered by a photoelectric pickup, connected to the circumferential actuator slider, and by a single-slot disc rotating with the compressor shaft. The photoelectric pickup could be moved relative to the stationary blade rows so that periodic-average "snapshots" of the flow field could be obtained for each of several time sequenced positions of the sampled rotor blades. Thus, periodic-average flow field changes could be measured for various sampled rotor positions. This kind of measurement was accomplished by moving the stationary blade rows and the photoelectric pickup relative to the stationary probe so that the probe was effectively made to traverse the flow field over one stator blade pitch spacing.

### 3.2. Single, Slanted Hot-Wire Three-Dimensional Velocity Measurement Technique

A single, slanted hot-wire was used to obtain three-dimensional flow field parameters. Before the measurement technique can be

discussed, some relationships linking probe geometry and hot-wire cooling velocities must be presented.

### 3.2.1. Probe Geometry

The hot-wire sensor, the probe coordinate system, and a general velocity vector are shown in Figure 8. The coordinate system is fixed to the probe with the x-z plane lying on the sensing portion of the probe and the probe axis and with the y-axis perpendicular to the x-z plane and centered on the sensor. The wire is slanted at an angle,  $\theta_0$ , to the x-axis. The velocity vector,  $\vec{V}$ , can be resolved into orthogonal components along x, y, and z for each orientation of the wire. When the probe and its coordinate system are rotated about the z-axis, the projected yaw angle,  $\theta_y$ , changes by the amount of turning, whereas, the pitch,  $\theta_p$ , remains the same. The sensor yaw angle,  $\alpha$ , was defined as the angle between the intersection of the probe sensor unit vector,  $\vec{A}$ , and the velocity vector,  $\vec{V}$ . To obtain a relationship between  $\alpha$  and  $\theta_0$ ,  $\theta_p$ , and  $\theta_y$ , the dot product of the two vectors is taken:

$$\vec{A} = \cos \theta_0 \vec{i} + \sin \theta_0 \vec{k} \quad (1)$$

$$\vec{V} = -V \cos \theta_p \cos \theta_y \vec{i} - V \cos \theta_p \sin \theta_y \vec{j} - V \sin \theta_p \vec{k} \quad (2)$$

$$\begin{aligned} \vec{A} \cdot \vec{V} &= |\vec{A}| |\vec{V}| \cos(180 - \alpha) = -|\vec{V}| \cos \theta_0 \cos \theta_p \cos \theta_y \\ &\quad - |\vec{V}| \sin \theta_0 \sin \theta_p \end{aligned} \quad (3)$$

$$\therefore \cos \alpha = \cos \theta_0 \cos \theta_p \cos \theta_y + \sin \theta_0 \sin \theta_p \quad (4)$$

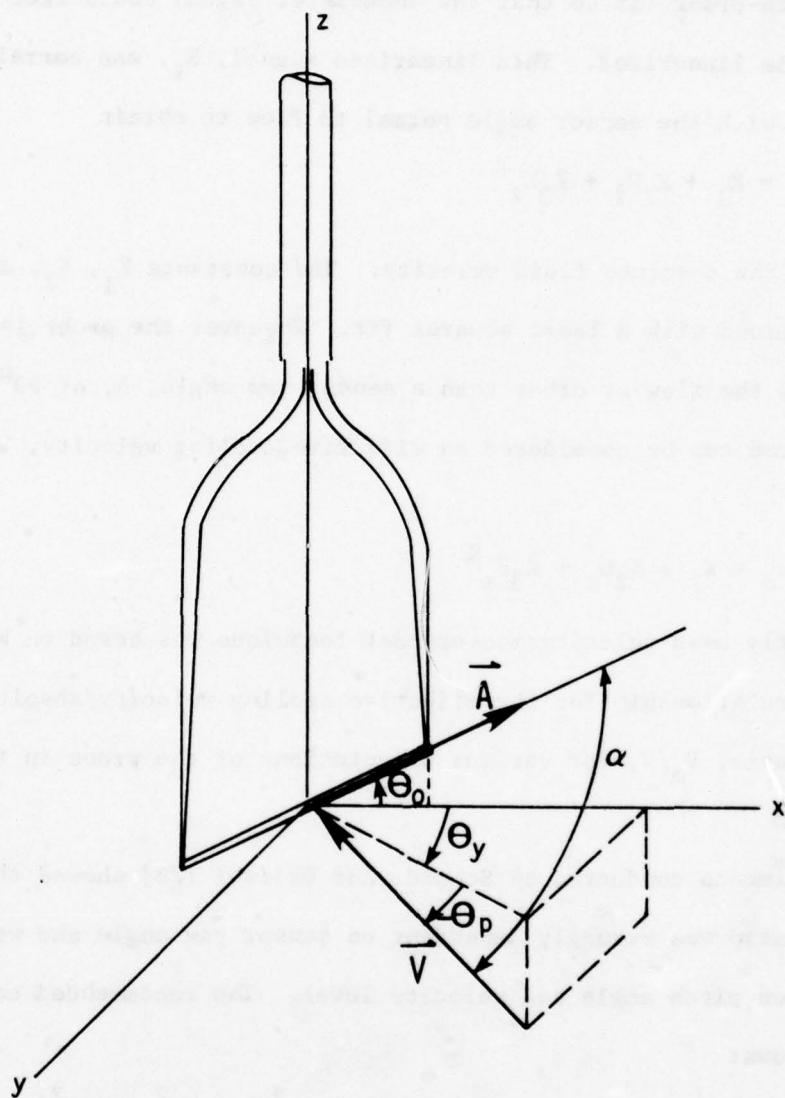


Figure 8. Hot-wire configuration relating velocity vector,  $\vec{V}$ , to hot-wire sensor and probe coordinates  $x$ ,  $y$ ,  $z$ .

### 3.2.2. Effective Cooling Velocity Ratio

The hot-wire anemometer output was correlated to fluid velocity with a fourth-order fit so that the anemometer signal could electronically be linearized. This linearized signal,  $E_\ell$ , was correlated to velocity with the sensor angle normal to flow to obtain

$$V = K_1 + K_2 E_\ell + K_3 E_\ell^2 \quad (5a)$$

where  $V$  is the absolute fluid velocity. The constants  $K_1$ ,  $K_2$ , and  $K_3$  were determined with a least squares fit. Whenever the probe is oriented to the flow at other than a sensor yaw angle,  $\alpha$ , of  $90^\circ$ , the velocity read can be considered an effective cooling velocity,  $V_e$ , where

$$V_e = K_1 + K_2 E_\ell + K_3 E_\ell^2 \quad (5b)$$

The presently used velocity measurement technique was based on knowing a precise relationship for the effective cooling velocity/absolute velocity ratio,  $V_e/V$ , for various orientations of the probe in the flow stream.

Experiments conducted by Schmidt and Okiishi [28] showed this velocity ratio was strongly dependent on sensor yaw angle and weakly dependent on pitch angle and velocity level. The recommended correlation is as follows:

$$\begin{aligned} V_e/V = & b_0 + b_1 \alpha + b_2 \theta_p + b_3 V + b_4 \alpha^2 + b_5 \theta_p^2 + b_6 V^2 \\ & + b_7 \alpha \theta_p + b_8 \alpha V + b_9 \theta_p V \end{aligned} \quad (6)$$

The coefficients  $b_0$  through  $b_9$  were determined from a least squares fit of data as described in the calibration procedure section (see section 3.3.3.).

### 3.2.3. Measurement Technique

The wire was rotated in the flow field to three different positions denoted as a, b, and c (see Figure 9). These probe positions relate to yaw angles of  $\theta_{y,a}$ ,  $\theta_{y,b}$ , and  $\theta_{y,c}$  which were set as indicated below:

$$\theta_{y,a} = \theta_y \quad (7)$$

$$\theta_{y,b} = \theta_y - m_b \quad (8)$$

$$\theta_{y,c} = \theta_y - m_c \quad (9)$$

where  $m_b$  and  $m_c$  are probe turning angle increments from the a location. For each orientation of the wire, effectively 1200 samples were taken and electronically and arithmetically averaged to obtain a time-average effective cooling velocity for that particular probe position. For each periodic-average velocity vector measurement, a total of six equations like Eqs. (4) and (6) were obtained for the orientations of the wire. These equations are:

#### For position a

$$\begin{aligned} V_{e,a}/V = & b_0 + b_1 \alpha_a + b_2 \theta_p + b_3 V + b_4 \alpha_a^2 + b_5 \theta_p^2 \\ & + b_6 V^2 + b_7 \alpha_a \theta_p + b_8 \alpha_a V + b_9 \theta_p V \end{aligned} \quad (10)$$

$$\cos \alpha_a = \cos \theta_0 \cos \theta_p \cos \theta_{y,a} + \sin \theta_0 \sin \theta_p \quad (11)$$

#### For position b

$$\begin{aligned} V_{e,b}/V = & b_0 + b_1 \alpha_b + b_2 \theta_p + b_3 V + b_4 \alpha_b^2 + b_5 \theta_p^2 \\ & + b_6 V^2 + b_7 \alpha_b \theta_p + b_8 \alpha_b V + b_9 \theta_p V \end{aligned} \quad (12)$$

$$\cos \alpha_b = \cos \theta_0 \cos \theta_p \cos \theta_{y,b} + \sin \theta_0 \sin \theta_p \quad (13)$$

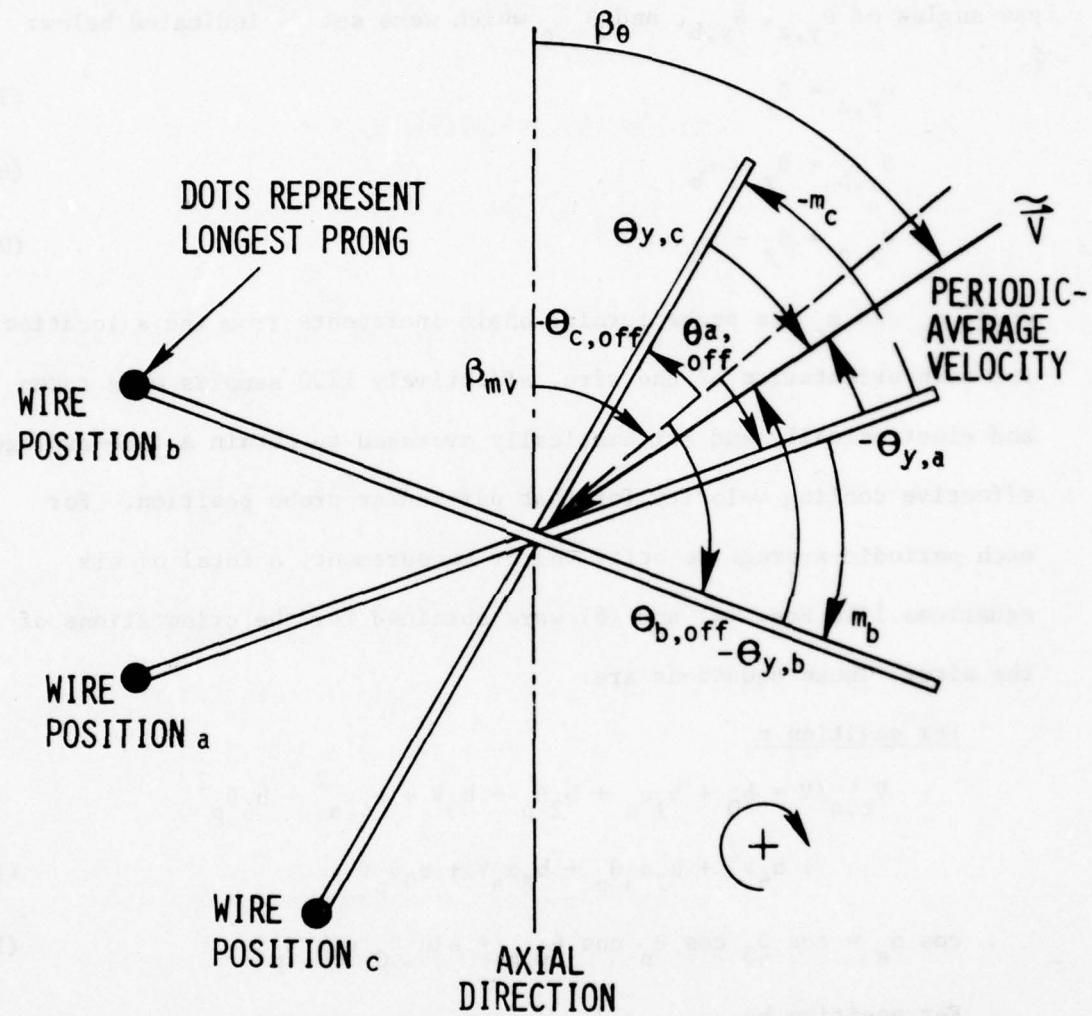


Figure 9. Hot-wire measurement positions and nomenclature, viewed from above along probe axis.

For position c

$$\begin{aligned} v_{e,c}/v = & b_0 + b_1 \alpha_c + b_2 \theta_p + b_3 v + b_4 \alpha_c^2 + b_5 \theta_p^2 \\ & + b_6 v^2 + b_7 \alpha_c \theta_p + b_8 \alpha_c v + b_9 \theta_p v \end{aligned} \quad (14)$$

$$\cos \alpha_c = \cos \theta_0 \cos \theta_p \cos \theta_{y,c} + \sin \theta_0 \sin \theta_p \quad (15)$$

By substituting Eqs. (7), (8), and (9) into Eqs. (11), (13), and (15), the six unknown variables  $\alpha_a$ ,  $\alpha_b$ ,  $\alpha_c$ ,  $\theta_p$ ,  $\theta_y$ , and  $v$  remain in the six equations, (10) through (15). These equations were solved simultaneously using the Newton-Raphson method. The three-dimensional velocity vector is completely described with the variables  $\theta_p$ ,  $\theta_y$ , and  $v$  known relative to the probe position, a.

To set the three wire orientations, a minimum effective velocity angle,  $\beta_{mv}$ , was obtained as shown in Figure 9. From this minimum effective velocity angle position, the probe was rotated to positions a, b, and c according to the following schedule:

$$\theta_{a,off} = 20^\circ$$

$$\theta_{b,off} = 60^\circ$$

$$\theta_{c,off} = -20^\circ$$

From Figure 9, it can be seen that

$$\theta_{y,a} = \theta_y \approx -20^\circ$$

$$\theta_{y,b} = \theta_y - m_b \approx -60^\circ$$

$$\theta_{y,c} = \theta_y - m_c \approx 20^\circ$$

and

$$m_b = 40^\circ$$

$$m_c = -40^\circ$$

This positioning of the probe was done for each velocity vector obtained. For all of the measurements, the probe was positioned so that the longer wire support was downstream of the sensing portion of the wire to minimize prong interference.

### 3.3. Calibration Procedures

The calibration nozzle was used to provide a uniform velocity air stream for velocity calibration. Plenum pressures were observed with inclined water-in-glass manometers. Velocity was calculated using the following equation:

$$V = \sqrt{\frac{2g_c \gamma_{H_2O} \Delta P_n}{\rho}} \quad (16)$$

where

$V$  = velocity, m/s

$g_c$  = gravitational constant  $1.0 \text{ kg m/Ns}^2$

$\gamma_{H_2O}$  = specific weight of water,  $\text{N/m}^3$

$\rho$  = density of air,  $\text{kg/m}^3$

$\Delta P_n$  = differential pressure between plenum pressure and atmospheric pressure, meters of water

Calibration programs were used with the desk-top calculator.

Sufficient warm-up time was allowed to obtain steady state conditions.

Fluid temperatures, barometric conditions, and test conditions were entered into the calculator before and after each calibration. Three hot-wire calibrations were made: linearizer velocity calibration, second order velocity calibration, and effective cooling velocity ratio calibration.

### 3.3.1. Linearizer Velocity Calibration

The linearizer was used to approximate the anemometer output with a fourth order polynomial. The "zero" degree term of the polynomial was set equal to zero, and coefficients were determined by a least squares curve fit, thus producing a linear relationship between velocity and anemometer output with the wire positioned normal to the flow of the calibration nozzle.

The wire was optically positioned 6.35 mm (0.25 in.) away from the nozzle exit plane. With the air velocity set close to the average value of the range of velocities anticipated, the cold combined resistance of the sensor, cables, and leads was measured,  $R_{s,c,d}$ . The operating resistance of the sensor was calculated with an overheat ratio of 1.8. Because the cable and lead resistances were not involved with the overheat calculation, they were subtracted from the cold resistance reading at the anemometer deck and then subsequently added back on as demonstrated in Eq. (17).

$$R_{s,op,d} = 1.8 (R_{s,c,d} - R_{cb} - R_{ph} - R_{pl}) \\ + R_{cb} + R_{ph} + R_{pl} \quad (17)$$

where

$R_{s,op,d}$  = deck setting of sensor operating resistance, ohms

$R_{s,c,d}$  = deck reading of combined sensor, cable, and lead resistance, ohms

$R_{cb}$  = cable resistance, ohms

$R_{ph}$  = probe holder resistance, ohms

$R_{pl}$  = probe lead resistance, ohms

Probe holder and cable resistance were measured with an impedance bridge while the probe lead resistance was taken as furnished by the manufacturer. The linearizer calibration system included the hot-wire, anemometer, DVM, and desk-top calculator only. The linearizer was correlated over a range of nozzle air velocities (0.0 to 23 m/s) in fifteen increments. The linearized polynomial velocity was compared to the actual nozzle velocity. The acceptance criterion for the points was that the error should be less than or equal to 1%. Linearizer calibration was performed with every effective cooling velocity ratio calibration.

### 3.3.2. Second Order Velocity Calibration

A second order correlation, Eq. (5), between the linearized anemometer output and velocity level with the wire normal to the flow was performed before and during effective cooling velocity calibration and data acquisition to avoid error due to electronic equipment drift.

The second order calibration was done with the wire positioned normal to the nozzle flow. Any other wire yaw angle does not indicate an absolute velocity but rather an effective cooling velocity,  $V_e$ . The second order calibration system included the probe, anemometer, linearizer, DVM, and desk-top calculator. Fourteen points were taken for each calibration for air velocities ranging from 4 to 23 m/s. Acceptable error was confined to less than 1% in the velocity working range.

### 3.3.3. Effective Cooling Velocity Calibration

The effective cooling velocity calibration was performed to obtain the coefficients in the effective cooling velocity ratio equation:

$$\begin{aligned} V_e/V = & b_0 + b_1 \alpha + b_2 \theta_p + b_3 V + b_4 \alpha^2 + b_5 \theta_p^2 \\ & + b_6 V^2 + b_7 \alpha \theta_p + b_8 \alpha V + b_9 \theta_p V \end{aligned} \quad (6)$$

The calibration was made over the range of expected compressor yaw and pitch angles and velocities.

The probe was positioned in the nozzle jet stream in the same manner as for the linearizer calibration. For a particular velocity level, the probe was set at each of six pitch angles and rotated from 0 to 90° in yaw angle increments of 5°. This procedure was continued until data for all of the velocity levels (11.6, 15.2, 19.2, 22.3 m/s) and pitch angles (-9 to 6°, in increments of 3°) were acquired. This procedure was also used for probe yaw angles of 0 to -90° because of the lack of perfect symmetry wire response to yaw angle. Thus, during data reduction, two sets of coefficients ( $b_0$  through  $b_9$ ) were used depending on the yaw orientation of the wire. Positions a and b involved the 0 to -90 calibration coefficients while position c used the 0 to +90 calibration coefficients.

After all the data were taken, the coefficients were solved using a least squares method. Acceptable error was limited to being less than 2% with the majority of the errors less than 1%.

### 3.4. Data Acquisition

The data acquisition system used was presented earlier in schematic form in Figure 6. Prior to taking data, equipment and room temperature equilibrium were established, inclined manometers zeroed, cold resistance of the wire at the deck set, flow coefficient set, linearizer

coefficients adjusted, second order calibration performed, probe actuator positioned in the compressor, and radial location set. After setting and adjusting the time delay device for  $Y_0/S_R = 0.0$ , the desired rotor sampling position could be set by adjusting the photoelectric pickup position relative to the stationary blade rows.

With the probe sensor positioned at  $90^\circ$  (normal to the compressor axis), a trace of the blade-to-blade variation of effective cooling velocity was made with an x-y storage oscilloscope where x was the potentiometer reading from the circumferential actuator, and y was the time-averaged hot-wire sampled signal. This survey was made by moving the stationary blade rows and photoelectric pickup relative to the probe. By using this procedure, the probe was effectively made to traverse the flow field for one stator blade pitch. From this x-y trace, the position of the wake could be detected and test points appropriately scheduled. The circumferential actuator system was then set back to zero,  $Y/S_s = 0.0$ , for the first test data point. With another x-y storage oscilloscope, a trace of effective cooling velocity against probe yaw angle was repeatedly obtained by rotating the probe about its axis. In this fashion, a minimum effective cooling velocity angle,  $\beta_{mv}$ , could be discerned. From this minimum angle position, the probe could be appropriately positioned to locations a, b, and c, discussed earlier (see Figure 9). After periodically sampled data were obtained, with the probe at each position (a, b, and c), the circumferential actuator was shifted to a new position,  $Y/S_s$ , and the above described procedure was repeated. Thirty velocity vectors from blade-to-blade were measured for each test run. After the last velocity

vector of a set of thirty was measured, the acquired data were stored on tape for further reduction. The probe was then removed from the compressor and a new second order velocity calibration was made. Second order velocity calibrations were performed before each set of thirty velocity vectors was measured.

### 3.5. Data Reduction

Data reduction programs for the desk-top calculator were used to accept and reduce stored data. The six equations, Eqs. (10) through (15), were solved simultaneously with the Newton-Raphson method. Usually, less than five iterations were necessary for convergence.

The compressor coordinate system is shown in Figure 10. The R coordinate direction is positive outward from the hub, while the  $\theta$  coordinate is positive in the direction of rotation. The Z-axis is aligned axially, positive in the direction of fluid motion. Also shown are the sign conventions relating to  $\beta_\theta$ ,  $\beta_r$ ,  $V_r$ ,  $V_z$ , and  $V_\theta$ . With the probe immersed in the compressor annulus, the probe coordinates x and y (see Figure 8) are always in the same plane as Y and Z of the compressor coordinate system, and z from the probe coordinate system coincides with the R compressor coordinate direction.

The calculated pitch angle and radial angle are related by:

$$\beta_r = -\theta_p$$

The tangential angle is related to the calculated yaw,  $\theta_y$ , by:

$$\beta_\theta = \beta_{mv} + \theta_{a,off} + \theta_y$$

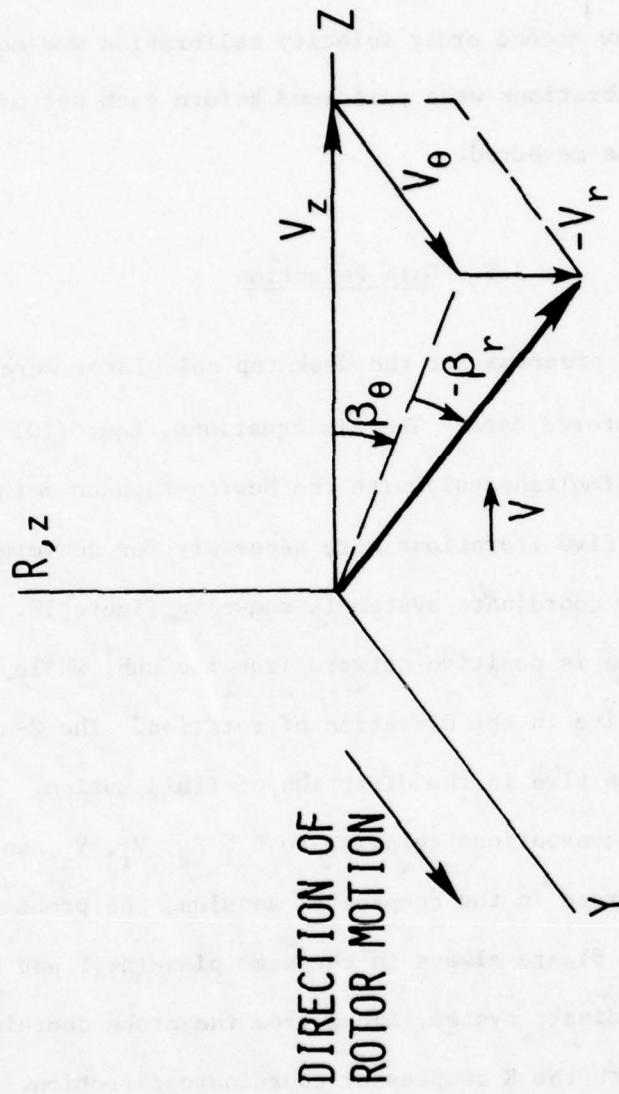


Figure 10. Compressor coordinate system showing nomenclature and sign convention for three-dimensional periodic-average velocity and angle parameters.

The values of  $\beta_{mv}$  and  $\theta_{a,off}$  were known from data acquisition. The other flow parameters calculated are presented below:

- (1) Axial velocity, m/s, Eq. (B-15)
- (2) Absolute tangential velocity, m/s, Eq. (B-16)
- (3) Radial velocity, m/s, Eq. (B-14)
- (4) Relative velocity, m/s, Eq. (B-17)
- (5) Relative tangential velocity, m/s, Eq. (B-18)
- (6) Relative tangential angle, degrees, Eq. (B-19)

A complete list of equations used appears in Appendix B.

## 4. PRESENTATION AND DISCUSSION OF DATA

The test results are presented and discussed in this section.

Primary flow variables involved are tabulated in Appendix C. Velocity and flow angle component variation graphs (scalar plots) are used to present the periodic-average data (see Figures 11, 12, and 13). The scalar plots include axial, tangential, and radial velocities supplemented with tangential, radial, and, when useful, relative tangential angles. Uncertainty and accuracy are previously discussed by Schmidt and Okiishi [28]. To reflect and facilitate data interpretation, blade-to-blade plane flow plots (cascade plots), showing the periodic-average location of blade wake flows at different radii for several distinct orientations of the rotor blades, were constructed (see Figure 14). The procedure used to construct the cascade plots is explained in detail below. A physical explanation of the wake transport and interaction effects involved is proposed.

#### 4.1. Construction of Cascade Wake Plots

Stationary blade section profiles were positioned on the plots to scale according to the minimum sound schedule (see section 2.2.). The rotor blade sections were located at their periodic sampling positions.

The obvious wake region locations suggested by the periodic-average flow data were drawn on the cascade plots first. It should be noted that periodic-average flow data collection was limited to thirty points along the broken lines connecting crosses between blade rows,  $Y/S_S =$

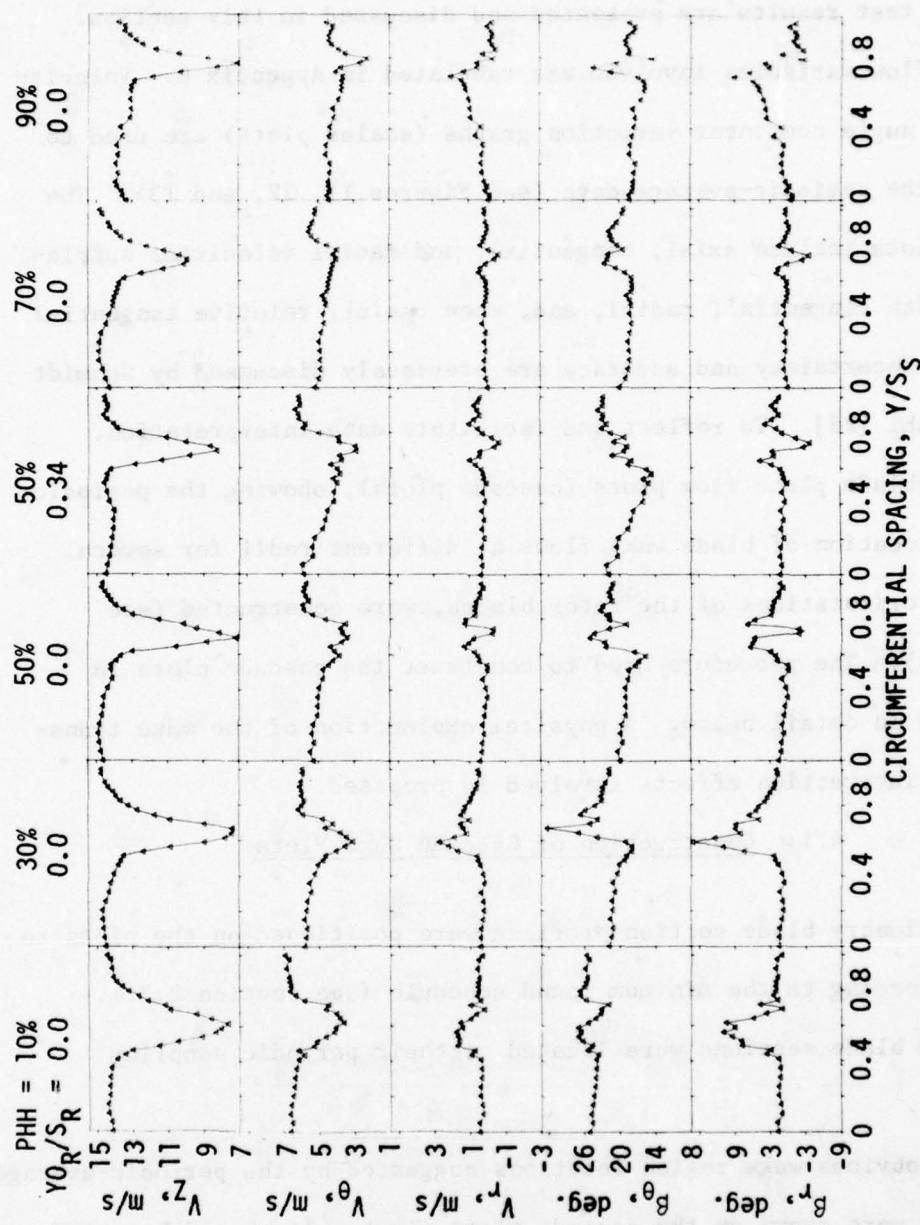
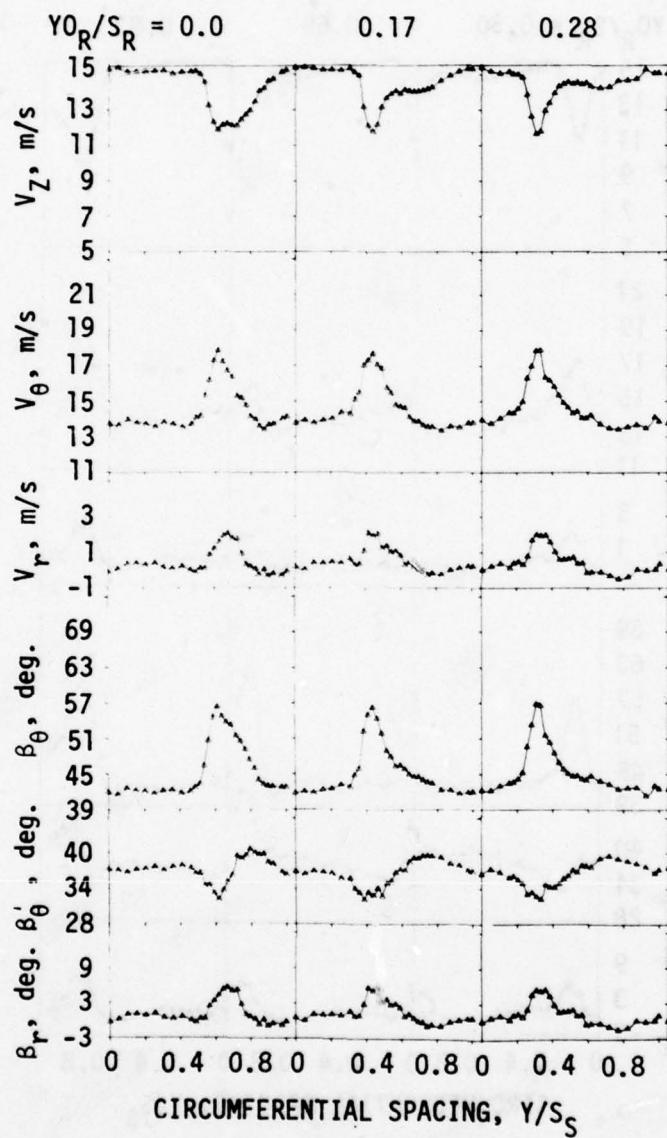
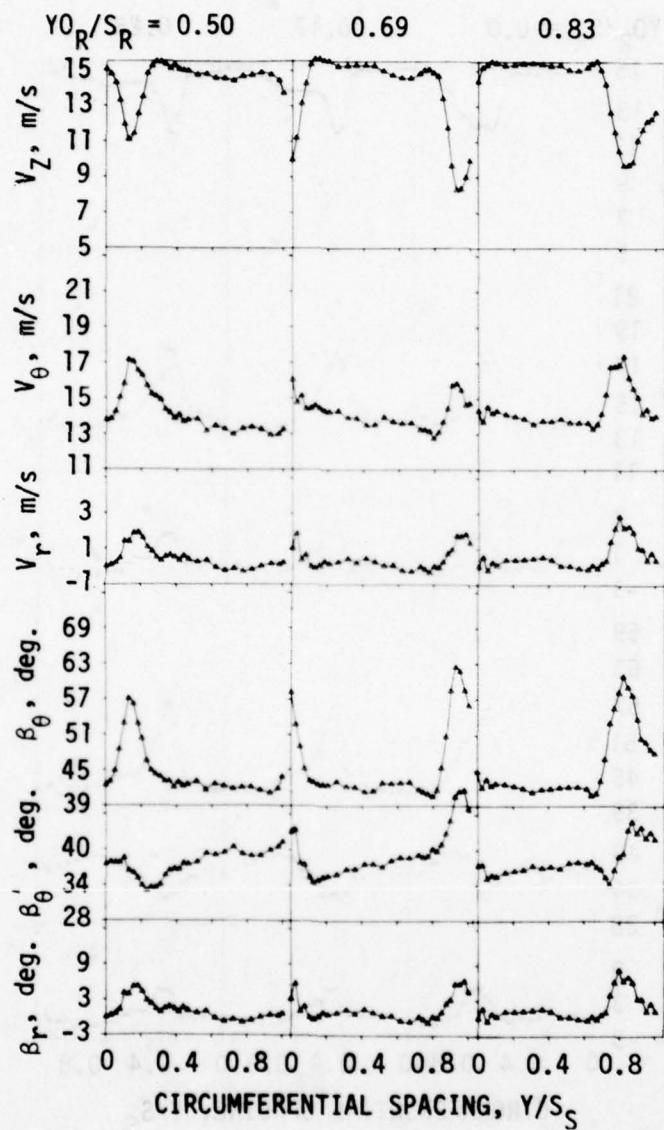


Figure 11. Blade-to-blade distribution of periodic-average flow-field parameters.  
Inlet guide vane exit flow.



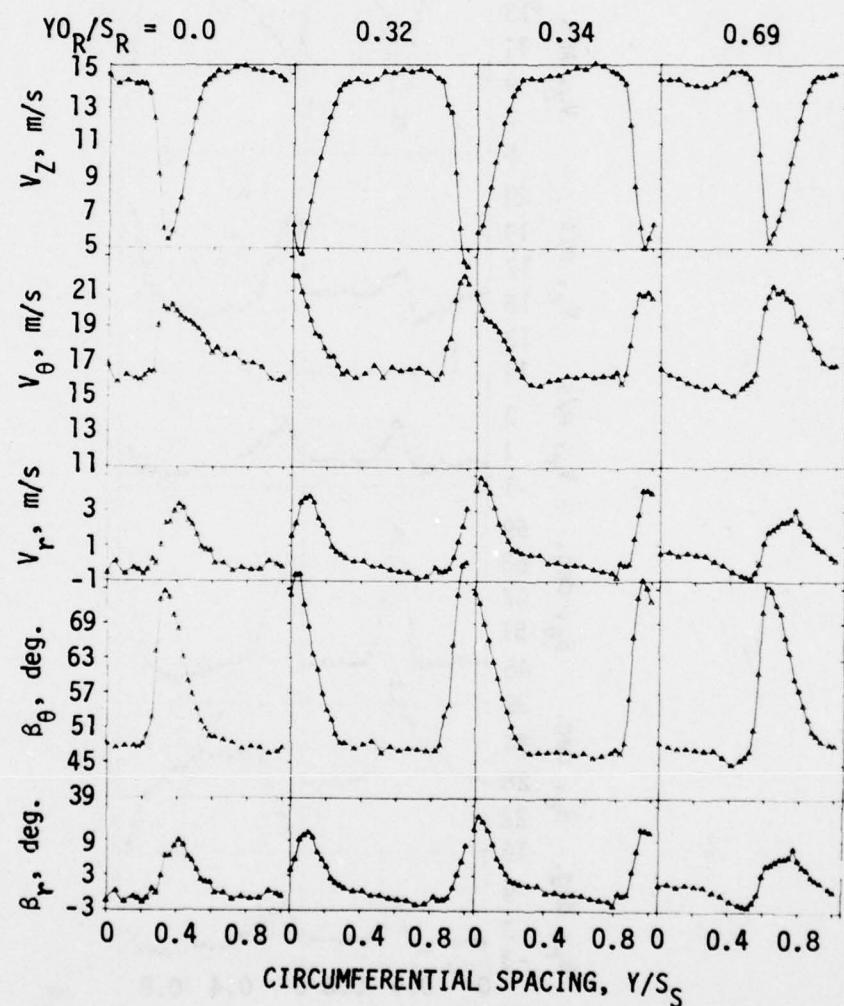
(a) First rotor exit flow at 50% passage height from hub.

Figure 12. Blade-to-blade distribution of periodic-average flow-field parameters. First rotor exit flow.



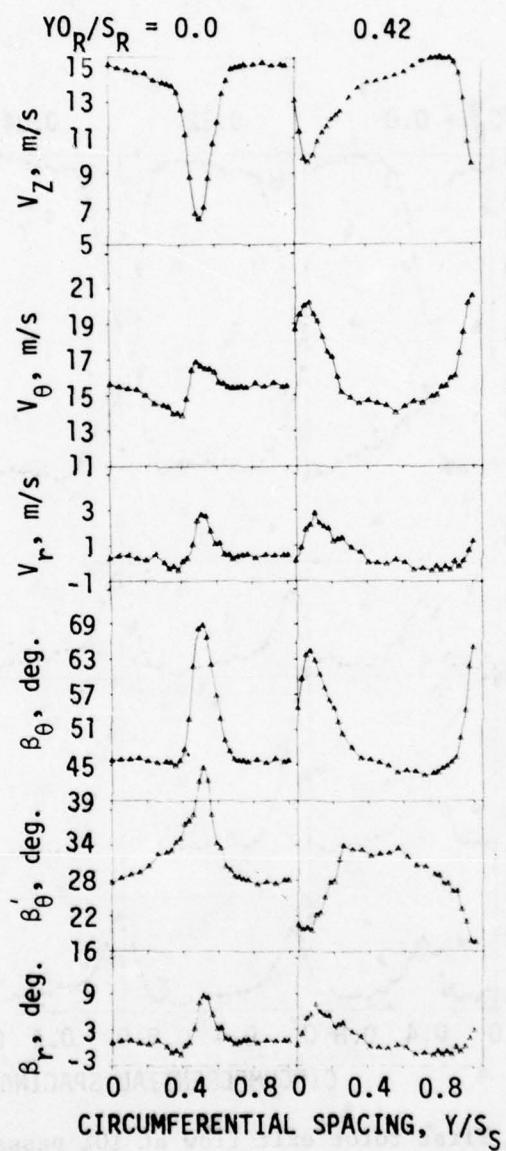
(a) (Continued).

Figure 12. (Continued).



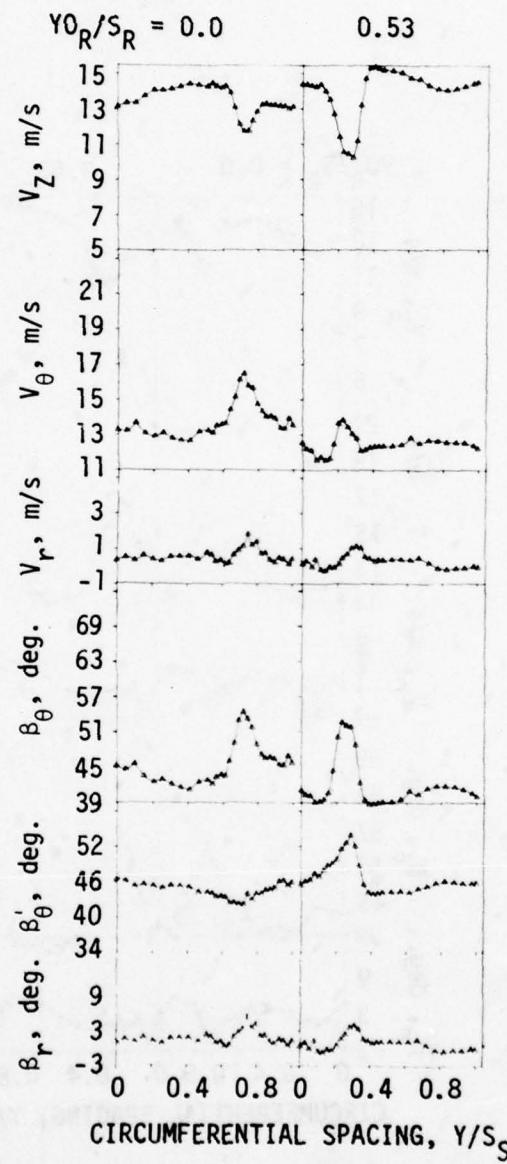
(b) First rotor exit flow at 10% passage height from hub.

Figure 12. (Continued).



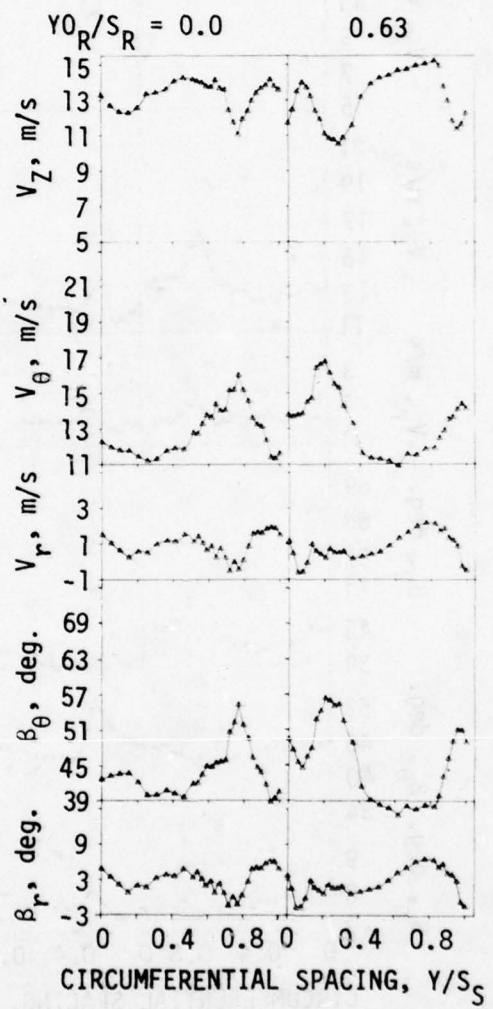
(c) First rotor exit flow at 30% passage height from hub.

Figure 12.1 (Continued).



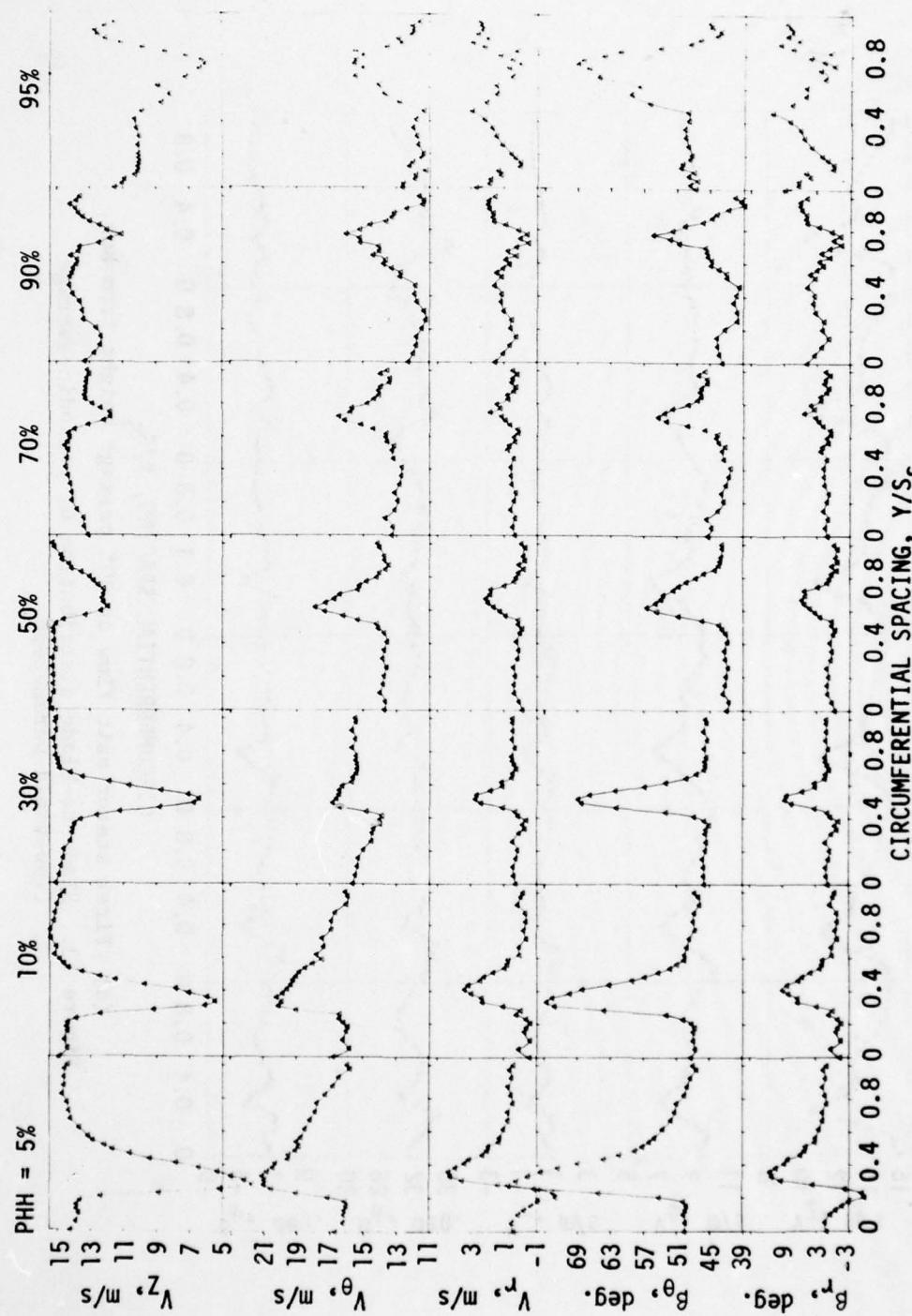
(d) First rotor exit flow at 70% passage height from hub.

Figure 12. (Continued).



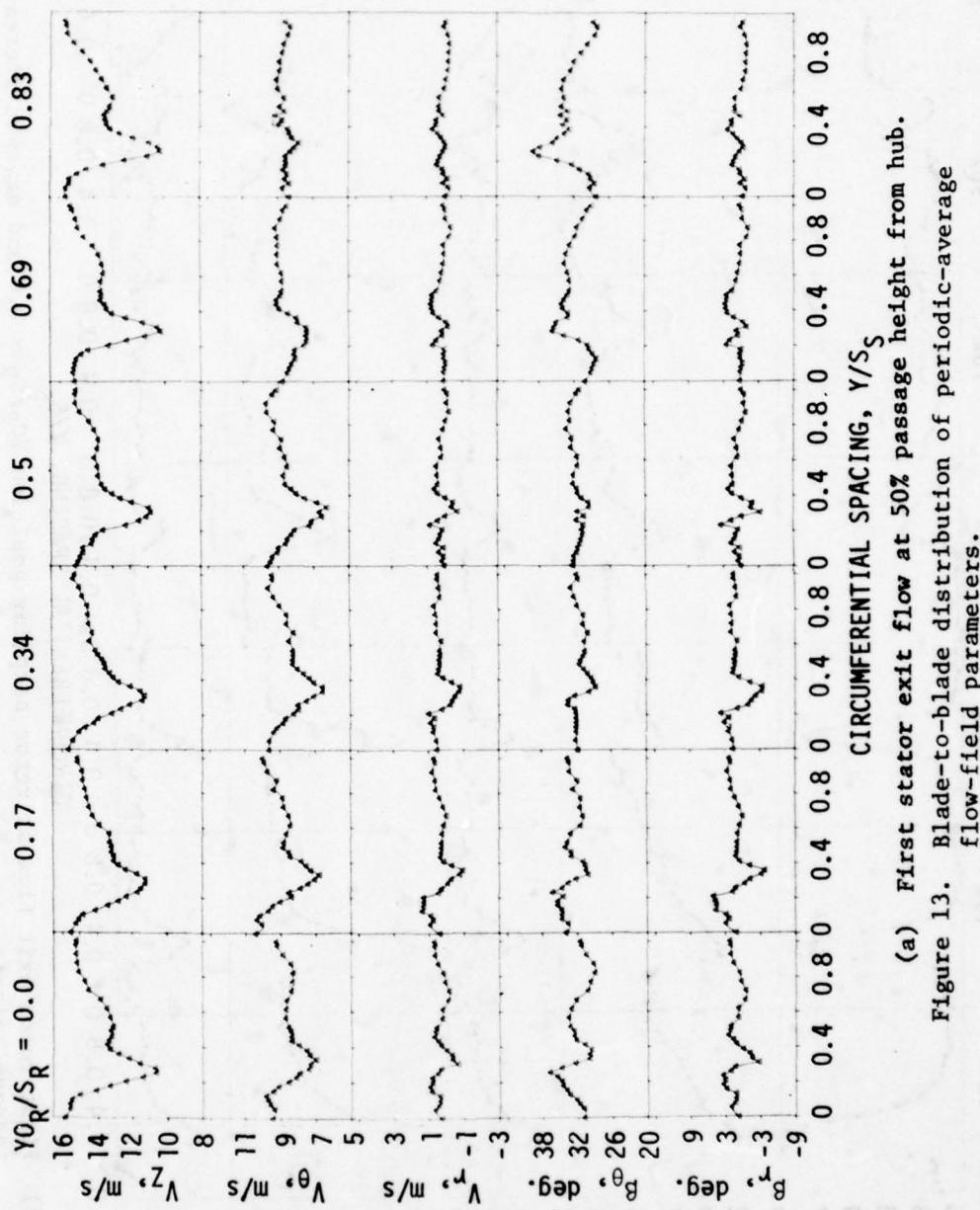
(e) First rotor exit flow at 90% passage height from hub.

Figure 12. (Continued).



(f) First rotor exit flow at rotor sampling position  $Y_{\text{OR}}/S_{\text{R}} = 0.0$  and seven different passage heights.

Figure 12. (Continued).



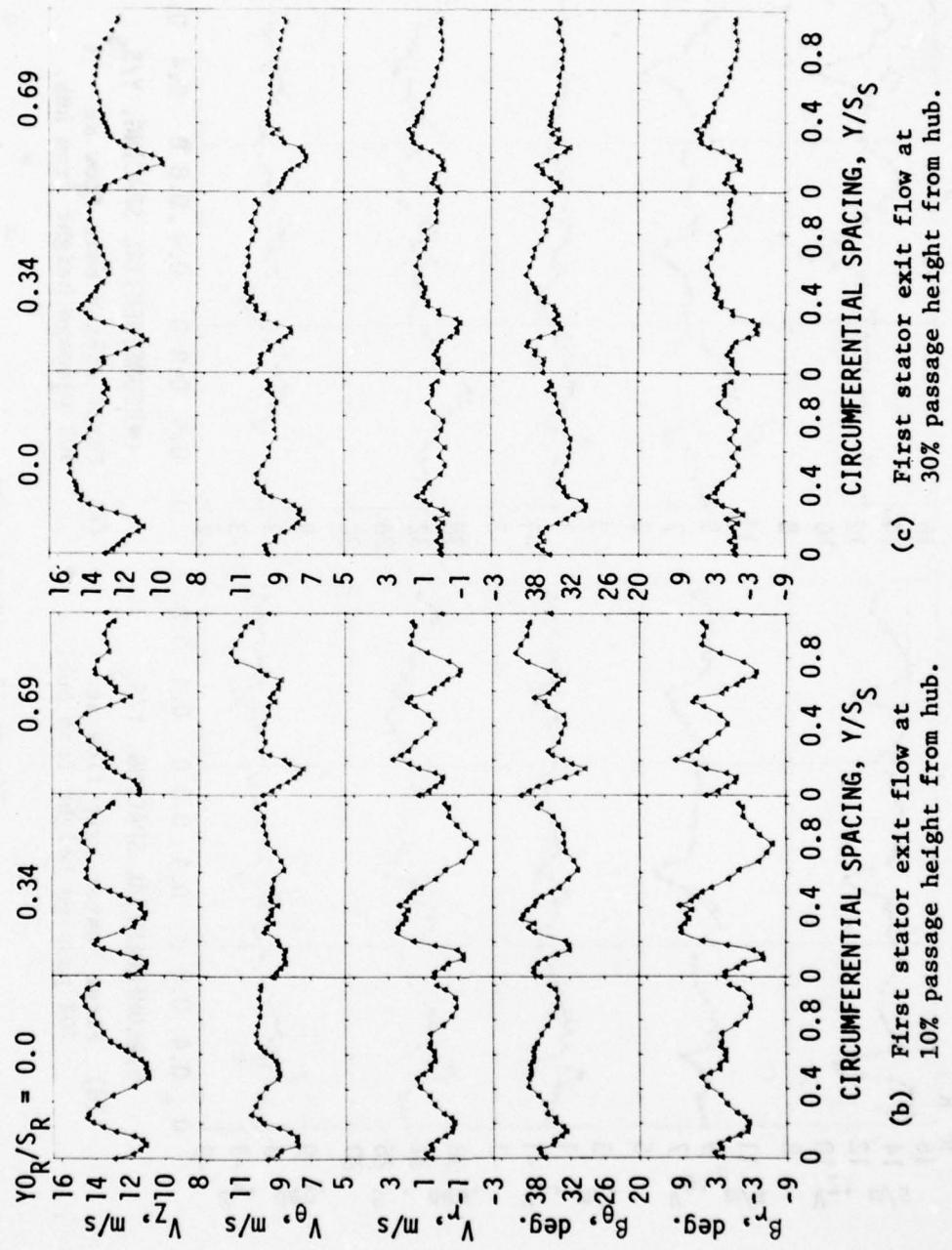


Figure 13. (Continued).

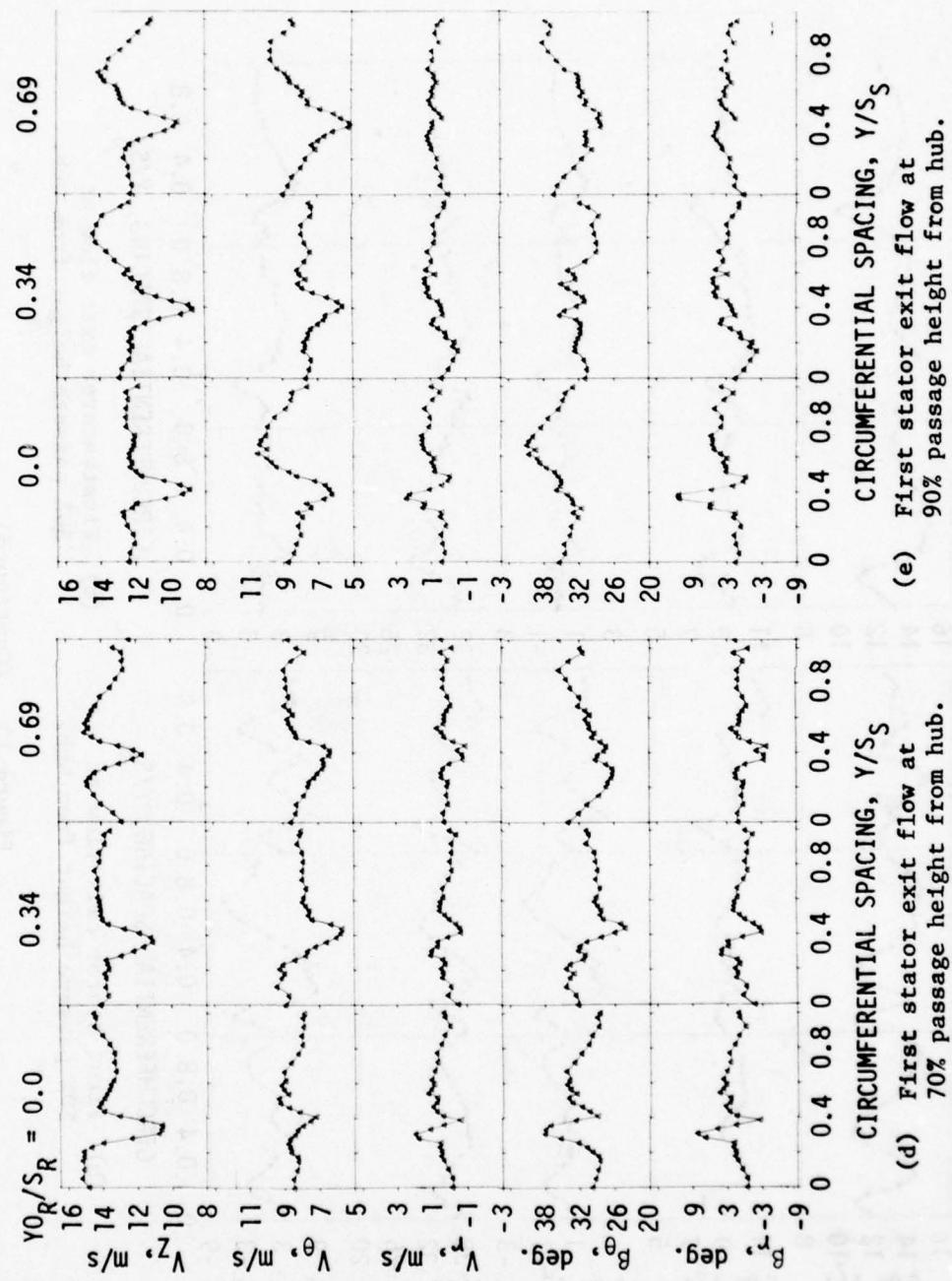
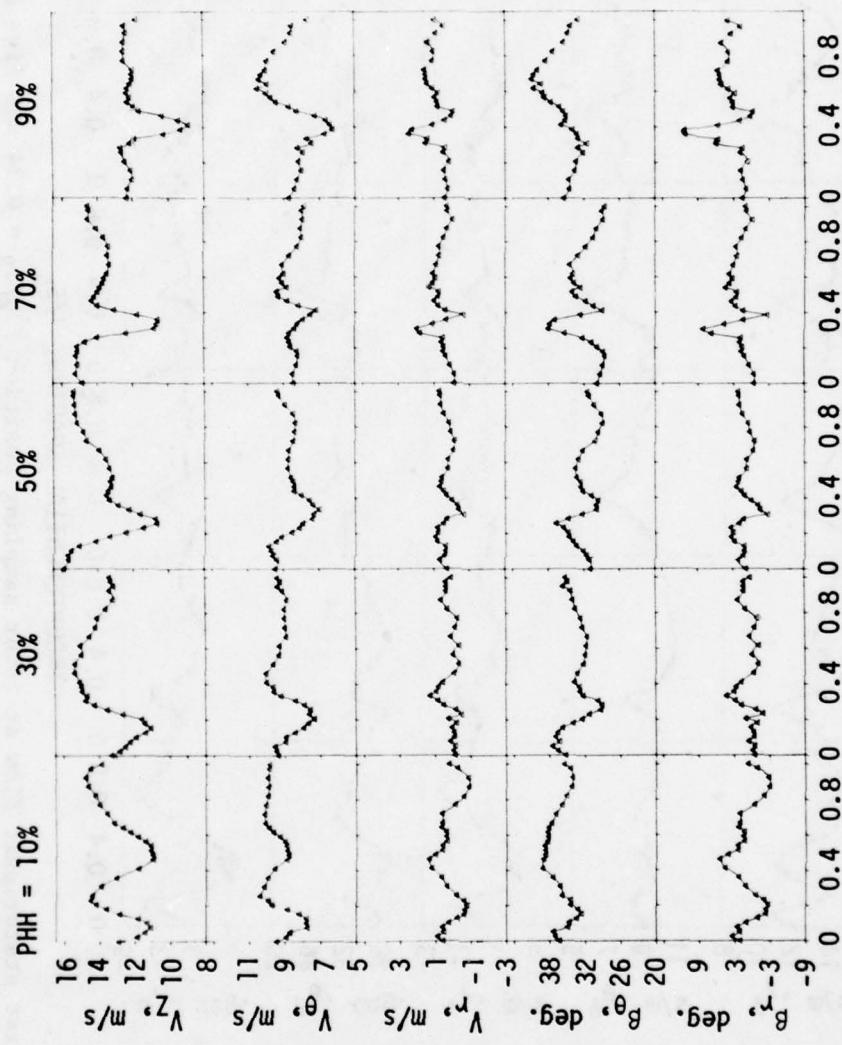
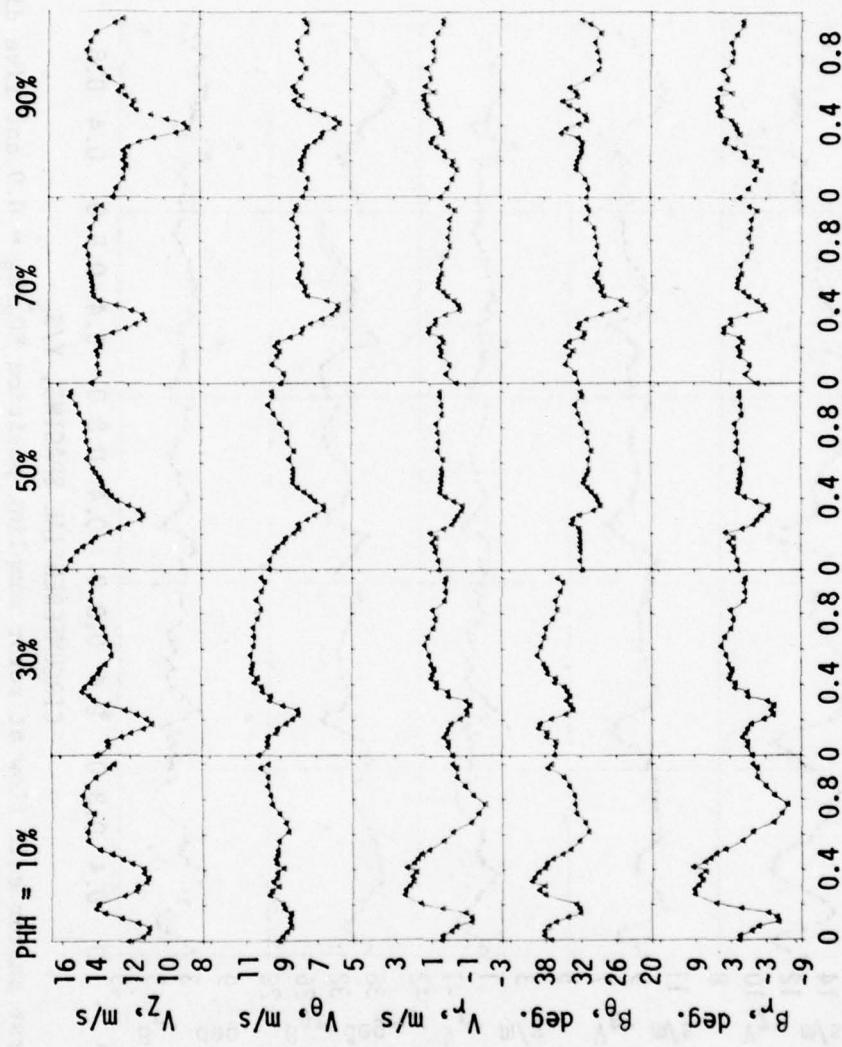


Figure 13. (Continued).



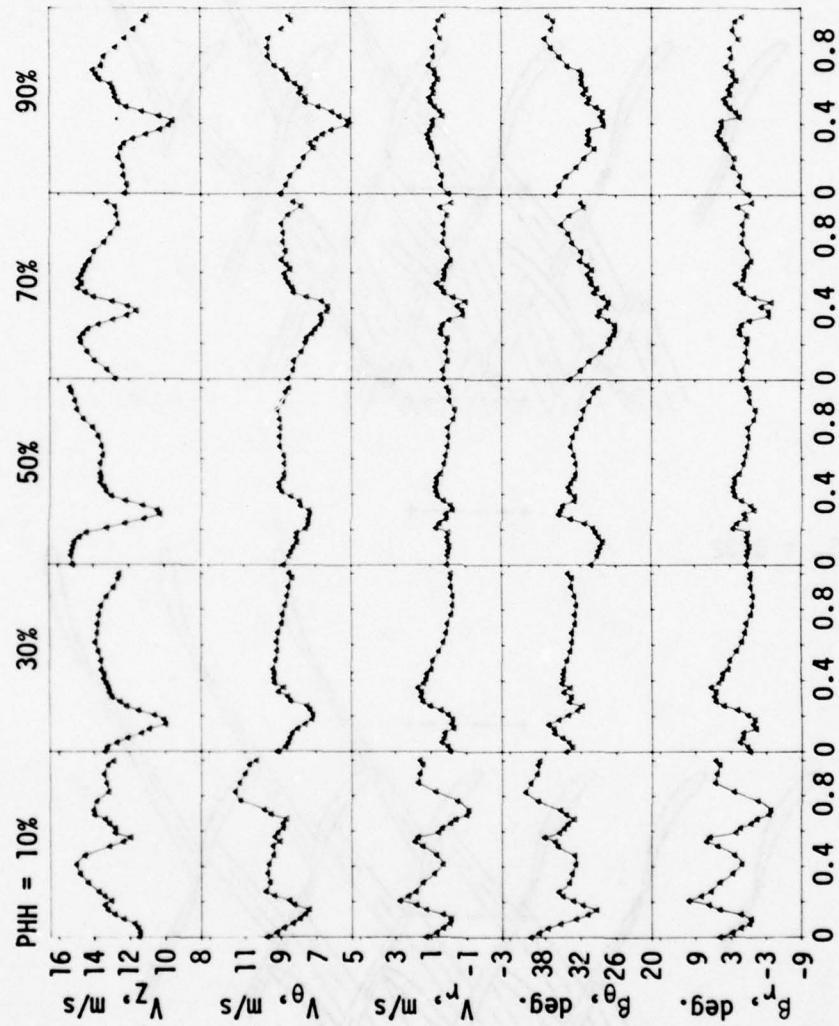
(f) First stator exit flow at rotor sampling position  $y/s_s = 0.0$  and five different passage heights.

Figure 13. (Continued).



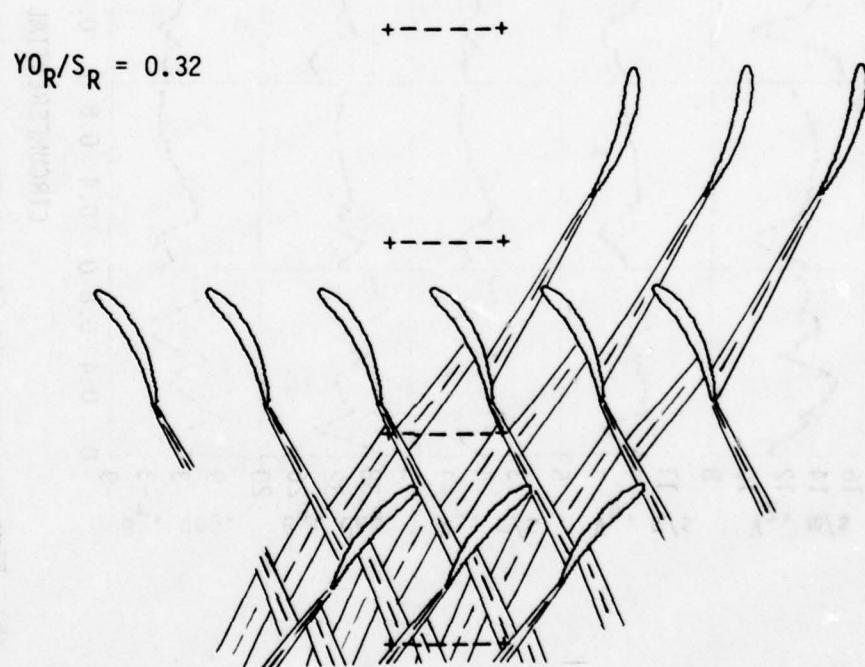
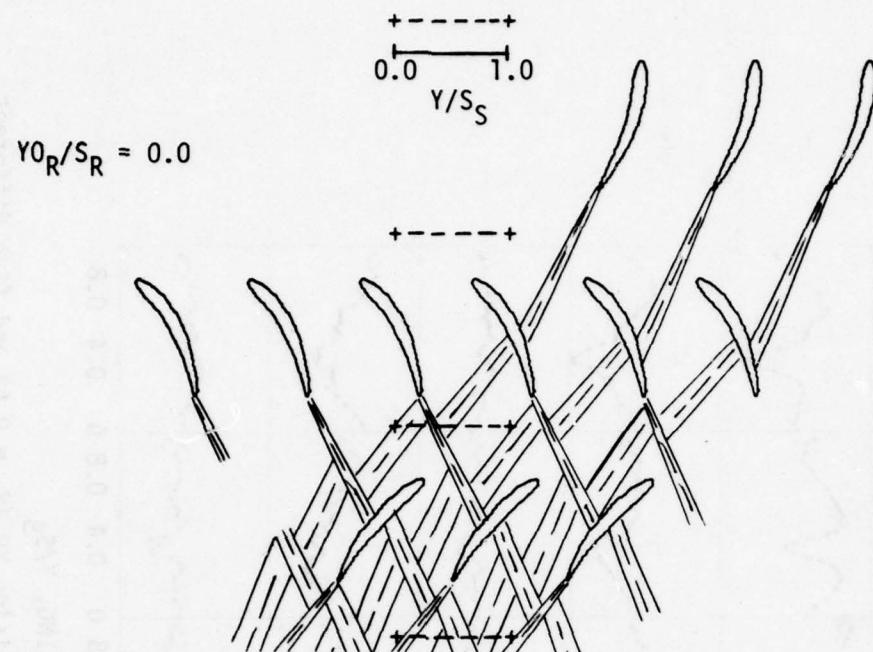
(g) First stator exit flow at rotor sampling position  $Y_R/S_R = 0.34$  and five different passage heights.

Figure 13. (Continued).



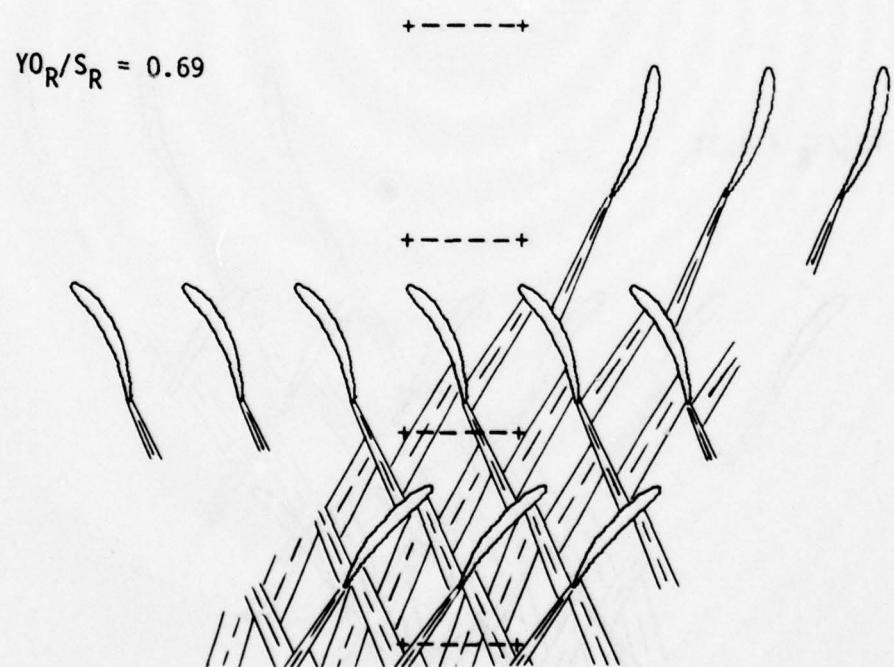
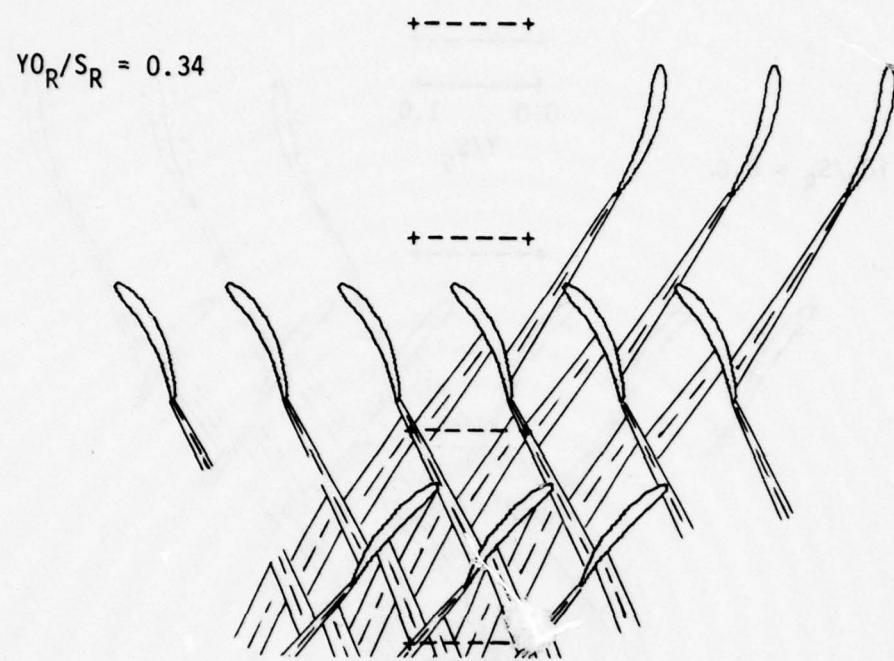
(h) First stator exit flow at rotor sampling position  $Y_0/R/S_R = 0.69$  and five different passage heights.

Figure 13. (Continued).



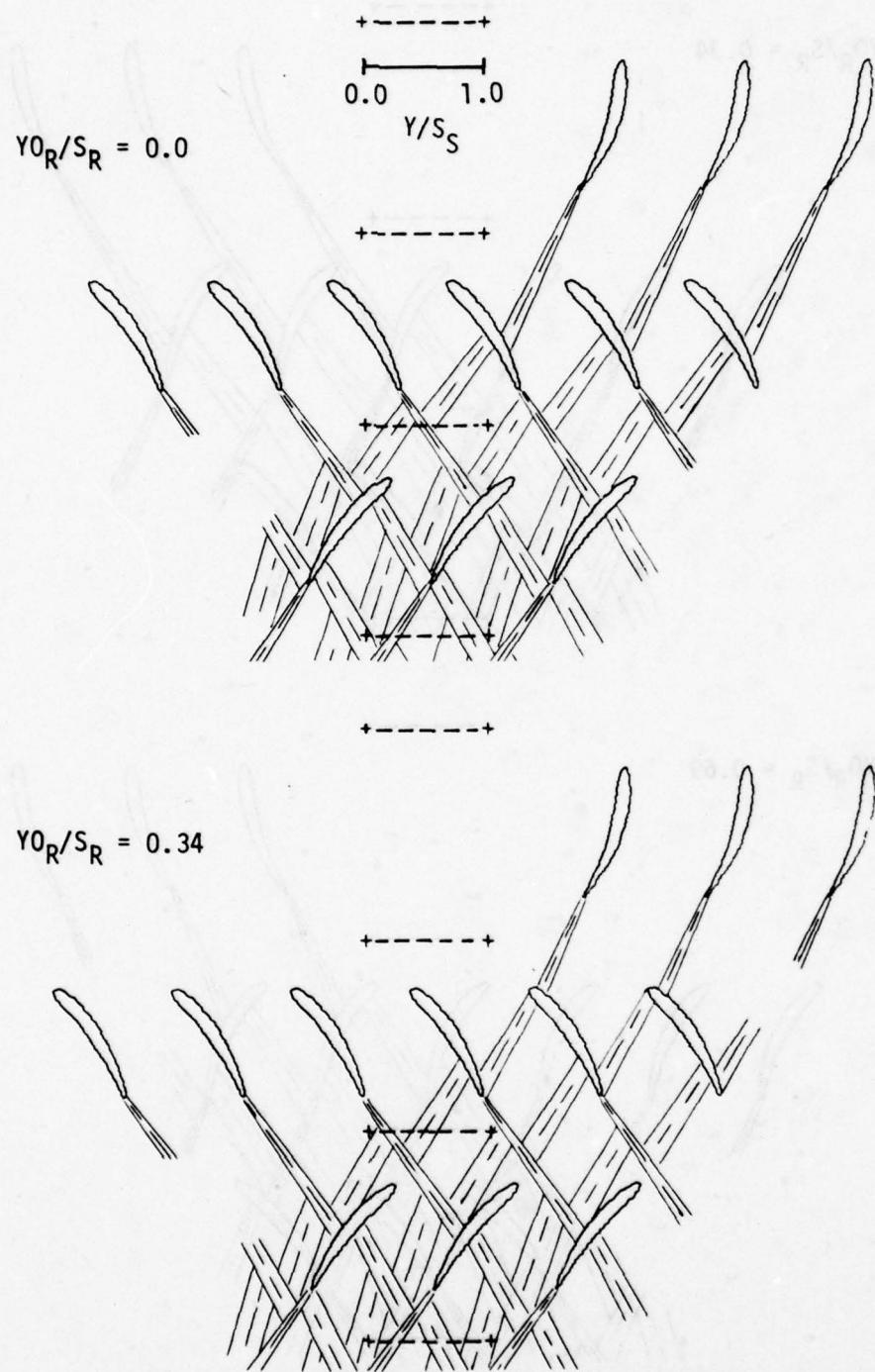
(a) Cascade plots for 10% passage height from hub.

Figure 14. Periodic-average cascade wake interaction plots for the first stage of the research compressor for minimum sound.



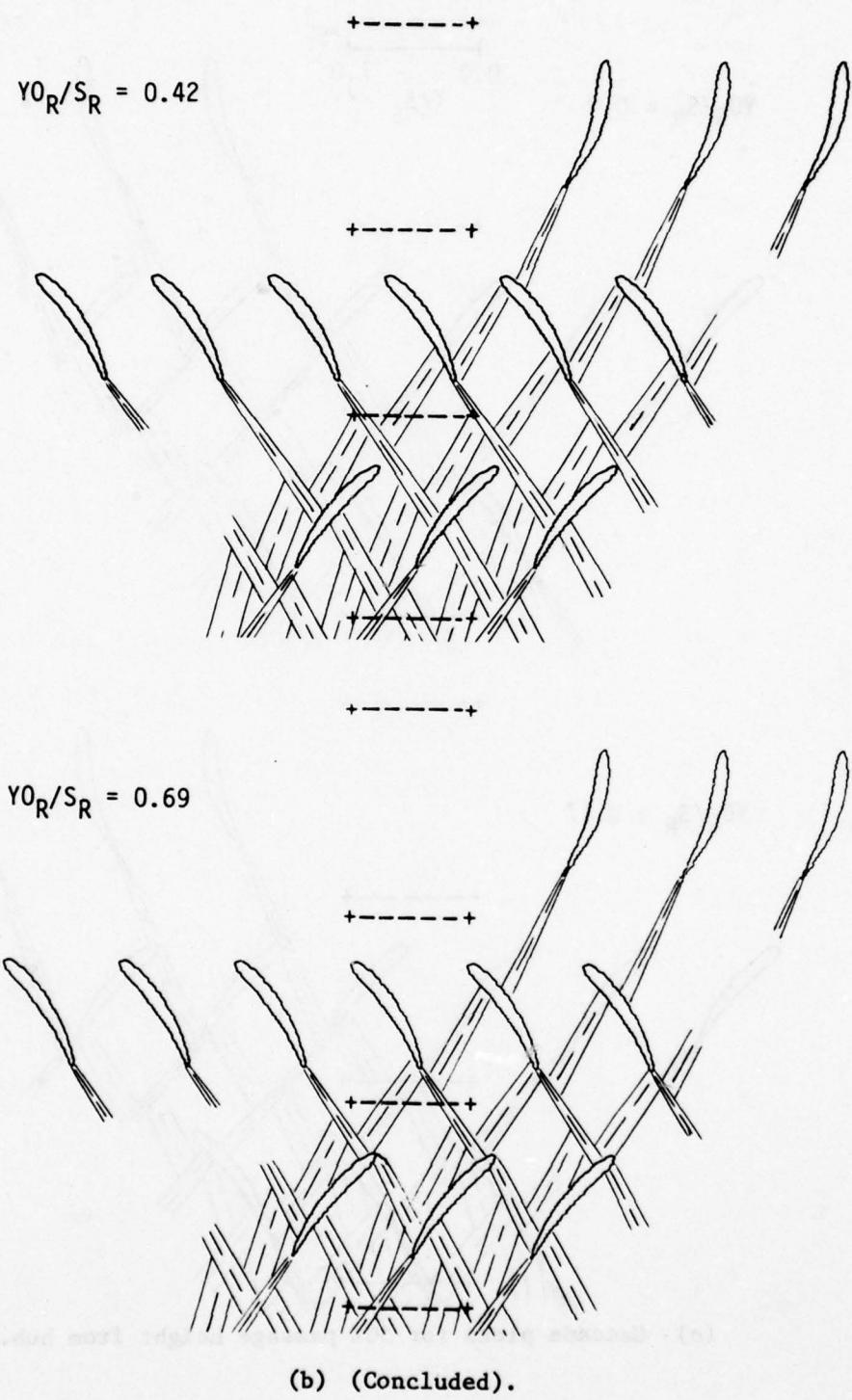
(a) (Concluded).

Figure 14. (Continued).



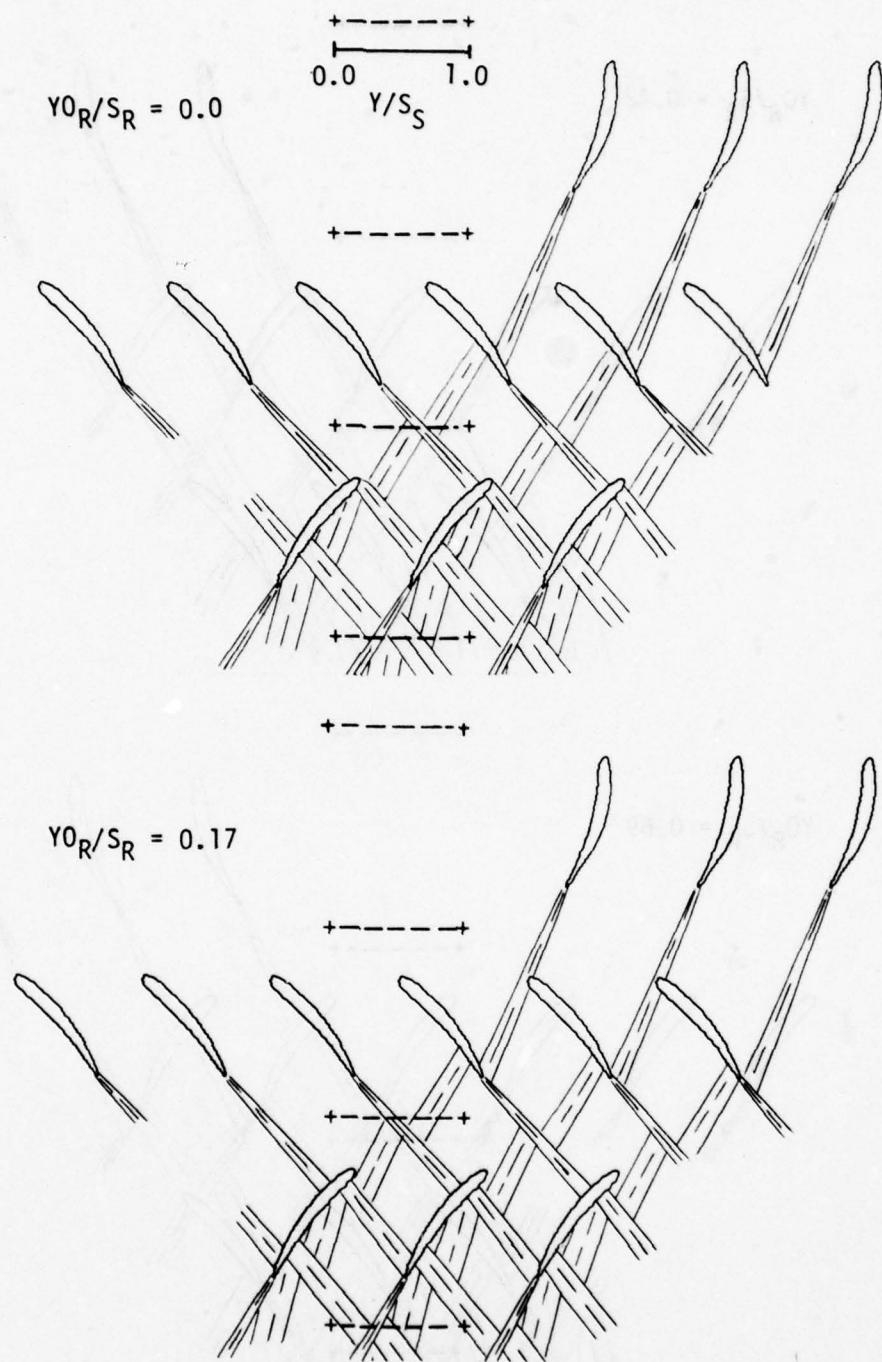
(b) Cascade plots for 30% passage height from hub.

Figure 14. (Continued).



(b) (Concluded).

Figure 14. (Continued).



(c) Cascade plots for 50% passage height from hub.

Figure 14. (Continued).

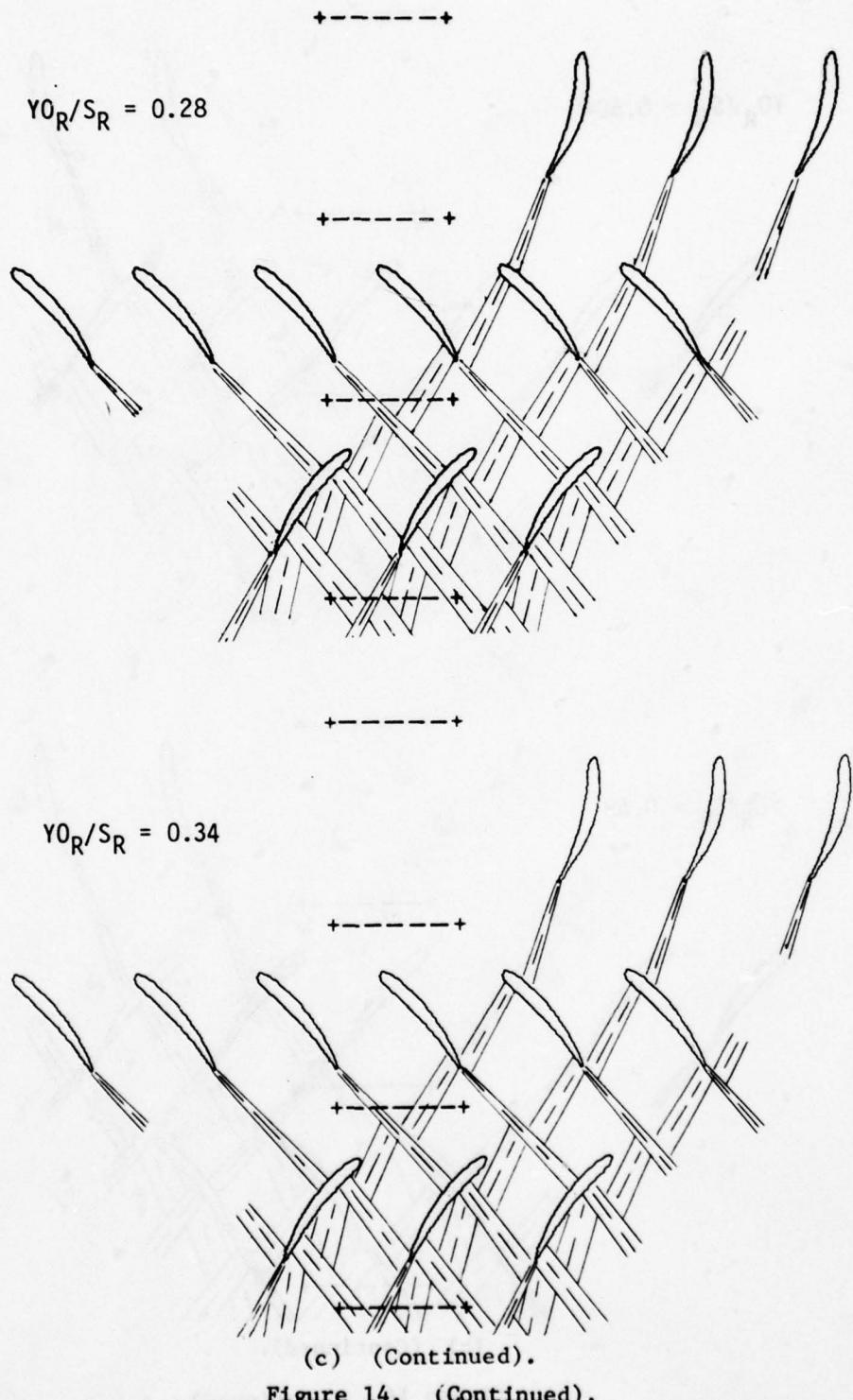
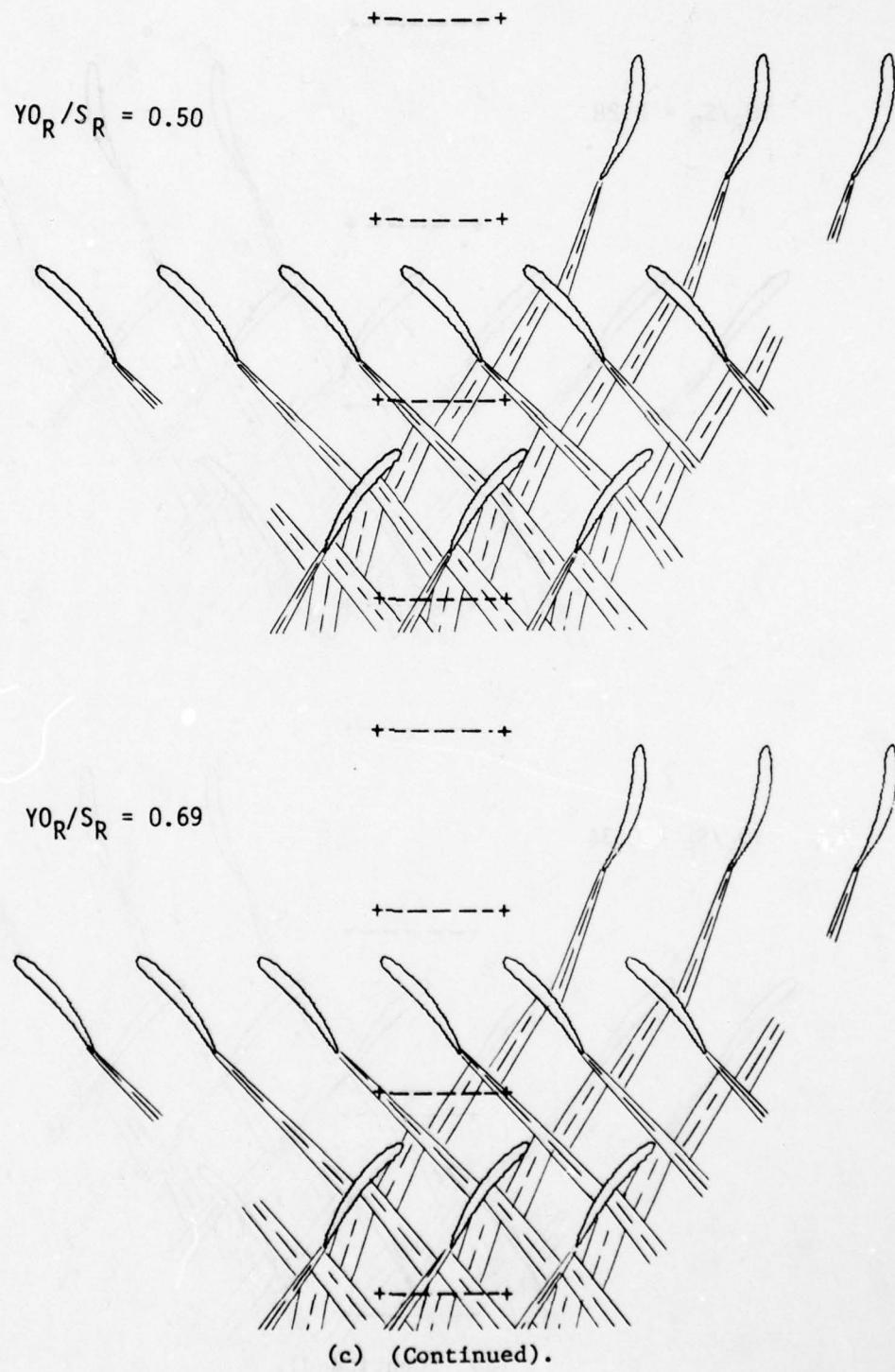


Figure 14. (Continued).



(c) (Continued).

Figure 14. (Continued).

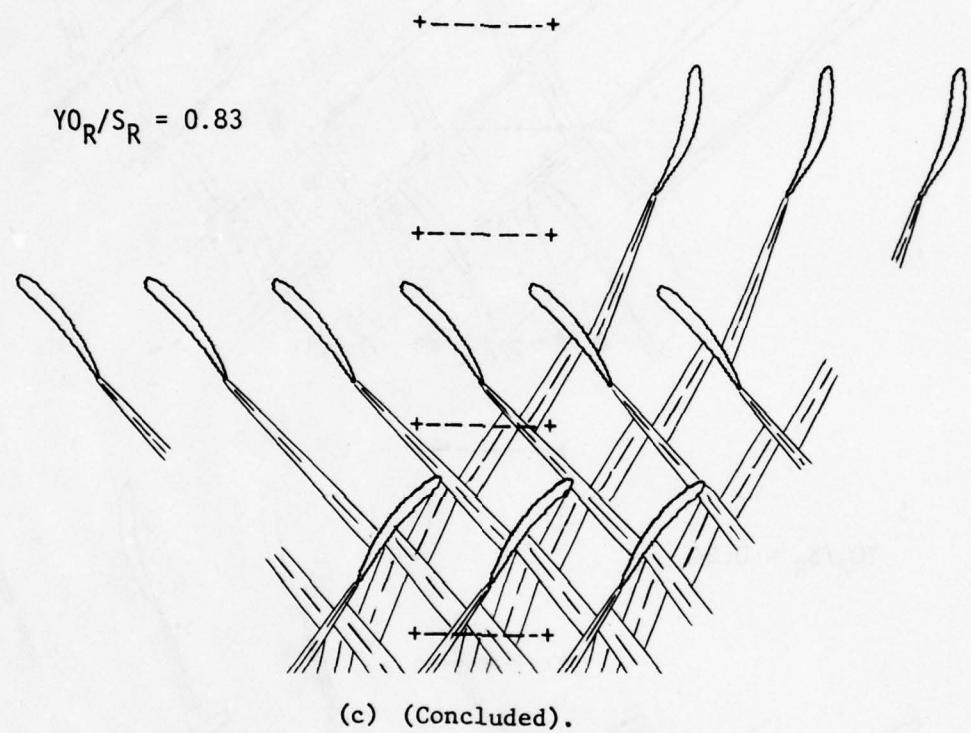
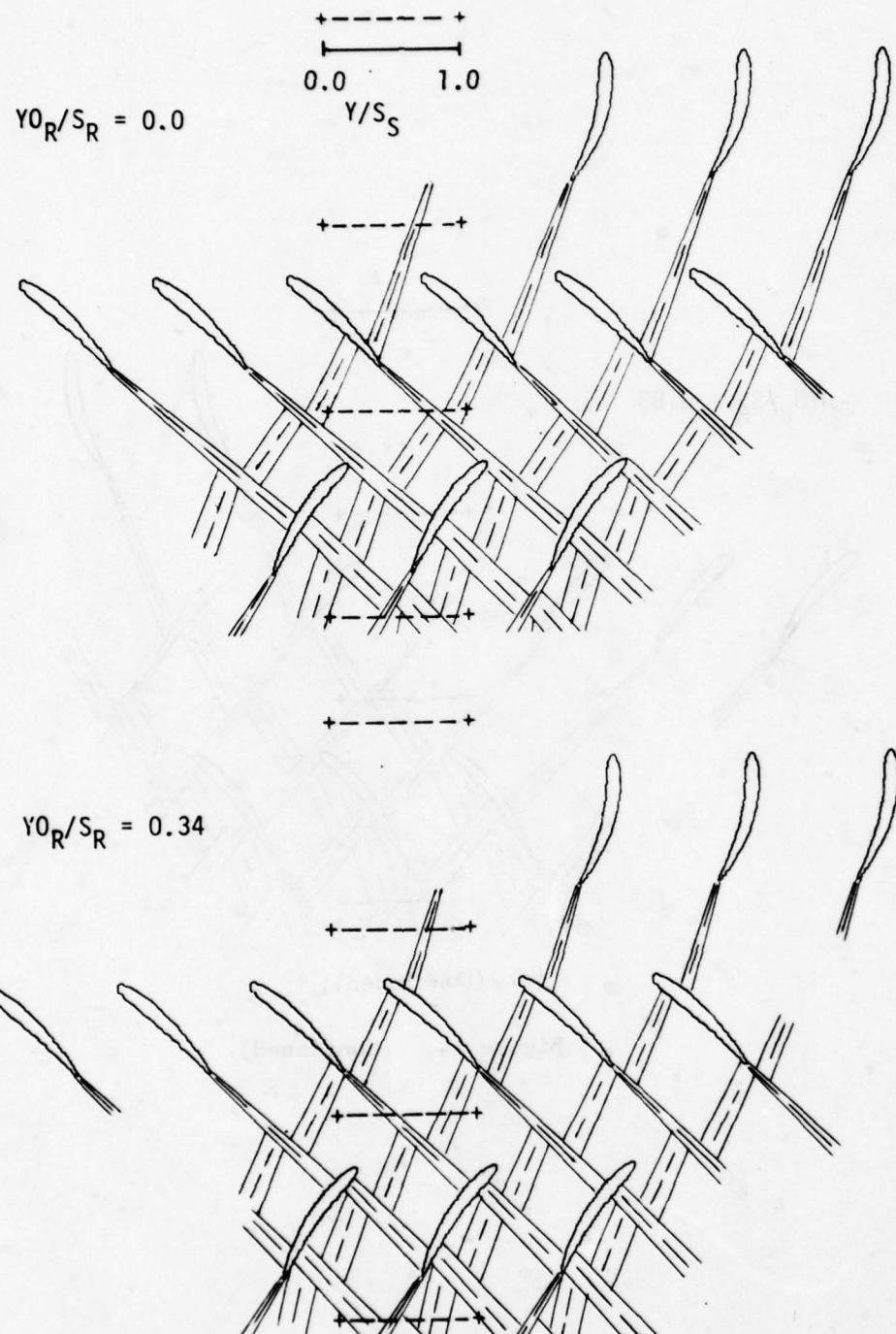


Figure 14. (Continued).



(d) Cascade plots for 70% passage height from hub.

Figure 14. (Continued).

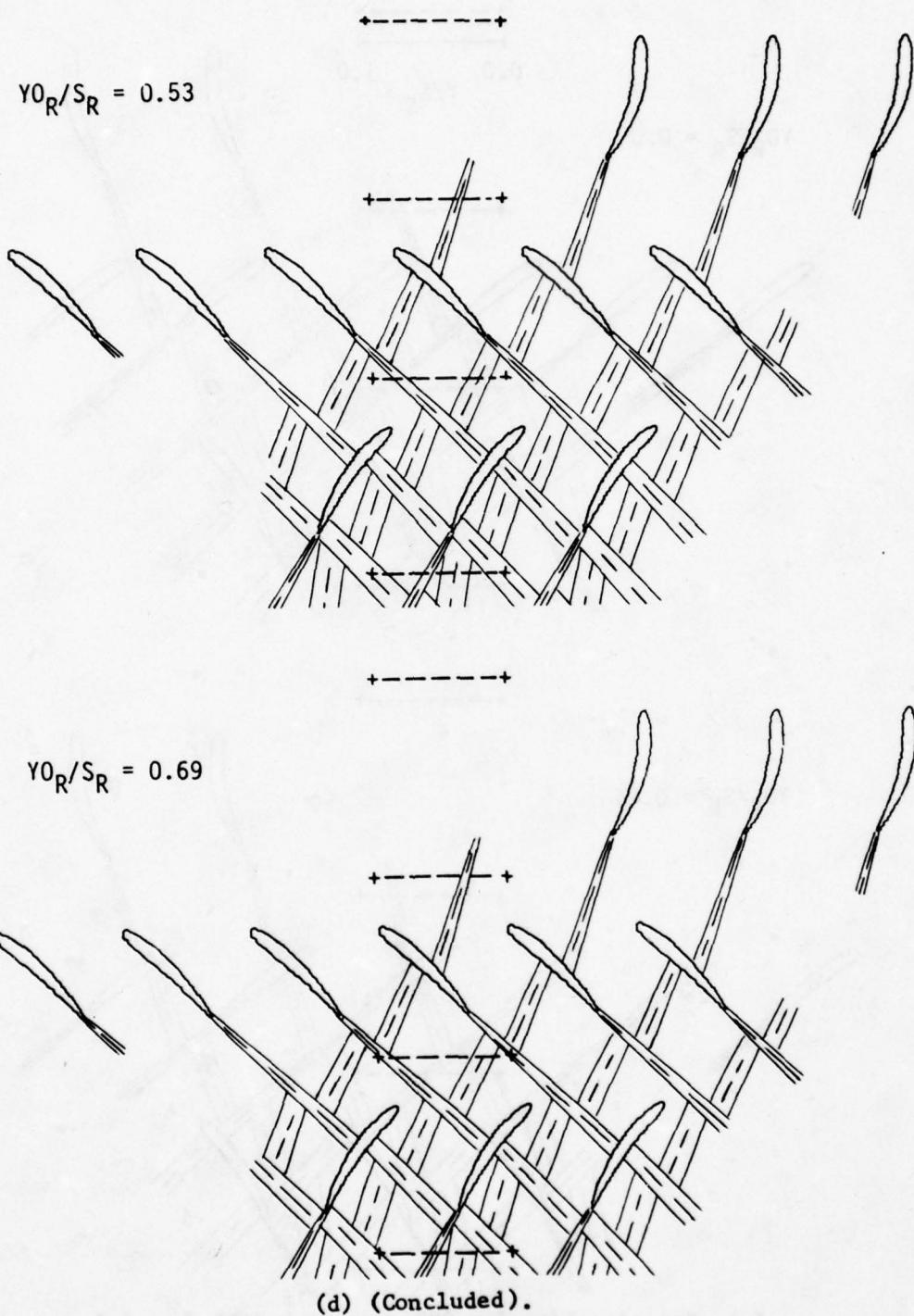
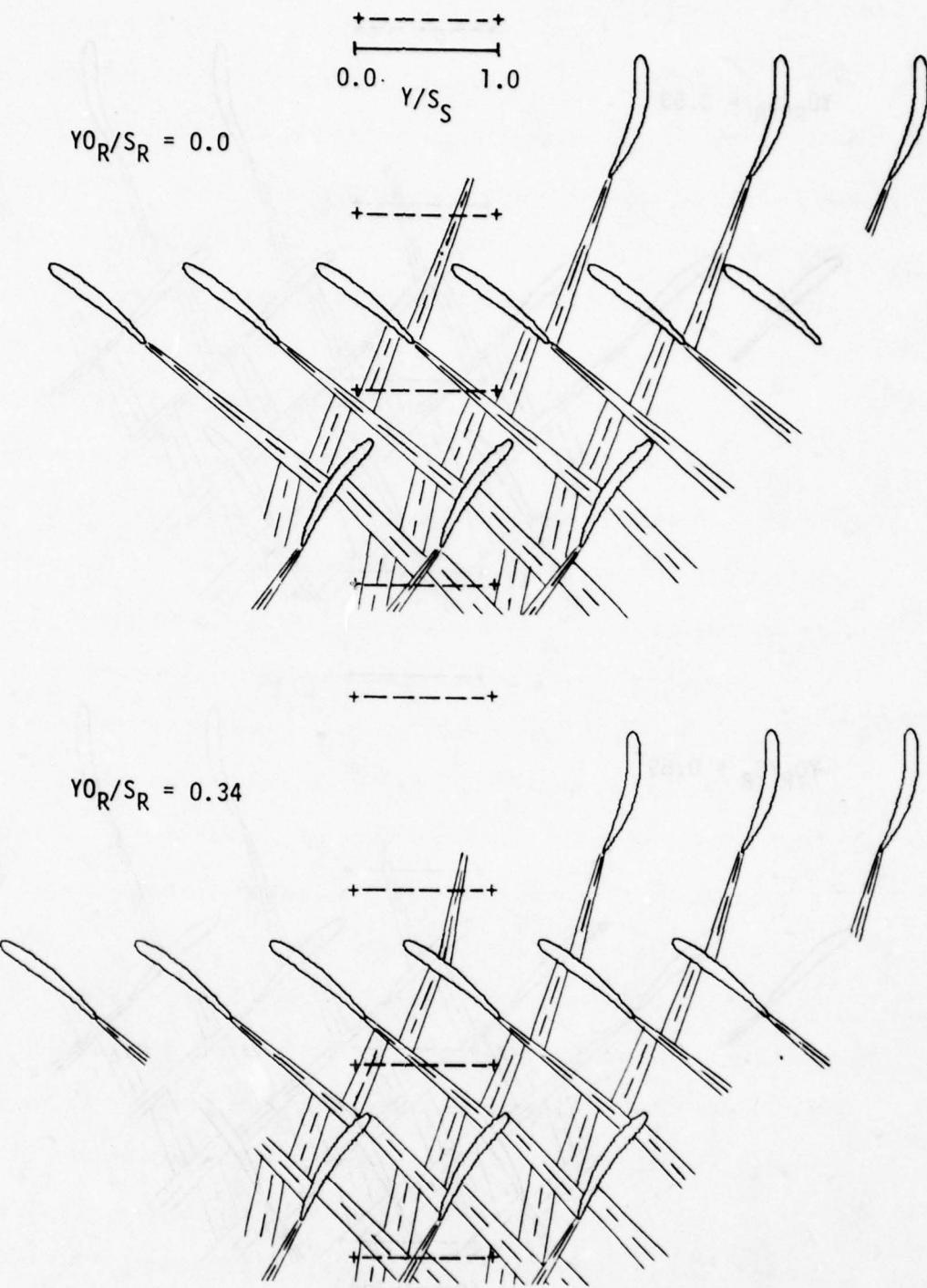
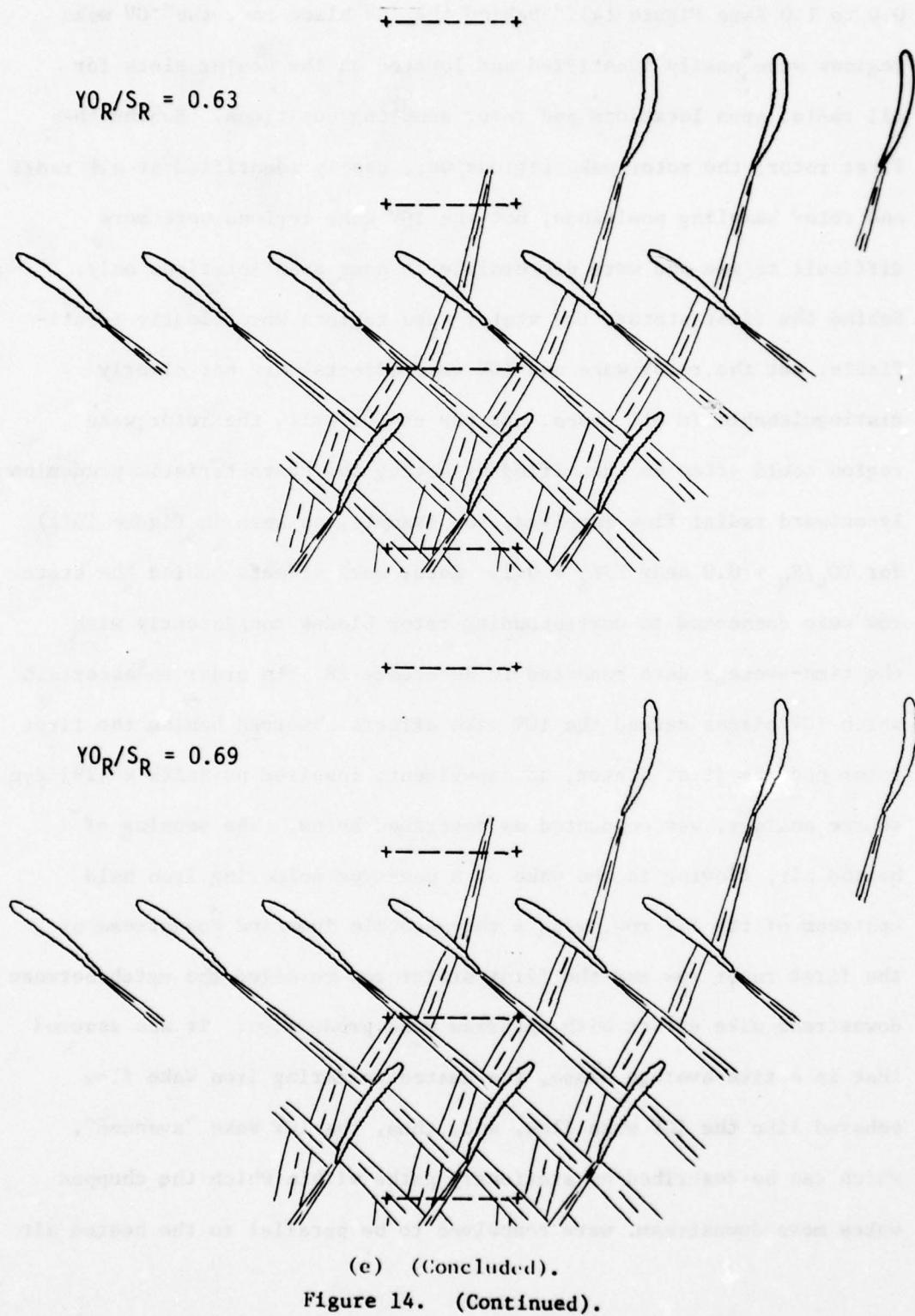


Figure 14. (Continued).



(e) Cascade plots for 90% passage height from hub.

Figure 14. (Continued).



(e) (Concluded).

Figure 14. (Continued).

0.0 to 1.0 (see Figure 14). Behind the IGV blade row, the IGV wake regions were easily identified and located on the scalar plots for all radial span locations and rotor sampling positions. Behind the first rotor, the rotor wake regions were easily identified at all radii and rotor sampling positions, but the IGV wake regions were more difficult to see and were discernible at some span locations only. Behind the first stator, the stator wake regions were clearly identifiable, but the rotor wake and IGV wake effects were not clearly distinguishable in all cases. At the stator exit, the rotor wake region could often be identified by noting the characteristic predominantly-outward radial flow involved, for example, as seen in Figure 13(a) for  $Y_0/R = 0.0$  near  $Y/S = 0.2$ . Rotor wake effects behind the stator row were connected to corresponding rotor blades consistently with the time-average data reported in Reference 28. In order to ascertain which IGV blades caused the IGV wake effects observed behind the first rotor and the first stator, an experiment, inspired by Smith's [29] dye source analogy, was conducted as described below. The sensing of heated air, flowing in the wake of a pen-type soldering iron held upstream of the IGV row, with a thermocouple immersed downstream of the first rotor row and the first stator row revealed the match between downstream wake effect with upstream wake production. It was assumed that in a time-average sense, the heated soldering iron wake flow behaved like the IGV wake flow, and, thus, the IGV wake "avenues", which can be described as stationary paths within which the chopped wakes move downstream, were conceived to be parallel to the heated air

flow paths (see Figure 15). The procedure described above resulted in cascade plots in skeletal form only. Completion of the cascade plots involved careful interpolation between cascade wake plots for the same radii and different rotor sampling positions and cascade plots for the same rotor sampling position but different radii. By using a wake template involving an increasing wake width, the wake thickening effect, suggested in References 2 and 3, was approximated. The chopped segments were caused to reflect Smith's [29] chopping effects by rotating each bounded wake segment (bounded either by blade surfaces or wake regions) around its center. The usefulness of these cascade plots should become evident as the test results are discussed in the next section.

#### 4.2. Discussion of Data

The results of the periodic-average measurements made behind the IGV row are presented in velocity and angle component form in Figure 11 and will be discussed first. The two plots of axial velocity variation from blade-to-blade at 50% passage height from the hub (PHH), for two different rotor sampling positions, indicate that the rotor potential flow field could influence the flow at the IGV row exit measurement plane. Walker and Oliver [3] suggest that 3% to 4% potential flow effect can be expected for this particular axial spacing between blade rows. The present data are in close agreement with this observation. Rotor leading edge locations, as seen in Figure 16 for rotor sampling positions,  $Y_{0R}/S_R = 0.0$  and 0.34, relate to the axial

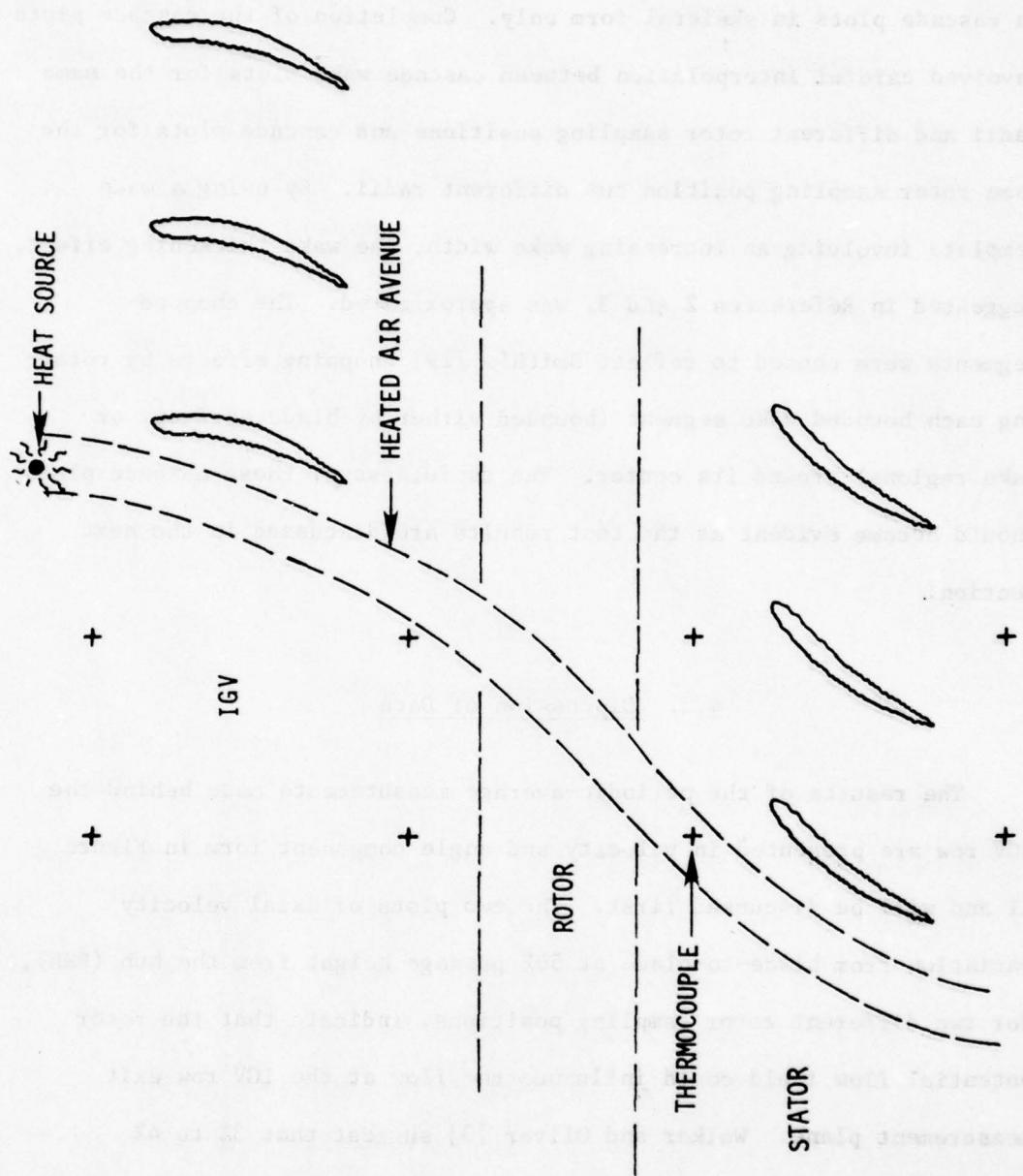


Figure 15. Time-average heated air avenue location in cascade.

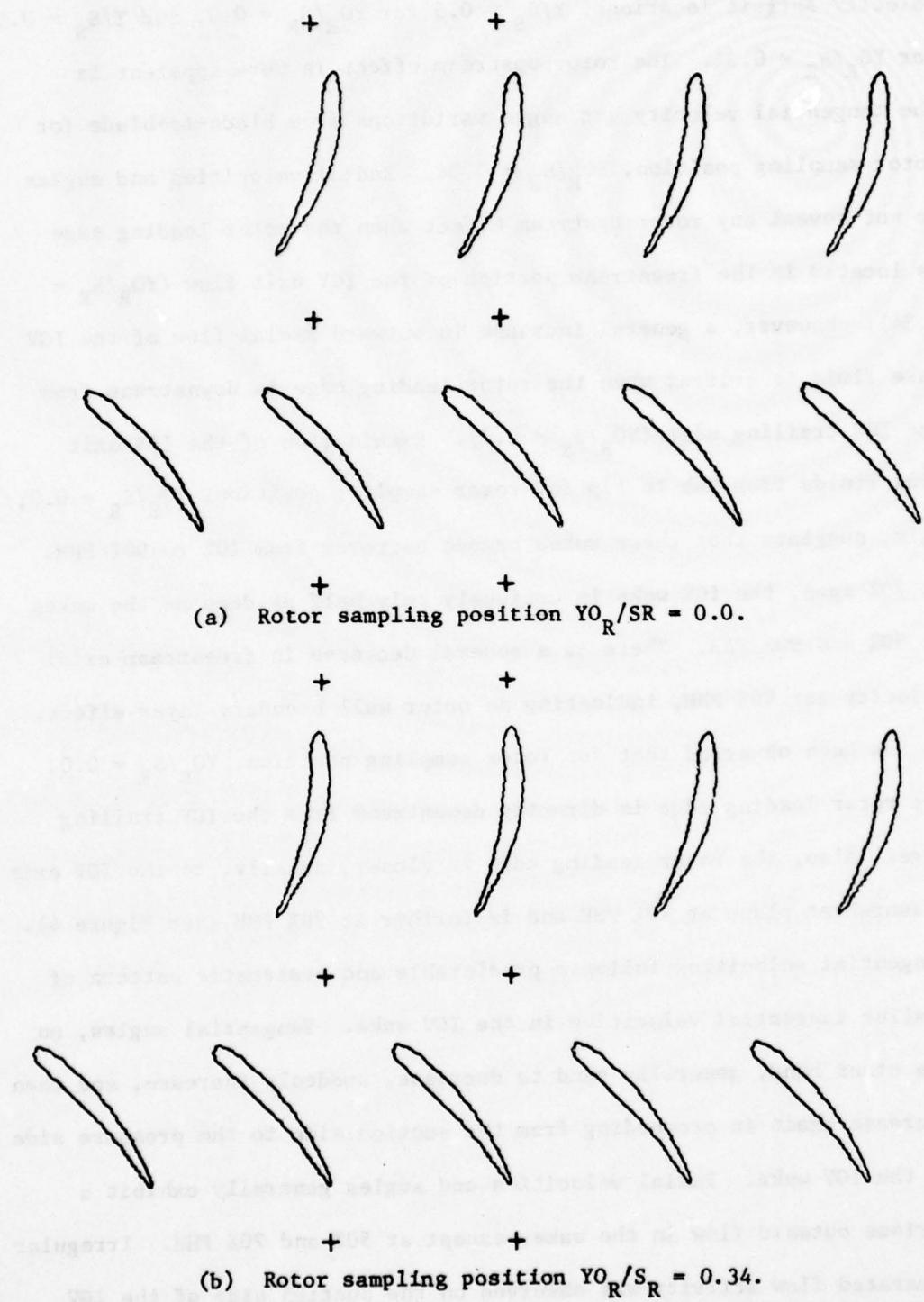


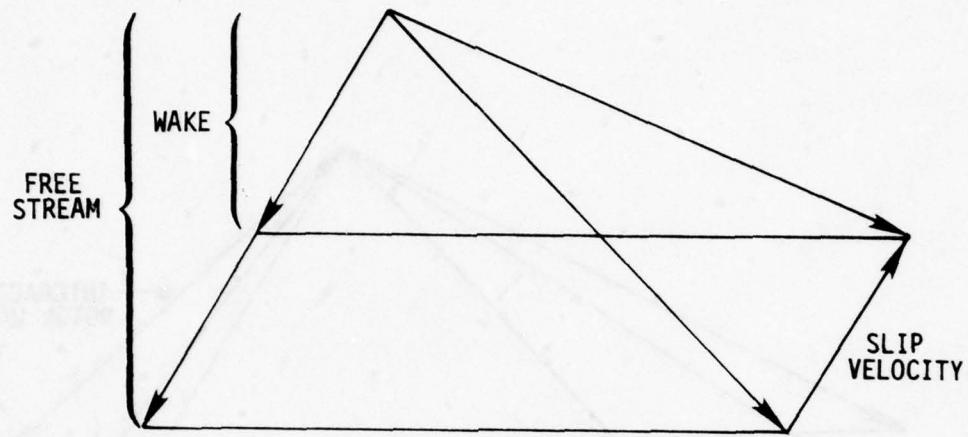
Figure 16. Relative locations of IGV and rotor blades for two rotor sampling positions at 50% span.

velocity deficit locations,  $Y/S_S = 0.5$  for  $Y_0/S_R = 0.0$ , and  $Y/S_S = 0.3$  for  $Y_0/S_R = 0.34$ . The rotor upstream effect is more apparent in the tangential velocity and angle variations from blade-to-blade for rotor sampling position,  $Y_0/S_R = 0.34$ . Radial velocities and angles do not reveal any rotor upstream effect when the rotor leading edge is located in the freestream portion of the IGV exit flow ( $Y_0/S_R = 0.34$ ). However, a general increase in outward radial flow of the IGV wake fluid is evident when the rotor leading edge is downstream from the IGV trailing edge ( $Y_0/S_R = 0.0$ ). Examination of the IGV exit flow fields from hub to tip for rotor sampling position,  $Y_0/S_R = 0.0$ , only, suggests that these wakes become narrower from 10% to 90% PHH. At 70% span, the IGV wake is curiously only half as deep as the wakes at 50% and 90% PHH. There is a general decrease in freestream axial velocity for 90% PHH, indicating an outer wall boundary layer effect. It has been observed that for rotor sampling position,  $Y_0/S_R = 0.0$ , the rotor leading edge is directly downstream from the IGV trailing edge. Also, the rotor leading edge is closer, axially, to the IGV exit measurement plane at 10% PHH and is farther at 90% PHH (see Figure 4). Tangential velocities follow a predictable and systematic pattern of smaller tangential velocities in the IGV wake. Tangential angles, on the other hand, generally tend to decrease, suddenly increase, and then decrease again in proceeding from the suction side to the pressure side of the IGV wake. Radial velocities and angles generally exhibit a curious outward flow in the wake, except at 50% and 70% PHH. Irregular separated flow activity was observed on the suction side of the IGV wake during data acquisition. There did not appear to be any difference

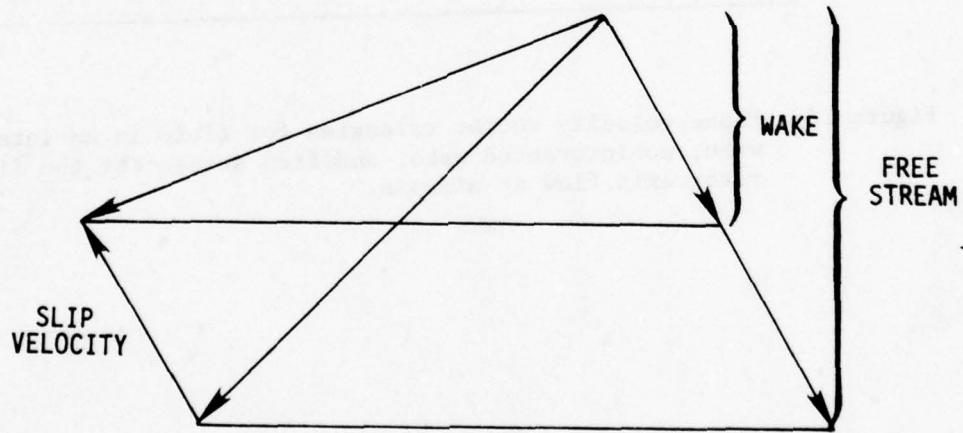
in separated flow irregularity from hub to tip. To obtain meaningful periodic-average data, approximately 2400 instead of the usual 1200, effective samples were obtained for each orientation of the wire.

First rotor periodic-average exit flow parameters for 50% span are presented in Figure 12(a) for six rotor sampling positions and will be discussed next. Corresponding cascade plots are displayed in Figure 14(c). Note that the rotor wake regions shown in the sequence of Figure 12(a) are not from the same rotor blade but rather are from two adjacent rotor blades. This fact may be verified by following the time-sequential movement of the rotor blades in the cascade plots (Figure 14[c]). The chopped IGV wakes seen in the cascade plots follow a fixed avenue. Within this avenue, the wake segments move downstream, as indicated in Figure 14(c). Periodically, the rotor wake region can occupy the same portion of the measuring plane between  $Y/S_S = 0.0$  and 1.0 as the IGV wake avenue. The deepest rotor wake region measured was for a rotor sampling position of  $Y_0/S_R = 0.69$ . For this rotor position, the cascade plot (Figure 14[c]) indicates that the rotor wake/IGV wake interaction occurs slightly upstream from the measurement plane. For this rotor position, the circumferential variation of flow parameters in the rotor wake region is characterized by larger deviation and absolute flow angles and by smaller absolute tangential velocities and axial velocities than observed in the rotor wake regions associated with other rotor sampling positions where the rotor wake/IGV wake interaction occurs closer to, at, or downstream of the measurement plane. When the rotor wake/IGV wake interaction occurs slightly upstream or at the measurement plane, the wake region

will hereafter be called an "interacted wake region." When the rotor wake/IGV wake interaction occurs downstream of the measurement plane, the wake region will hereafter be referred to as a "noninteracting wake region." The dependence of rotor wake region behavior on rotor sampling position is explainable in terms of the physical reasoning proposed by Kerrebrock and Mikolajczak [2] which involved the relative flow of chopped stator wake fluid moving toward the rotor pressure surface due to slip (see Figure 17[a]). Such flow in a chopped IGV wake would tend to result in locally reduced axial velocities and increased deviation angles on the rotor suction surface side and in an accumulation of more slower-moving fluid on the rotor pressure surface side. The mechanism for these interaction effects is best explained in this way: The collected IGV wake fluid "builds up" a surplus of fluid on the pressure side of the rotor blade, while on the suction side of the rotor blade, fluid is drawn towards the rotor pressure surface of the adjacent blade. Thus, a narrower wake region, larger deviation angles, and smaller axial velocities occur at the measurement plane when the suction side wake interaction effects predominate there, as for  $Y_0/S_R = 0.69$ . For  $Y_0/S_R = 0.69$ , the smaller absolute tangential velocities and larger absolute tangential angles are explained with the help of a velocity vector diagram (Figure 18), where three velocity triangles are drawn, one each for free stream, noninteracting wake, and interacting wake regions. For rotor sampling position  $Y_0/S_R = 0.83$ , the rotor blade pressure surface side interaction effect is stronger at the measurement plane, resulting in a wake zone spreading effect. For  $Y_0/S_R = 0.0$ , both



(a) Slip velocity of IGV wake fluid through a rotor row.



(b) Slip velocity of rotor wake fluid through a stator row.

Figure 17. Velocity triangles showing slip velocities.

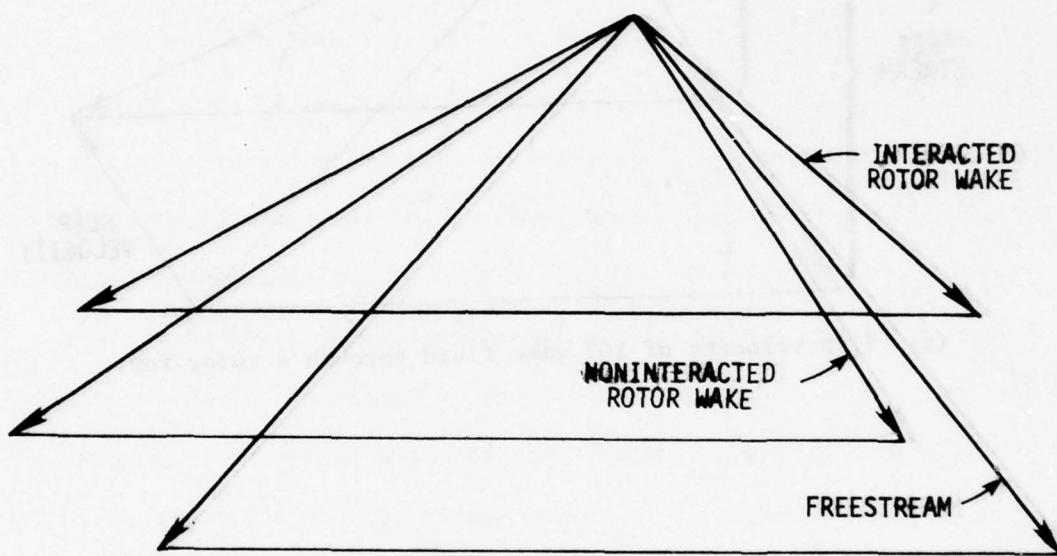


Figure 18. Plane velocity vector triangles for fluid in an interacted wake, noninteracted wake, and free stream for the first rotor exit flow at midspan.

pressure and suction surface interaction effects are noticeable at the measurement plane, the result being a broad shallow wake region. Radial velocity and angle distribution change trends for different rotor sampling positions are not apparent from the data.

Prior to measuring the details of the flow behind the first rotor at other radial locations in the compressor annulus, approximate periodic-average hot-wire oscilloscope traces of a combination of axial and radial velocities were studied to determine which rotor sampling positions should be selected to show the largest changes in periodic-average flow due to rotor wake/IGV wake interaction. For radial locations 10% and 90% PHH, the appropriate rotor sampling positions to use were not obvious because the approximate data were not definitive enough. At the other radial positions, more reasonable rotor sampling position selections could be made with the approximate data.

The rotor exit data for 10% PHH for four rotor sampling positions are shown in Figure 12(b). The related cascade plots are displayed in Figure 14(a). Unfortunately, the data involved only rotor wake/IGV wake interaction at the measurement plane for rotor sampling positions,  $Y_0/S_R = 0.0, 0.32, 0.34$ , and downstream of the measurement plane for  $Y_0/S_R = 0.69$ . Data for interaction occurring slightly upstream of the measurement plane, which might have indicated a comparatively deeper rotor wake region, were not obtained. The information shown is consistent with data already discussed.

The periodic-average data for 30% PHH are shown in Figure 12(c) for rotor sampling positions  $Y_0/S_R = 0.0$  and  $0.42$ , which represent

interacted and noninteracted wake regions respectively. The cascade plots (Figure 14[b]) indicate that, for rotor sampling position  $Y_0/S_R = 0.0$ , a characteristically deeper interacted wake region is produced with the rotor wake/IGV wake interaction occurring upstream of the measurement plane. The flow parameters exhibit the same characteristics as did the rotor exit flow for 50% span,  $Y_0/S_R = 0.69$ , namely, reduced axial velocities and absolute tangential velocities and increased deviation angles and absolute tangential angles in the interacted wake region.

First rotor exit flow component plots for 70% PHH and two rotor sampling positions,  $Y_0/S_R = 0.0$  and 0.53, are shown in Figure 12(d). The corresponding cascade plots (Figure 14[d]) reflect an interacted rotor wake (interaction occurring upstream of the measurement plane) for  $Y_0/S_R = 0.53$  and a noninteracted wake for rotor sampling position  $Y_0/S_R = 0.0$ . For  $Y_0/S_R = 0.53$ , the rotor wake/IGV wake interaction effects are typical of a deeper interacted rotor wake region except for the tangential angles which are slightly smaller in the interacted wake region than in the noninteracted region. This inconsistency is probably due to the fact that the interacted wake region for  $Y_0/S_R = 0.53$  is not the deepest possible and, thus, does not totally possess all of the characteristics exhibited by such wake regions.

The data for 90% PHH rotor exit flow at two rotor sampling positions,  $Y_0/S_R = 0.0$  and 0.63, are shown in Figures 12(e) and 14(e). The rotor wake/IGV wake interaction occurs at the measurement plane for  $Y_0/S_R = 0.63$  and slightly downstream for  $Y_0/S_R = 0.0$ . The rotor

wake region corresponding to interaction upstream of the measurement plane was not observed. The trends indicated by the present data are consistent with similar measurements made elsewhere in the compressor annulus.

Rotor exit flow data for seven radial positions but only one rotor sampling position,  $Y_0/R = 0.0$ , are presented in Figure 12(f). Axial velocity variations indicate a general decrease in wake depth from hub to tip and a circumferential shifting consistent with change of blade twist (see Figure 19) except at 95%. The curious circumferential shift at 95% PHH is not clearly explainable, but it possibly is due to tip scraping and clearance leakage effects. Radial velocities and angles show tip scraping and clearance leakage effects at 90% and 95% PHH. The outer casing boundary layer extends to 95% PHH. A hub boundary layer is not apparent in the velocity data for 5% PHH although radial velocities and angles do indicate the presence of secondary flows. At 10% and 30% PHH, deep interacted wake regions are observed while shallower noninteracted wake regions are seen at 70% and 90% PHH. IGV wake effects seem to be more apparent toward the tip, possibly due to the smaller amount of rotation of the chopped IGV wakes involved there.

First stator periodic-average exit flow data for 50% PHH and six rotor sampling positions are presented in Figure 13(a) with associated cascade plots in Figure 14(c). It is obvious that the first stator exit flow is much more complicated than the first rotor exit flow because of the stator, rotor, and IGV wake interactions involved. The stator exit flow variation with rotor sampling position will be discussed

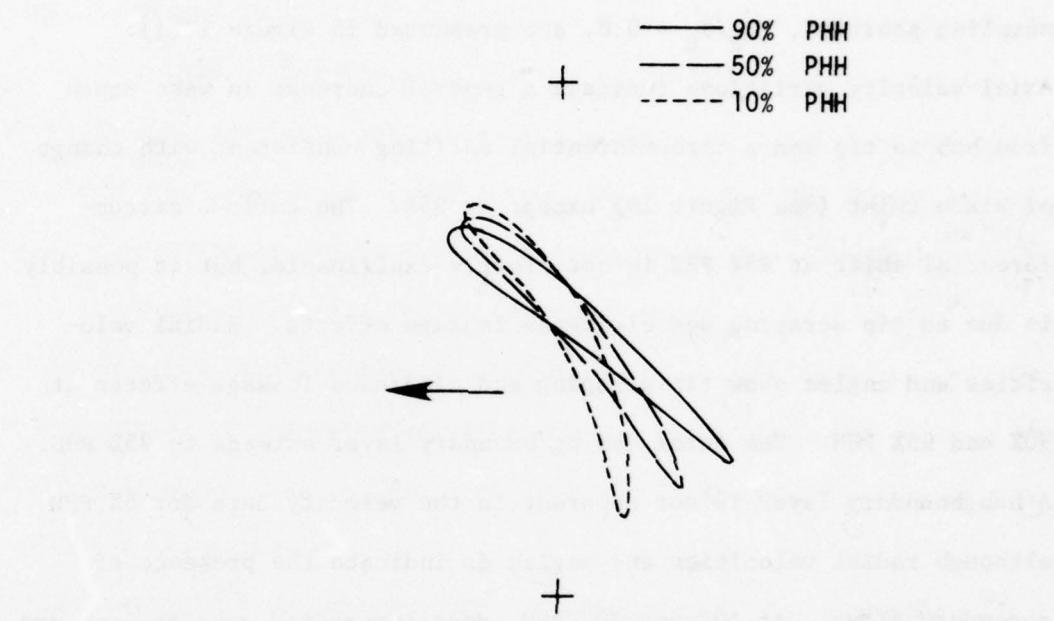


Figure 19. Compressor rotor blade sections at hub, midspan, and tip locations.

first. As mentioned by Kerrebrock and Mikolajczak [2], the chopped rotor wake fluid tends to move from the stator suction surface to the pressure surface because of the slip motion in the wake (see Figure 17[b]). Such flow in a chopped rotor wake would tend to result in locally-reduced flow and increased deviation angles on the suction side of the stator blade with an accumulation of more slower moving fluid on the pressure side. The deeper stator wake regions are those corresponding to rotor sampling positions  $Y_0/R = 0.0, 0.69$ , and  $0.83$ , with the deepest wake region being associated with  $Y_0/R = 0.83$ . For rotor sampling position,  $Y_0/R = 0.83$ , the larger tangential velocities of the chopped rotor wake fluid appear to cancel the smaller tangential velocities of the stator wake fluid with the result being only a small amount of circumferential variation of tangential velocities. Also, the outward radial flow of the chopped rotor wake fluid cancels the inward flow of the stator wake fluid. The flow angle tendencies of both the chopped rotor wake and the stator wake seem to add together. For rotor position  $Y_0/R = 0.5$ , the stator wake is least affected by other wake fluid as it is most like an isolated stator blade wake. Whenever the chopped rotor wake is clearly discernible at the measurement plane,  $Y_0/R = 0.0, 0.69$ , and  $0.83$ , it is interacting with IGV and stator wake fluid, and thus, its distribution tends to be characteristically spread out and shallow. For rotor position  $Y_0/R = 0.34$ , little of the chopped rotor wake fluid is identifiable at the measurement plane.

First stator exit flow data for 10% PHH are shown in Figure 13(b) with related cascade plots in Figure 14(a). As the rotor blade passes

in stop-action sequence from  $Y_0/R_S$  0.0 to 0.69, the effect of chopped rotor and IGV wakes on the stator exit flow may be clearly seen. For rotor sampling position  $Y_0/R_S = 0.0$ , a nearly isolated (noninteracted) stator wake region and a separate and distinct rotor wake/IGV wake interaction region are present at the measurement plane. For  $Y_0/R_S = 0.34$ , the stator wake region is slightly deeper with a wake sequence of stator-rotor-IGV for  $Y/S_S = 0.0$  to 1.0. The rotor wake/stator wake interaction occurs just upstream of the measurement plane, and thus, the stator wake region is of the interacted variety. For  $Y_0/R_S = 0.69$ , the wake sequence is stator-IGV-rotor, with an appreciably different flow pattern from the ones for  $Y_0/R_S = 0.0$  and 0.34. For  $Y_0/R_S = 0.69$ , the large axial velocity deficit at  $Y/S_S = 0.6$  is the result of an IGV wake region. The rotor wake/stator wake interaction occurs downstream of the measurement plane. The rotor wakes are easily identified in the tangential angle plots.

Stator exit flow data for 30% PHH are shown in Figure 13(c). The deepest stator wake region occurs for rotor sampling position  $Y_0/R_S = 0.69$ . The corresponding cascade plot indicates strong stator wake/rotor wake interaction at the measurement plane. The least influenced stator wake region occurs for rotor position  $Y_0/R_S = 0.34$ . In this case, the velocity and angle data reflect expected trends for a noninteracted stator wake region. For all of the rotor sampling positions, the tangential angles are the most sensitive indicator of rotor wake location at the measurement plane.

Stator exit flow data for 70% PHH are presented in Figure 13(d). The rotor wake broadly influences the stator exit flow and makes crisp

delineation of effects difficult. The deepest stator wake region is measured for a rotor sampling position of  $Y_0/R_S = 0.0$ , consistent with the cascade plot (see Figure 14[d]). For this rotor sampling position, the stator wake/rotor wake interaction is occurring just upstream of the measurement plane. Radial velocities show classical outward rotor wake flow and inward stator and IGV wake flow.

Stator exit flow data for 90% (Figure 13[e]) indicate broad influence of the rotor on the stator exit flow. Thus, as at 70% span, wake interaction effects are relatively difficult to sort out. Deeper stator wake regions are associated with stator wake/rotor wake interaction just upstream or at the measurement plane. Tangential angles are again good indicators of chopped rotor wake influence location.

By looking at the stator exit flow from hub to tip for three rotor sampling positions, namely,  $Y_0/R_S = 0.0$ , 0.34, and 0.69 (see Figure 13[f-h]), several observations can be made. Boundary layer growth has extended to 90% PHH behind the stator, compared to only 95% PHH behind the rotor. An upstream effect of the second rotor on the flow at the first stator exit measurement plane is not noticeable, although small (less than 4%) upstream effects are probably present.

## 5. CONCLUSIONS

Summarized below are the conclusions reached after studying the periodic-average flow data acquired to date. Further data analysis and acquisition are being continued.

The IGV suction surface flow is separated at all radial locations. The separated flow appears to involve unusual unsteady effects that deserve further observation. The IGV exit flow is also complicated by noticeable first rotor upstream effects at the measurement plane.

The chopped IGV wakes move downstream within fixed avenues. Imbedded rotor and stator periodic unsteadiness was found to be appreciable in portions of the compressor annulus, depending on the extent of wake interaction involved at the particular location considered. The concept of chopped wake fluid flowing in the general direction of the slip velocity appears to be verified by the present data. Chopped wake interaction with the suction side fluid of a blade section typically resulted in locally large deviation angles and small axial velocities. As a result of wake interaction, tangential and radial velocities were observed to cancel, while tangential angles added. Behind the rotor, large differences in flow fields could result from rotor wake/IGV wake interactions. Behind the stator, stator wake/rotor wake interactions did not lead to as much variation in flow fields. Stator wake/IGV wake interaction effects were difficult to discern, even though IGV effects could be seen at most spanwise locations. Secondary flow effects were noticeable in some cases near the hub, while leakage and scraping effects were discernible near the tip. Outer

casing boundary layer growth was evident from the first rotor exit flow to the first stator exit flow.

In general, the periodic-average flow was orderly and explainable with consistent wake transport and interaction plots and physical reasoning.

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## 7. APPENDIX A: CALCULATOR PROGRAMS AND STORAGE

Calculator data acquisition and reduction programs used in the present study are listed in this section. All programs and data are stored on cassette tape and are labeled and indexed as specified below:

Flow coefficient program: Calculation of overall flow coefficient from flow rate venturi meter data. Cassette 4C, file 16.

Actuator position correlation program: Linear least squares correlation between actuator potentiometer voltage readout and actuator motion for probe and circumferential positioning actuators. Cassette 4C, file 3.

Hot-wire effective cooling velocity/actual velocity ratio calibration program: Calibration of hot-wire with respect to sensor yaw angle, pitch angle, and velocity for the determination of the ten coefficients in Eq. (6); consists of two parts: 1) calibration data acquisition, and 2) least squares calibration data correlation. Cassette 8B, files 8, 9, 11-13.

Hot-wire linearizer velocity calibration program: Velocity calibration to determine the four polynomial coefficients required by the anemometer linearizer through a least squares correlation of calibration data. Cassette 11B, files 2-4.

Hot-wire second order velocity calibration program: Velocity calibration to determine the three coefficients in the second order velocity calibration Eq. (5) through a least squares correlation of calibration data. Cassette 11B, file 5.

Periodic-average hot-wire data acquisition program: Acquisition of hot-wire, periodic-average, three-dimensional, circumferential survey data. Cassette 11B, files 5-7.

Periodic-average hot-wire reduction program: Reduction of periodic-average hot-wire data to obtain three-dimensional point flow-field parameters. Cassette 11B, files 8-18.

Periodic-average hot-wire data: Storage of periodic-average hot-wire data obtained with the single inclined hot-wire sensor in the research compressor. Cassette 11B, files 57, 58; data cassette #1, files 1-40.

## 8. APPENDIX B: PARAMETER EQUATIONS

The equations used in the periodic-average measurement system for the calibration procedures and the acquisition and reduction of data are presented below. Symbols and notations are presented on page xii, while sign conventions are generally shown in Figure 10.

8.1. General Parameters8.1.1. Basic Fluid Properties

Barometric pressure, N/m<sup>2</sup>:

$$P_{atm} = h_{hg@t_{baro}} [1.0 - 0.0018 (t_{baro} - 273.15)] \gamma_{hg@273^oK} \quad (B-1)$$

Density of air, kg/m<sup>3</sup>:

$$\rho = \frac{P_{atm}}{R t} \quad (B-2)$$

Specific weight of water, N/m<sup>3</sup>:

$$\begin{aligned} \gamma_{H_2O} = \frac{g}{g_c} [ & 996.86224 + 0.1768124 \left( \frac{9}{5} t - 459.67 \right) \\ & - 2.64966 \times 10^{-3} \left( \frac{9}{5} t - 459.67 \right)^2 \\ & + 5.00063 \times 10^{-6} \left( \frac{9}{5} t - 459.67 \right)^3 ] \end{aligned} \quad (B-3)$$

8.1.2. Blade-Element Quantity

Percent passage height from hub:

$$PHH = \left( \frac{r-0.14224}{0.06096} \right) \times 100 \quad (B-4)$$

8.1.3. Miscellaneous

Venturi volume flow rate, m<sup>3</sup>/s:

$$Q_v = 0.05229 \sqrt{\frac{2g_c \gamma_{H_2O} \Delta P_{vent}}{\rho}} \quad (B-5)$$

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ANALYSIS OF MULTISTAGE, AXIAL FLOW TURBOMACHINE WAKE PRODUCTION--ETC(U)

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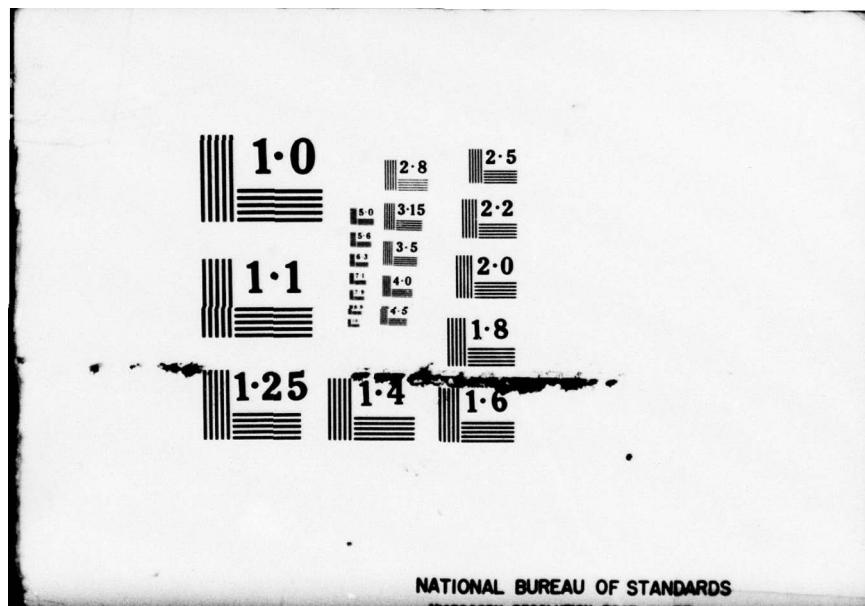
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Blade velocity, m/s:

$$U = \frac{r\pi RPM}{30.0} \quad (B-6)$$

Calibration nozzle jet velocity, m/s:

$$V = \sqrt{\frac{2g_c \gamma H_2 \Delta P_n}{\rho}} \quad (B-7)$$

Venturi flow coefficient:

$$\phi_v = \frac{Q_v}{A U_t} \quad (B-8)$$

## 8.2. Three-Dimensional Periodic-Average Hot-Wire Parameters

Effective cooling velocity, m/s:

$$V_e = K_1 + K_2 E_\ell + K_3 E_\ell^2 \quad (B-9)$$

Sensor yaw angle relationship (see Figure 8):

$$\cos \alpha = \cos \theta_0 \cos \theta_p \cos \theta_y + \sin \theta_0 \sin \theta_p \quad (B-10)$$

Effective cooling velocity/actual velocity ratio:

$$\begin{aligned} V_e/V = & b_0 + b_1 \alpha + b_2 \theta_p + b_3 V + b_4 \alpha^2 + b_5 \theta_p^2 \\ & + b_6 V^2 + b_7 \alpha \theta_p + b_8 \alpha V + b_9 \theta_p V \end{aligned} \quad (B-11)$$

Absolute tangential flow angle (see Figures 8 and 9), degrees:

$$\beta_\theta = \beta_{mv} + \theta_{a,off} + \theta_{y,c} \quad (B-12)$$

Radial flow angle (see Figure 8), degrees:

$$\beta_r = -\theta_p \quad (B-13)$$

Radial component of fluid velocity, m/s:

$$V_r = V \sin \beta_r \quad (B-14)$$

Axial component of fluid velocity, m/s:

$$V_z = V \cos \beta_r \cos \beta_\theta \quad (B-15)$$

Tangential component of absolute fluid velocity, m/s:

$$V_\theta = V \cos \beta_r \sin \beta_\theta \quad (B-16)$$

Tangential component of relative fluid velocity, m/s:

$$V'_\theta = U - V_\theta \quad (B-17)$$

Relative fluid velocity, m/s:

$$V' = \sqrt{(V'_\theta)^2 + (V_z)^2} \quad (B-18)$$

Relative tangential flow angle, degrees:

$$\beta'_\theta = \sin^{-1} (V'_\theta / V_z) \quad (B-19)$$

## 9. APPENDIX C: TABULATION OF PERIODIC-AVERAGE HOT-WIRE DATA

The periodic-average circumferential survey data are tabulated in this section. The data represent flow field parameters downstream from the IGV row (station 2), first rotor row (station 3), and the first stator row (station 4). Each vector is completely defined by velocity magnitude, tangential flow angle, and radial flow angle. The computer headings are defined as follows:

$Y/SS$  = circumferential spacing,  $Y/S_S$

$V$  = absolute velocity,  $V$ , m/s

BETA Y = absolute tangential flow angle,  $\beta\theta$ , degrees

BETA R = radial flow angle,  $\beta_r$ , degrees

PHH = percent passage height from hub, PHH

$YOR/SR$  = circumferential ratio blade sampling position,  $Y_0/S_R$

Table C-1. Hot wire circumferential survey data obtained with the periodic-average measurement method at minimum noise condition.

STATION 2									
Y/SS	V	BETA Y	BETA R	Y/SS	V	BETA Y	BETA R	Y/SS	V
M/S	M/S	DEG	DEG	M/S	M/S	DEG	DEG	M/S	M/S
PHH=10.00 YOR/SR=0.00	PHH=30.00 YOR/SR=0.00	PHH=50.00 YOR/SR=0.00							
0.000	15.588	25.046	1.608	0.001	15.631	21.789	1.502	0.000	15.548
0.053	15.642	24.622	1.817	0.052	15.658	21.389	1.472	0.051	15.724
0.103	15.832	24.840	1.018	0.103	15.676	21.118	1.334	0.102	15.736
0.154	15.824	24.588	1.207	0.154	15.777	21.017	1.094	0.154	15.831
0.206	15.679	24.362	1.562	0.205	15.732	21.077	0.746	0.205	15.827
0.257	15.652	24.138	1.527	0.257	15.550	21.250	0.575	0.257	15.557
0.307	15.612	24.060	1.291	0.308	15.294	20.070	1.371	0.309	15.492
0.360	15.511	23.914	1.257	0.361	15.250	21.040	0.690	0.360	15.451
0.411	15.511	23.894	1.455	0.411	14.977	20.391	1.044	0.411	15.199
0.433	14.628	23.536	3.505	0.462	14.681	20.100	1.639	0.462	14.869
0.462	14.371	25.380	2.771	0.488	14.452	19.161	2.169	0.515	14.946
0.488	12.733	24.623	7.771	0.514	14.262	18.817	2.368	0.540	14.844
0.514	12.747	22.622	7.889	0.539	13.019	17.925	2.776	0.565	14.477
0.540	10.353	26.984	9.451	0.565	11.615	17.899	1.011	0.592	14.139
0.565	9.467	26.537	1.0906	0.591	9.832	18.976	1.633	0.616	13.595
0.591	9.089	27.050	8.967	0.617	8.715	23.739	1.237	0.642	11.423
0.617	8.565	25.325	1.0456	0.642	8.642	26.923	8.020	0.668	9.546
0.642	9.424	24.077	5.327	0.667	8.812	22.28	3.886	0.694	7.906
0.668	10.233	23.170	3.614	0.693	9.675	24.972	6.600	0.720	8.096
0.694	11.806	23.099	0.904	0.720	11.908	23.656	4.867	0.745	10.145
0.720	12.055	21.838	4.701	0.745	13.194	26.580	6.540	0.771	10.778
0.745	13.427	25.679	1.664	0.771	14.159	25.758	4.914	0.796	12.458
0.771	14.142	26.659	0.438	0.796	14.874	24.222	3.970	0.822	14.058
0.796	14.788	26.597	2.002	0.822	14.874	23.043	0.849	0.849	14.733
0.822	15.057	25.916	2.033	0.848	15.191	23.029	3.970	0.874	15.148
0.848	15.389	25.599	1.963	0.874	15.315	22.911	3.064	0.901	15.251
0.874	15.571	25.407	1.670	0.899	15.601	22.902	2.396	0.925	15.461
0.925	15.714	25.769	2.048	0.925	15.793	22.381	1.722	0.951	15.615
0.956	15.753	25.172	1.922	0.977	15.899	21.754	0.976	0.976	15.384
1.000	16.003	25.640	1.085	1.000	15.906	21.774	1.000	1.000	15.092

Table C-1. (Continued).

STATION 2									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S
PHH=50•00 YOR/SR=0.34					PHH=70•00 YOR/SR=0.00				
PHH=90•00 YOR/SR=0.00					PHH=90•00 YOR/SR=0.00				
0•000	15•608	21•822	1•228	0•000	15•225	19•945	-0•106	0•000	14•010
0•052	15•537	22•187	0•915	0•052	15•316	18•869	0•192	0•025	14•238
0•103	15•253	20•374	1•260	0•103	15•432	18•403	0•535	0•051	14•237
0•154	15•197	20•562	1•135	0•154	15•622	18•164	0•461	0•078	17•705
0•205	15•081	20•220	0•578	0•206	15•614	18•247	0•227	0•103	17•599
0•257	14•859	18•600	1•058	0•257	15•596	18•445	0•385	14•581	-0•360
0•308	14•809	18•073	0•625	0•309	15•460	18•069	0•107	14•446	0•854
0•360	14•727	17•694	1•121	0•360	15•122	17•815	0•868	0•205	14•415
0•411	14•887	16•018	0•385	0•412	15•019	18•243	0•470	0•308	14•578
0•463	15•014	15•858	0•880	0•463	14•751	17•956	0•616	0•360	18•147
0•514	15•089	15•009	-0•046	0•489	14•757	18•197	0•127	0•411	18•840
0•540	15•024	16•022	0•622	0•514	14•728	18•788	0•087	0•462	17•486
0•565	14•925	14•629	0•825	0•539	14•536	17•928	-0•125	0•513	13•684
0•592	14•772	15•432	1•208	0•565	14•440	17•768	0•169	0•566	16•282
0•618	13•578	18•565	2•650	0•593	14•366	17•849	-0•003	0•617	13•640
0•642	11•979	17•964	4•708	0•617	14•664	17•870	0•588	0•642	13•568
0•668	10•503	12•1•295	2•248	0•642	13•047	18•871	1•778	0•669	13•536
0•694	8•737	19•209	2•567	0•668	11•955	20•545	1•487	0•694	12•465
0•720	9•108	18•645	-2•740	0•694	10•820	22•441	0•016	0•720	11•071
0•745	10•712	20•935	-3•835	0•720	10•593	21•907	-2•598	0•746	12•410
0•771	11•751	19•110	-1•478	0•745	11•087	21•792	-3•743	0•771	12•394
0•796	13•565	21•419	0•858	0•771	11•786	19•267	-0•742	0•796	13•523
0•822	15•030	22•924	1•820	0•796	12•909	20•255	1•395	0•822	14•138
0•848	15•206	22•071	3•709	0•822	13•955	21•111	0•482	0•848	12•233
0•874	15•672	22•838	2•506	0•848	14•566	21•518	0•342	0•874	-2•635
0•899	15•844	22•657	1•345	0•874	14•895	21•794	-0•099	0•899	1•648
0•926	15•757	23•529	1•853	0•900	15•197	21•028	-0•241	0•925	13•504
0•951	15•580	22•656	1•544	0•925	15•232	20•391	0•136	0•950	18•934
0•976	15•726	22•647	1•713	0•977	15•499	19•468	0•316	0•977	17•608
1•000	15•789	22•725	1•365	1•000	15•616	19•214	0•294	1•001	14•461

Table C-1. (Continued).

STATION 3																												
Y/SS	V M/S	BETA Y CEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG																	
PHH= 5.00 YOR/SR=0.00										PHH=10.00 YOR/SR=0.32																		
-0.000 21.935 49.546 1.073 0.000 22.434 48.357 -1.516 0.000 22.829 73.899 3.912										5.779 77.302 77.494 0.025 22.494 77.351 9.136																		
0.053 21.330 49.312 1.320 0.052 21.401 47.655 0.285 0.051 21.721 77.351 9.136	0.102 21.171 49.623 0.305 0.103 21.838 48.022 -1.579 0.051 21.455 72.248 9.891	0.129 21.256 49.222 -0.675 0.155 21.547 48.032 -0.664 0.077 21.03 21.133 6.835 10.387	0.154 21.172 45.061 -1.811 0.180 21.523 47.736 -0.990 0.103 21.128 20.987 6.84 9.491	0.181 21.102 49.598 -3.026 0.206 21.678 48.329 -1.649 0.128 21.190 6.104 3.912	0.206 21.235 54.031 0.231 0.231 21.528 49.825 -0.971 0.154 21.287 5.691 10.576	0.231 21.062 64.984 -5.06 0.258 20.587 52.875 0.678 0.180 21.05 21.397 5.405 7.69	0.257 21.164 75.861 -2.333 0.283 21.161 64.197 -0.101 0.205 21.844 5.244 4.42 2.477	0.282 21.235 81.32 0.309 0.309 21.037 73.123 2.867 0.231 21.884 5.244 4.42 2.477	0.309 21.533 79.924 9.342 0.334 20.775 74.613 6.373 0.257 21.877 4.9.970 1.736	0.335 21.971 77.878 11.058 0.360 21.221 73.560 6.359 0.283 21.727 4.8.449 1.295	0.362 21.898 75.557 11.435 0.385 21.279 70.402 8.026 0.308 21.946 4.8.486 1.792	0.385 21.782 72.410 10.354 0.411 21.369 68.136 9.089 0.359 21.794 4.7.484 0.323	0.412 22.023 62.851 8.786 0.437 21.906 63.272 8.059 0.412 21.884 4.8.229 0.386	0.436 21.898 64.152 8.173 0.465 22.538 59.094 6.194 0.464 22.321 4.8.757 0.425	0.462 22.328 61.324 5.873 0.488 22.862 56.834 5.526 0.514 22.086 4.6.710 0.395	0.489 22.466 58.414 5.129 0.514 23.255 54.365 4.036 0.566 22.502 4.7.860 0.607	0.514 22.541 57.074 4.938 0.539 23.431 52.212 2.321 0.617 22.477 4.7.151 1.153	0.540 23.003 55.844 3.251 0.565 23.244 50.976 1.949 0.668 22.474 4.7.539 1.253	0.565 22.941 54.864 3.164 0.592 22.945 49.632 1.859 0.719 22.627 4.7.394 1.999	0.616 23.042 52.393 2.550 0.620 23.381 49.545 0.043 0.771 22.378 4.7.133 1.819	0.670 23.097 51.889 1.864 0.668 22.965 49.049 0.088 0.822 21.809 4.7.137 0.558	0.720 22.921 51.217 1.920 0.720 23.264 48.597 0.995 0.848 21.980 4.8.164 1.168	0.772 22.899 50.870 1.122 0.772 22.913 47.569 -0.525 0.875 22.258 53.252 0.991	0.822 22.701 49.455 1.423 0.824 22.768 48.002 -0.833 0.899 22.395 54.973 0.668	0.874 22.281 48.948 1.913 0.875 22.544 47.754 -0.783 0.926 22.571 65.380 0.964	0.926 22.001 48.340 1.369 0.925 21.967 46.798 0.435 0.950 22.446 73.897 3.265	0.976 21.478 47.201 1.724 0.975 21.759 47.002 -0.116 0.977 22.421 78.877 5.500	1.001 21.455 47.889 1.067 1.000 21.712 47.737 -0.455 1.000 22.077 79.384 8.215

Table C-1. (Continued).

STATION 3									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S
PHH=10.00 YOR/SR=0.34					PHH=10.00 YOR/SR=0.69				
PHH=30.00 YOR/SR=0.00					PHH=30.00 YOR/SR=0.00				
-0.000	22.036	74.150	10.842	-0.000	22.0145	48.527	1.445	0.000	21.739
0.026	21.053	72.403	13.146	0.052	21.0735	47.921	1.710	0.051	21.086
0.051	21.023	68.163	10.787	0.103	21.0351	47.635	1.130	0.155	21.071
0.077	21.041	66.0425	8.370	0.154	21.0205	47.485	1.550	0.205	21.074
0.104	21.064	62.0213	5.935	0.205	21.0257	47.304	1.359	0.257	20.952
0.129	21.092	55.0642	4.635	0.257	21.0308	47.234	1.234	0.305	20.927
0.154	21.073	56.0213	4.365	0.308	21.0283	47.166	1.134	0.357	20.896
0.180	21.066	53.0643	3.368	0.360	21.0223	46.156	1.045	0.404	20.850
0.205	22.013	51.0386	2.034	0.411	21.0342	44.946	-0.212	0.445	20.811
0.232	21.0828	49.0514	1.866	0.463	21.0665	45.649	-1.021	0.360	19.732
0.258	21.0801	47.0772	1.514	0.468	21.0742	46.232	-1.802	0.386	19.500
0.308	21.0563	46.0821	1.348	0.514	21.0693	47.190	-2.008	0.411	19.399
0.360	21.0413	46.0873	1.273	0.539	20.5983	50.091	-2.201	0.436	18.755
0.411	21.074	46.0909	0.909	-0.002	0.565	21.0124	60.580	0.016	0.470
0.463	21.0852	47.0020	0.145	0.591	21.0269	70.977	3.163	0.515	18.023
0.514	22.0151	46.0955	-0.304	0.617	21.0434	75.553	4.845	0.540	18.018
0.565	22.037	46.0422	-0.236	0.642	22.0157	74.596	0.051	0.555	18.025
0.617	22.0270	46.0818	-0.371	0.663	22.0059	72.312	5.063	0.591	18.025
0.668	22.0421	45.0939	-0.831	0.694	22.0416	70.868	6.179	0.615	18.023
0.720	22.0375	46.0345	-1.127	0.720	22.0592	66.723	6.181	0.642	20.631
0.771	22.0185	46.0904	-1.261	0.745	22.0888	64.137	6.444	0.668	20.562
0.796	22.0281	47.0525	-1.0978	0.772	22.0640	59.635	7.873	0.695	21.792
0.822	21.0623	46.0835	-1.0135	0.796	23.0253	57.742	5.391	0.720	21.446
0.848	21.0778	48.0540	-0.316	0.822	23.0408	55.024	4.661	0.745	21.521
0.874	21.0617	56.0082	-0.117	0.849	23.0206	52.644	4.139	0.771	21.608
0.899	21.0631	66.0690	-3.767	0.875	22.0959	50.519	2.894	0.822	21.456
0.925	21.0933	73.0448	7.277	0.899	22.0925	4.9.850	2.395	0.874	21.727
0.952	21.0728	76.0429	10.049	0.925	22.0505	4.8.895	1.977	0.926	21.748
0.977	22.0052	75.0049	10.818	0.975	22.0416	4.8.312	0.986	0.976	21.611
1.000	21.0917	72.0818	10.483	1.000	22.0493	4.8.240	0.487	0.999	21.601

Table C-1. (Continued).

STATION 3									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	V/SS M/S	BETA Y DEG	BETA R DEG	V/SS M/S	BETA Y DEG	BETA R DEG
<b>PHH=30.00 YOR/SR=0.42</b>								<b>PHH=50.00 YOR/SR=0.00</b>	
								<b>PHH=50.00 YOR/SR=0.17</b>	
0.000	22.806	54.909	0.301	0.001	20.545	41.928	-0.362	-0.000	20.640
0.025	22.560	55.848	2.185	0.052	20.605	41.579	0.210	0.051	20.775
0.051	22.333	63.857	4.805	0.103	20.658	42.662	1.026	0.103	20.621
0.077	22.410	64.537	5.419	0.154	20.532	42.241	1.096	0.155	20.659
0.103	22.065	62.894	7.269	0.206	20.502	42.096	0.649	0.206	20.717
0.128	22.021	60.623	5.893	0.257	20.418	41.704	1.172	0.257	20.924
0.154	21.538	56.338	5.481	0.308	20.591	42.208	0.457	0.309	20.940
0.180	21.046	56.283	5.421	0.359	20.383	42.321	0.272	0.334	21.383
0.205	21.111	54.912	3.412	0.411	20.339	41.778	0.636	0.359	21.521
0.232	20.228	52.549	3.879	0.462	20.468	42.098	0.009	0.387	21.562
0.257	19.697	50.550	4.193	0.489	20.738	42.760	-0.378	0.411	21.244
0.308	19.936	48.217	2.541	0.515	20.665	43.288	0.324	0.439	21.359
0.359	20.052	46.400	1.818	0.545	21.151	46.413	0.256	0.463	21.083
0.412	20.361	46.159	0.150	0.565	20.937	50.680	2.399	0.489	21.433
0.463	20.301	45.644	0.058	0.592	21.273	54.557	3.285	0.516	20.938
0.514	20.309	45.096	-0.132	0.617	21.533	56.387	4.269	0.539	20.840
0.565	20.134	43.948	0.581	0.643	21.186	54.781	5.259	0.565	20.469
0.618	20.542	44.345	0.160	0.668	20. P33	53.780	5.707	0.592	20.267
0.668	21.013	44.212	-0.892	0.694	20.501	53.400	5.073	0.617	20.366
0.719	21.146	43.485	-0.669	0.723	19.620	51.545	5.204	0.642	19.937
0.771	21.535	43.863	-1.040	0.746	19.702	50.597	2.766	0.669	19.744
0.797	21.579	44.004	-0.323	0.773	19.526	49.196	1.031	0.694	19.756
0.822	21.558	44.702	-0.991	0.797	19.222	47.457	0.668	0.719	19.647
0.848	21.883	45.047	-0.395	0.825	19.446	46.023	0.253	0.745	19.638
0.874	22.241	45.718	-0.898	0.848	19.601	43.997	-0.696	0.771	19.685
0.899	22.131	46.444	0.376	0.874	19.572	43.295	0.209	0.822	19.814
0.925	22.656	49.799	-0.444	0.899	20.076	43.048	-0.887	0.875	20.191
0.950	22.717	54.482	0.392	0.925	20.169	42.929	-0.490	0.925	20.316
0.976	22.809	61.894	1.841	0.977	20.660	42.783	-0.818	0.976	20.586
1.000	22.658	65.177	3.195	1.000	20.803	42.614	-0.176	1.000	20.633

Table C-1. (Continued).

STATION 3									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S
PHH=50.00 YOR/SR=0.28					PHH=50.00 YOR/SR=0.50				
PHH=50.00 YOR/SR=0.69					PHH=50.00 YOR/SR=0.69				
0.000	20.775	42.225	-0.014	-0.000	20.408	42.426	-0.007	0.000	18.903
0.053	20.483	42.192	0.622	0.027	20.244	43.054	0.250	0.025	18.610
0.104	20.607	43.004	1.027	0.053	20.174	44.873	0.554	0.052	18.747
0.156	20.884	43.469	0.307	0.077	20.066	48.476	1.299	0.077	20.264
0.180	20.775	43.644	1.077	0.103	19.850	52.872	4.041	0.105	21.053
0.206	20.953	44.193	1.304	0.129	20.469	57.066	3.848	0.127	21.338
0.231	21.085	45.268	0.814	0.156	20.587	56.147	5.168	0.154	21.269
0.258	21.417	49.642	2.073	0.181	20.823	52.813	5.199	0.180	21.122
0.283	21.243	53.349	4.534	0.205	21.192	50.312	4.264	0.207	21.057
0.309	21.409	56.844	5.357	0.231	21.609	46.565	2.799	0.257	20.898
0.335	21.512	56.574	5.140	0.257	21.546	44.921	2.205	0.311	20.673
0.361	20.952	51.562	5.391	0.283	21.634	44.207	1.303	0.360	20.656
0.385	21.070	50.371	4.304	0.309	21.481	43.863	0.785	0.414	20.419
0.411	21.120	48.571	2.797	0.334	20.998	43.070	1.433	0.463	20.400
0.436	21.013	46.936	1.585	0.360	20.699	43.132	1.753	0.514	20.482
0.463	20.799	45.868	1.674	0.385	20.676	42.127	1.470	0.565	20.125
0.488	20.696	45.105	1.768	0.411	20.645	43.017	0.904	0.617	19.940
0.515	20.396	44.653	1.983	0.437	20.341	42.314	1.675	0.668	19.955
0.539	20.516	44.755	1.007	0.463	20.374	42.587	1.156	0.720	19.869
0.566	20.174	44.198	1.113	0.514	20.234	43.100	0.731	0.745	20.019
0.591	20.212	44.013	-0.517	0.565	19.819	41.603	1.028	0.771	19.991
0.618	20.213	44.753	-0.294	0.617	19.847	42.653	-0.169	0.797	19.577
0.669	19.802	43.889	-0.310	0.668	19.724	42.319	-0.664	0.822	19.664
0.720	19.693	43.305	-0.865	0.719	19.435	41.886	-0.322	0.846	19.151
0.771	19.909	42.483	-1.427	0.771	19.843	42.087	-0.810	0.874	18.615
0.824	20.015	42.698	-1.075	0.822	19.946	42.144	-0.453	0.899	18.445
0.874	20.151	42.777	0.118	0.874	19.954	41.140	0.094	0.925	17.924
0.926	20.416	41.378	-0.097	0.930	19.654	41.140	0.453	0.950	17.627
0.976	20.671	43.078	0.073	0.976	19.423	41.885	0.299	0.977	17.229
1.000	20.508	42.559	0.968	1.000	19.091	43.919	0.661	1.000	17.802

Table C-1. (Continued).

STATION 3									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	V/SS M/S	V M/S	BETA Y DEG	BETA R DEG	V/SS M/S	V M/S
PHH=50.00 YOR/SR=0.83					PHH=70.00 YOR/SR=0.53				
-0.000	20.311	44.803	0.019	-0.000	1.8.682	45.370	1.031	-0.000	1.9.075
0.026	20.446	42.048	1.548	0.051	1.8.797	44.717	1.488	0.026	1.8.851
0.051	21.115	43.497	-0.549	0.103	1.9.133	45.666	0.986	0.052	1.8.750
0.077	21.014	42.356	0.708	0.154	1.9.022	43.541	1.864	0.077	1.8.363
0.103	21.049	42.803	0.246	0.209	1.9.109	42.540	1.195	0.103	1.8.552
0.154	20.813	42.676	1.131	0.257	1.9.236	42.959	0.839	0.128	1.6.144
0.206	20.765	42.361	1.092	0.309	1.9.102	42.134	1.620	0.155	1.7.828
0.257	20.729	41.968	1.089	0.360	1.9.142	41.593	1.726	0.180	1.7.778
0.308	20.543	41.271	1.332	0.411	1.9.207	41.257	1.771	0.207	1.7.073
0.361	20.635	41.801	1.520	0.464	1.9.501	42.508	1.165	0.231	1.7.410
0.411	20.550	41.904	1.124	0.515	1.9.499	42.848	2.310	0.257	1.7.047
0.464	20.655	42.122	0.151	0.541	1.9.554	42.161	1.957	0.282	1.6.634
0.514	20.320	42.206	0.153	0.565	1.9.707	43.438	0.866	0.309	1.7.917
0.565	20.277	42.222	-0.190	0.594	1.9.662	43.846	1.146	0.334	1.7.914
0.617	20.396	41.895	-0.389	0.617	1.9.724	43.588	0.363	0.360	1.9.074
0.642	20.348	40.777	0.279	0.642	2.0.038	46.013	0.553	0.315	1.9.627
0.670	20.620	41.063	-0.228	0.668	2.0.039	49.147	1.914	0.412	1.9.732
0.694	20.626	43.204	-0.523	0.694	2.0.206	52.979	2.646	0.436	1.9.652
0.719	20.841	46.921	0.960	0.720	2.0.285	54.491	3.278	0.463	1.9.597
0.745	21.033	52.991	3.862	0.745	1.9.782	53.295	5.397	0.514	1.9.542
0.771	20.466	55.522	6.073	0.771	1.9.969	51.841	3.506	0.565	1.9.511
0.797	20.073	58.235	8.193	0.796	1.9.581	48.979	4.161	0.617	1.9.565
0.822	20.010	60.944	6.328	0.824	1.9.521	47.251	2.147	0.668	1.9.306
0.849	18.989	59.089	6.947	0.848	1.9.338	46.544	2.295	0.719	1.9.124
0.873	18.518	57.721	6.089	0.875	1.9.301	46.761	1.165	0.772	1.8.946
0.900	18.724	53.690	3.248	0.899	1.9.207	46.501	0.730	0.822	1.8.839
0.925	18.359	50.415	3.191	0.928	1.8.809	45.441	1.588	0.875	1.8.837
0.951	18.835	49.814	1.160	0.951	1.8.752	45.437	1.361	0.925	1.8.955
0.976	18.561	48.556	2.435	0.976	1.9.064	47.042	0.475	0.976	1.9.001
1.000	18.886	47.864	1.156	1.000	1.8.846	45.877	1.353	1.000	1.8.966

Table C-1. (Continued).

STATION 3									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S
PHH=90.00 YOR/SR=0.00									
0.000	18.138	42.709	4.794	-0.220	18.102	4.9.628	3.651	0.001	17.246
0.053	17.487	43.307	3.417	0.026	18.505	4.7.691	1.426	0.026	16.867
0.102	17.083	43.671	2.132	0.052	19.306	4.5.700	-1.753	0.051	16.232
0.154	17.032	43.759	0.819	0.077	19.728	4.4.734	-1.607	0.077	15.729
0.205	17.021	42.244	1.997	0.103	19.985	4.6.603	-0.117	0.102	15.157
0.257	17.448	40.010	1.665	0.128	19.820	4.8.060	-2.883	0.129	15.866
0.309	17.590	40.074	2.51	0.155	20.095	5.2.954	1.693	0.153	15.051
0.365	18.050	40.822	3.820	0.180	20.481	5.4.346	1.235	0.180	15.269
0.411	18.455	40.295	3.577	0.206	20.145	5.6.545	0.648	0.204	15.177
0.463	18.602	39.740	4.879	0.231	19.479	5.6.123	2.213	0.258	15.453
0.516	19.145	42.014	3.415	0.257	18.915	5.5.240	1.642	0.310	15.481
0.540	19.025	42.097	4.379	0.283	18.590	5.5.116	1.673	0.360	15.771
0.567	19.334	43.466	3.129	0.309	17.974	5.2.674	1.941	0.411	15.677
0.591	19.511	44.928	1.342	0.360	17.653	4.8.971	0.674	0.462	15.243
0.617	19.387	44.743	2.700	0.412	17.586	4.1.474	1.080	0.514	15.552
0.642	20.206	45.465	0.704	0.464	17.973	3.9.237	1.449	0.565	15.972
0.669	19.609	45.664	2.273	0.514	18.214	3.8.429	1.844	0.616	16.628
0.696	19.574	45.907	0.706	0.565	18.261	3.7.858	2.953	0.668	17.030
0.719	19.637	50.651	-1.454	0.617	18.322	3.6.601	4.295	0.721	16.203
0.746	19.230	52.272	0.033	0.668	18.819	3.8.217	5.186	0.746	16.602
0.771	19.472	55.354	-1.240	0.719	19.078	3.7.727	6.227	0.771	16.623
0.797	19.325	51.823	0.330	0.771	19.332	3.8.350	6.679	0.796	16.608
0.827	19.153	49.597	2.653	0.822	19.487	3.8.117	6.515	0.822	17.619
0.848	18.877	46.345	4.836	0.849	19.577	3.9.951	5.219	0.848	16.857
0.875	18.919	44.662	5.069	0.873	18.997	4.3.245	5.866	0.874	17.039
0.899	19.034	43.823	4.909	0.899	18.840	4.6.344	4.289	0.899	17.716
0.926	18.663	41.724	5.748	0.925	18.151	4.9.183	3.986	0.925	17.522
0.951	18.305	38.748	6.110	0.951	18.169	5.1.189	2.937	0.950	17.527
0.975	17.946	39.372	6.083	0.976	18.558	5.1.122	-0.615	0.976	17.180
1.000	17.936	40.510	4.845	1.000	18.742	4.9.106	-1.316	1.000	16.423

Table C-1. (Continued).

STATION 4									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S
PHH=10.00 YOR/SR=0.34									
0.000	16.007	36.358	2.118	0.000	15.258	37.042	1.515	0.000	15.098
0.025	14.836	37.024	3.414	0.025	14.462	38.207	1.395	0.025	14.652
0.051	14.363	37.354	2.124	0.051	14.060	38.151	-0.664	0.051	14.483
0.077	13.538	34.904	2.587	0.077	1.386	37.488	-1.836	0.077	14.321
0.104	13.721	34.655	1.012	0.103	14.463	36.179	-5.191	0.103	14.633
0.129	14.130	33.415	-0.973	0.129	15.202	34.132	-4.952	0.128	14.898
0.154	15.222	32.530	-2.353	0.154	15.936	32.001	-3.473	0.154	15.174
0.180	16.295	33.140	-2.383	0.180	16.530	32.139	-1.069	0.181	15.799
0.205	17.170	32.631	-2.631	0.206	16.509	32.682	5.312	0.206	15.753
0.231	17.576	34.473	-1.960	0.257	15.903	38.038	9.010	0.232	16.475
0.257	17.771	35.923	-0.596	0.283	15.028	28.636	9.113	0.257	16.961
0.310	17.206	36.239	1.178	0.308	14.954	38.048	8.333	0.308	17.232
0.360	16.382	37.435	2.024	0.334	14.742	40.309	7.693	0.360	17.686
0.412	15.195	35.925	4.548	0.360	14.550	39.903	8.209	0.412	17.797
0.463	14.133	35.047	5.541	0.386	14.736	39.237	7.157	0.463	17.256
0.490	14.035	38.697	4.090	0.411	14.558	38.074	9.370	0.514	16.187
0.514	14.214	32.701	2.409	0.438	15.257	37.765	7.290	0.541	15.363
0.541	14.193	38.803	2.395	0.462	15.681	36.742	6.363	0.565	15.198
0.565	14.496	38.323	1.316	0.499	16.075	35.112	5.393	0.593	15.700
0.592	15.004	38.176	1.450	0.514	16.591	33.728	7.603	0.617	15.505
0.617	15.557	38.290	1.354	0.565	17.087	32.249	0.868	0.643	15.776
0.642	16.043	38.047	2.305	0.617	16.811	30.622	-0.878	0.669	15.831
0.668	16.572	37.566	2.024	0.668	16.830	31.337	-4.287	0.694	16.884
0.720	16.903	36.861	2.075	0.720	16.870	32.814	-5.692	0.720	17.135
0.771	17.180	35.617	-0.496	0.771	17.567	33.018	-6.692	0.772	17.806
0.822	17.306	35.031	-2.118	0.822	17.653	33.061	-4.489	0.822	17.495
0.874	17.632	34.241	-2.941	0.874	17.474	33.957	-3.072	0.874	17.488
0.926	17.807	34.095	-2.792	0.925	17.033	34.803	-1.615	0.925	17.382
0.976	17.503	34.412	-0.874	0.977	16.642	37.554	-1.637	0.976	17.035
1.000	16.977	35.487	0.699	1.000	16.155	36.857	-0.842	1.000	16.541

Table C-1. (Continued).

STATION 4									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S
PHH=30.00 YOR/SR=0.00									
0.000	16.219	35.230	-C.478	0.000	17.269	36.367	-0.843	-0.000	16.195
0.026	15.603	36.493	-0.157	0.052	16.461	36.219	0.081	0.027	15.805
0.051	15.705	37.456	-0.538	0.078	16.558	36.179	0.028	0.051	15.277
0.078	15.124	36.239	0.517	0.104	15.870	37.816	0.955	0.078	14.726
0.104	14.846	36.922	-0.426	0.126	15.168	37.672	0.367	0.104	14.022
0.128	14.127	36.532	-0.364	0.154	14.549	36.177	0.006	0.128	13.486
0.155	13.557	35.720	0.101	0.180	13.869	38.834	-0.671	0.154	12.494
0.180	13.694	33.960	-1.001	0.206	13.891	36.871	-2.806	0.180	12.345
0.206	13.644	32.377	0.996	0.232	14.170	34.461	-4.253	0.206	12.747
0.232	14.703	31.149	-1.977	0.257	14.722	33.214	-4.013	0.232	13.575
0.257	15.422	29.229	-0.742	0.283	16.047	33.630	-4.471	0.257	14.261
0.283	16.217	28.996	1.717	0.308	17.084	34.361	-3.434	0.282	15.101
0.308	16.923	29.678	2.723	0.334	17.211	33.677	-0.079	0.308	15.569
0.334	17.285	32.021	3.221	0.359	17.823	34.193	-0.002	0.335	15.984
0.360	17.717	32.755	2.893	0.386	17.728	34.605	-2.168	0.360	15.859
0.387	17.755	32.797	2.517	0.411	17.677	36.329	2.005	0.385	16.234
0.411	18.042	33.650	1.686	0.437	17.453	36.092	3.095	0.412	16.368
0.463	18.278	33.208	-0.359	0.463	17.433	37.586	-2.834	0.438	16.312
0.514	18.096	32.197	-1.121	0.489	17.389	37.794	2.595	0.462	16.480
0.565	17.593	31.935	-0.621	0.513	17.103	38.237	3.279	0.514	16.435
0.617	17.642	31.768	-1.242	0.566	16.926	39.310	3.663	0.566	16.522
0.668	16.990	31.577	0.350	0.617	16.871	38.644	4.467	0.617	16.665
0.720	16.742	32.376	-0.526	0.669	17.035	38.099	3.809	0.668	16.529
0.771	16.677	32.649	-0.980	0.720	17.173	37.825	2.697	0.719	16.377
0.822	16.267	33.571	1.022	0.771	17.217	36.621	2.304	0.771	16.295
0.874	15.854	34.013	2.326	0.822	17.214	35.811	2.794	0.822	16.109
0.925	16.154	34.306	1.264	0.874	17.523	36.107	0.442	0.874	15.775
0.951	16.692	35.618	1.243	0.925	17.410	35.426	0.474	0.925	15.401
0.977	16.122	34.790	1.313	0.976	17.255	34.944	0.590	0.976	15.068
1.000	16.286	35.162	0.159	1.000	17.217	35.901	0.360	1.001	15.104

PHH=30.00  
YOR/SR=0.34

PHH=30.00  
YOR/SR=0.69

Table C-1. (Continued).

STATION 4									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S
PHH=50.00 YOR/SR=0.17					PHH=50.00 YOR/SR=0.34				
PHH=50.00 YOR/SR=0.17					PHH=50.00 YOR/SR=0.34				
0.000	18.428	30.844	0.760	-0.000	18.666	33.919	1.107	0.000	18.335
0.052	18.166	30.994	2.038	0.052	18.237	33.508	2.682	0.053	18.092
0.077	18.289	31.621	1.301	0.077	18.251	34.565	2.253	0.102	17.647
0.102	18.270	32.181	1.199	0.102	17.969	34.492	2.780	0.129	17.265
0.129	17.952	33.318	1.132	0.129	17.125	34.522	4.598	0.155	16.685
0.154	16.954	33.843	2.827	0.158	16.396	35.255	4.447	0.181	16.514
0.180	16.149	34.538	3.346	0.180	15.661	35.250	4.614	0.207	15.514
0.205	15.055	34.670	2.977	0.207	14.897	34.204	4.964	0.232	14.895
0.232	14.014	35.489	3.405	0.232	14.440	36.212	1.719	0.257	14.493
0.257	13.269	36.724	2.018	0.257	13.909	34.527	1.192	0.283	13.626
0.282	13.010	34.661	1.268	0.282	13.458	33.536	-0.361	0.308	13.299
0.308	13.695	32.306	-2.798	0.309	13.252	32.157	-1.299	0.334	13.248
0.335	14.054	29.845	-1.520	0.335	13.547	30.022	-3.245	0.361	13.760
0.360	14.768	29.945	-1.185	0.360	14.369	30.440	-3.837	0.385	14.545
0.386	15.281	29.585	-0.944	0.387	14.828	30.536	-1.253	0.411	14.936
0.413	15.654	30.749	0.944	0.411	15.190	30.890	0.495	0.437	15.299
0.438	15.783	32.189	1.644	0.437	15.526	32.913	1.178	0.462	15.733
0.462	15.644	32.548	2.508	0.462	15.677	32.936	0.953	0.489	15.908
0.491	15.718	32.542	2.154	0.489	15.850	33.934	0.892	0.514	16.002
0.514	15.674	33.191	2.032	0.517	15.898	33.567	0.885	0.540	16.163
0.566	15.909	33.426	0.754	0.567	15.899	33.093	0.899	0.565	16.306
0.617	16.051	32.936	0.713	0.617	16.229	31.323	0.509	0.621	16.708
0.668	16.358	32.164	-0.366	0.672	16.608	30.934	-0.034	0.669	16.414
0.721	16.826	30.134	-0.516	0.721	16.937	31.420	0.613	0.720	16.816
0.771	16.982	29.645	0.278	0.774	17.060	31.276	1.279	0.772	16.876
0.822	17.272	28.898	0.546	0.822	17.424	32.624	1.954	0.822	17.108
0.873	17.414	29.128	1.537	0.874	17.467	31.900	2.160	0.873	17.533
0.925	17.688	30.355	2.055	0.925	17.774	33.262	2.634	0.925	17.832
0.977	17.900	31.239	2.143	0.977	18.070	33.317	2.032	0.974	17.971
1.001	17.910	31.450	2.232	1.000	18.134	33.550	2.276	1.000	17.904

Table C-1. (Continued).

STATION 4									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S	BETA Y DEG	BETA R DEG	Y/SS M/S	V M/S
PHH=50.00 YOR/SR=0.50									
0.000	17.929	32.512	0.055	-0.000	17.500	30.148	0.199	0.000	17.711
0.026	17.751	32.260	0.561	0.051	17.411	29.114	0.101	0.051	17.875
0.053	17.593	32.231	0.264	0.077	17.309	28.880	-0.036	0.077	17.504
0.077	17.291	31.687	1.187	0.103	17.187	28.556	-0.320	0.103	17.686
0.104	17.293	31.519	0.215	0.129	17.015	28.068	0.064	0.130	17.219
0.129	16.630	31.137	1.706	0.155	16.797	28.761	0.063	0.155	16.862
0.154	16.529	30.985	1.095	0.181	16.436	29.310	-0.095	0.181	15.971
0.180	16.255	30.639	0.910	0.205	15.202	29.702	-2.528	0.205	14.973
0.206	15.798	30.830	0.426	0.232	14.232	31.226	1.866	0.231	13.921
0.231	14.274	31.539	3.433	0.259	12.605	35.704	5.17	0.263	13.175
0.258	13.539	31.752	1.643	0.285	12.056	35.773	0.773	0.283	13.133
0.283	12.861	30.405	-0.166	0.308	12.781	35.379	-1.014	0.308	13.366
0.308	12.727	31.645	-3.390	0.334	13.742	35.060	-0.445	0.334	14.335
0.334	12.631	29.663	-1.697	0.360	14.564	33.035	1.152	0.359	15.060
0.362	13.771	29.948	-1.985	0.387	14.564	33.065	2.291	0.386	15.556
0.386	14.666	30.894	-0.099	0.412	16.012	33.874	2.533	0.413	16.086
0.411	15.305	31.564	1.509	0.437	16.188	34.107	2.631	0.437	15.999
0.437	15.798	31.632	2.151	0.463	16.389	33.606	2.325	0.464	16.282
0.463	16.161	32.325	1.598	0.489	16.208	33.057	0.488	0.488	15.812
0.513	16.319	32.230	1.530	0.514	16.224	32.821	1.592	0.513	15.866
0.566	16.281	31.503	1.633	0.571	16.218	32.827	0.638	0.566	15.556
0.617	16.479	31.600	0.639	0.617	16.099	33.127	0.533	0.617	16.060
0.668	16.527	32.725	0.797	0.668	16.296	33.668	0.170	0.668	16.539
0.720	16.423	31.162	1.192	0.723	16.456	33.338	-0.095	0.721	16.814
0.774	16.864	33.507	0.301	0.774	16.691	32.684	0.095	0.772	16.321
0.822	17.062	32.783	-0.200	0.822	17.162	31.843	-0.962	0.824	17.104
0.874	17.739	32.996	-0.869	0.875	17.550	31.546	-1.238	0.874	17.481
0.925	17.796	32.633	-0.464	0.925	17.473	30.496	-0.258	0.926	17.600
0.973	17.688	30.518	-0.518	0.975	17.557	29.746	-0.197	0.973	17.636
1.000	17.631	31.112	0.016	1.000	17.543	28.975	0.197	1.000	17.831

Table G-1. (Continued).

STATION 4									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S
PHH=70.00 YOR/SR=0.00					PHH=70.00 YOR/SR=0.34				
YOR/SR=0.34					YOR/SR=0.69				
-0.092	17.012	29.795	-3.436	0.000	16.665	32.309	-1.778	0.000	15.363
-0.052	17.377	29.250	-0.453	0.053	16.170	32.678	0.230	0.051	15.615
0.103	17.179	28.802	0.354	0.103	16.674	34.375	-0.370	0.103	16.186
0.154	17.209	28.448	1.138	0.154	16.486	33.635	1.397	0.154	16.336
0.180	17.155	28.724	1.644	0.180	16.473	32.940	1.134	0.206	16.534
0.205	17.347	29.757	1.474	0.205	16.423	34.297	1.574	0.237	16.516
0.232	16.992	30.691	1.912	0.232	16.371	33.602	0.701	0.257	16.236
0.257	16.465	32.175	1.710	0.257	16.087	31.356	1.230	0.283	15.929
0.283	15.146	35.185	6.670	0.282	15.052	31.041	4.018	0.308	15.271
0.308	13.744	37.934	8.123	0.308	14.388	32.229	3.908	0.334	14.534
0.334	13.449	37.798	6.623	0.334	13.643	31.906	3.708	0.359	13.877
0.359	13.195	37.911	1.374	0.360	12.871	30.728	2.858	0.386	13.221
0.387	13.617	32.587	-2.667	0.386	12.670	28.235	0.281	0.412	13.623
0.411	14.834	28.863	1.410	0.411	13.147	25.912	-3.013	0.437	14.576
0.437	16.156	30.375	2.693	0.436	13.436	24.117	-1.639	0.464	15.864
0.464	16.769	31.688	3.614	0.462	14.991	24.897	-1.588	0.489	16.616
0.488	16.897	33.335	2.580	0.489	15.768	27.926	-0.125	0.514	17.044
0.515	16.790	33.971	2.484	0.514	15.951	28.486	0.775	0.539	16.929
0.540	16.393	32.570	4.416	0.540	15.931	28.239	1.624	0.565	17.158
0.565	16.397	33.660	3.711	0.565	16.055	28.830	1.802	0.591	17.052
0.592	16.407	34.240	3.083	0.590	16.177	25.389	1.056	0.617	16.782
0.618	16.140	33.831	3.735	0.617	16.212	26.010	1.082	0.643	16.783
0.668	16.073	34.368	3.084	0.668	16.188	26.513	1.255	0.669	16.769
0.720	15.916	32.972	2.497	0.719	16.327	26.289	0.103	0.719	16.422
0.771	15.689	31.722	2.697	0.771	16.455	26.960	-0.565	0.772	16.020
0.822	15.782	30.430	1.681	0.822	16.180	25.306	-0.167	0.822	15.752
0.874	16.156	29.865	0.947	0.873	16.229	29.447	-0.516	0.873	15.466
0.925	16.509	28.701	-0.206	0.925	16.083	30.880	-0.534	0.926	15.329
0.976	16.468	28.954	0.874	0.976	16.032	30.387	-1.559	0.973	14.977
1.201	16.506	23.425	1.330	1.000	15.651	30.784	-0.187	1.000	15.628

Table C-1. (Continued).

STATION 4									
Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S	BETA Y DEG	BETA R DEG	Y/SS	V M/S
PHH=90.00 YOR/SR=0.00					PHH=90.00 YOR/SR=0.69				
-0.000	15.089	35.341	1.506	-0.000	14.969	30.861	1.432	0.001	15.169
0.052	14.855	34.401	0.849	0.051	14.717	30.554	0.099	0.051	14.909
0.104	14.558	34.521	1.369	0.102	14.319	31.021	-1.191	0.103	14.938
0.155	14.475	33.488	1.696	0.155	14.533	32.562	-2.312	0.154	14.690
0.205	14.842	33.293	0.544	0.180	14.323	32.515	-1.119	0.205	14.760
0.257	14.866	32.045	1.799	0.206	14.525	32.364	-1.042	0.253	14.771
0.283	15.077	32.966	1.407	0.231	14.292	31.962	0.415	0.283	14.325
0.309	14.267	31.415	5.857	0.258	14.380	31.603	0.603	0.308	13.699
0.334	14.190	32.958	5.473	0.283	14.663	31.665	2.088	0.334	12.908
0.360	12.429	33.137	1.1067	0.309	13.364	31.562	3.952	0.361	11.964
0.386	11.342	33.084	1.162	0.334	12.510	32.252	3.644	0.386	11.088
0.412	10.925	35.0763	5.549	0.359	11.382	35.0165	1.278	0.412	10.865
0.437	11.472	34.5945	1.225	0.385	10.450	34.267	1.843	0.437	11.746
0.462	12.562	34.0948	5.013	0.412	10.487	32.0174	2.135	0.463	12.700
0.483	14.208	35.378	-0.6411	0.457	11.458	32.0817	2.681	0.488	13.934
0.513	14.734	36.024	3.006	0.463	12.552	31.129	4.707	0.515	14.717
0.539	15.453	37.445	2.963	0.488	13.679	32.292	4.993	0.541	14.880
0.566	15.523	38.578	3.0403	0.514	14.212	32.770	4.794	0.565	15.118
0.591	15.021	38.941	2.739	0.540	14.262	34.896	5.099	0.591	15.144
0.617	16.168	40.461	3.356	0.565	14.386	32.361	5.031	0.618	15.407
0.642	15.715	39.221	5.015	0.591	14.908	32.655	2.568	0.643	16.167
0.668	15.923	40.773	5.262	0.616	14.810	33.663	4.680	0.668	16.355
0.694	15.869	40.417	5.275	0.668	15.216	30.648	3.387	0.695	16.621
0.719	15.680	39.784	5.604	0.720	15.487	28.728	4.126	0.719	16.497
0.770	15.994	38.412	3.368	0.771	16.152	28.718	2.593	0.772	16.587
0.823	15.745	37.212	3.765	0.822	16.364	29.046	2.462	0.822	16.072
0.875	15.375	35.403	4.989	0.874	16.181	29.424	2.764	0.874	15.619
0.926	15.075	34.651	3.331	0.925	15.633	28.291	1.757	0.925	14.767
0.967	15.119	34.247	1.689	0.976	14.746	30.250	0.774	0.975	13.969
1.000	14.067	32.93	3.847	1.000	14.444	31.702	0.733	1.000	13.931